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A liquid organic carrier of hydrogen as a fuel for automobiles

(Nuclear power as a motive power for cars)

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A LIQUID ORGANIC CARRIER OF HYDROGEN

AS A FUEL FOR AUTOMOBILES

(Nuclear power as a motive power for cars)

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September 1979

*** temporary co-worker**

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Abstract

A system of storing energy in a hydrogen containing fuel for the motor car is discussed. The recyclable liquid chemical carrier is:

$$
(\texttt{Method} \cdot \texttt{name}_{liquid}) \xrightarrow{\texttt{dehydrogenation}} (\texttt{Volume}_{liquid}) * (\texttt{Hydrogen}_{gas})
$$

The reverse reaction, the hydrogenation of toluene, occurs in a regional plant connected to a source of hydrogen (electrolysis of water) with a significant by-product being heat at 200 ^OC for district heating. The system is able to store hydrogen in liquid form under ambient temperature and pressure even in a small motor car. The concentration of hydrogen is 6.1 *%* by weight. The release of gaseous hydrogen from the liquid methylcyclohexane needs a chemical catalytic reactor having a temperature of 300 $^{\circ}$ C and a pressure of some bars. Thes reaction has been well studied. The thermal energy for the dehydrogenation is taken from the exhaust gases at $760 °C$.

A layout of the most important processes of the system is given. As a reference case a "motor car of tomorrow" is taken, having a total power of 20 kW (me) and an average power of 10 kW (me). Distances travelled on the open road are 320 km, in the city 170 km. The mean efficiency of the hydrogen engine is 25 *%•* For this car the proposed system consists of a 120 litre tank for the liquid organic carrier, and a catalytic tank of 15 litres. During a longer parking period the continuous heating of the insulated reactor requires less than 100 watts taken from a battery and can be supplied for some hours. Starting the cold engine and acceleration makes it necessary to burn small amounts of the toluene directly $$ less than 2 *%* of the amount carried.

The local regional hydrogen filling station in supplying a population of approx 10*000 people having 1000 hydrogen driven vehicles, includes an electrolysis plant, hydrogenation plant for converting toluene into methylcyclohexane and the tanks for storing the two liquids.

The hydrogen is produced during periods of low electrical load (8 hours). The electrical power required is 6 MW(el): the daily production 1140 kg hydrogen which provides annually for 1000 small vehicles the power to travel 15'000 km. For the economic solution in Switzerland one has the following parameters $(1 SFr = S$ wiss France $2^2 0, 5$ U\$ (1979) :

1. Liquid organic carrier of hydrogen as an automobile fuel.

One of the important aims of future energy planners is to try and couple the nuclear power plant with the fuel system of the motor car. One possible scheme might be:

Nuclear fuel \rightarrow nuclear power station \rightarrow electricity \rightarrow electrolysis of water \rightarrow hydrogen \rightarrow fuel for the motor car.

The best known system is to use solid metal hydrides as the storage for automobile fuel:

From source
\n(rilling station)
\n
$$
\begin{array}{c}\n\bullet (\text{H}_{2}) + [\text{Me}] \xrightarrow{\text{Ioda}} [\text{NeH}_{x}] \\
\bullet \qquad \qquad \text{unloaded} \\
\text{stronge} \qquad \qquad \text{forage} \\
\text{Used by engine} \qquad \bullet \qquad \text{H}_{2}) + [\text{Ne}] \xrightarrow{\text{unloaded}} [\text{MeH}_{x}]\n\end{array}
$$
in the
\n
$$
\begin{array}{c}\n\bullet \qquad \text{In the second part of the right,}\n\end{array}
$$

As an alternative Sultan and Shaw, 1975 (26) have proposed the following system using liquid "organic hydrides":

From source \longrightarrow (H_2) \rightarrow $[TO]$ $\frac{10aq}{q}$ $[MC]$ (filling station) $\qquad \qquad \bullet$ i unloaded [loaded] in the car U by engine \rightarrow $-V$ $($ H ₂ $)$ + \uparrow

The chemical reaction is:

 $(3H_2)$ + ${C_7H_8}$ unload ${C_7H_1}$ + \sim 210 kJ/mol MC

The enthalpy $Q_{de} = 210$ kJ/mol MC comes from the hydrogenation of toluene and is not connected with the enthalpy of hydrogen burning.

The system could be u:;ed as a storage method for hydrogen fuel for the motor car as shown in figure 1. The reference car system is given in table 1.

The system proposed is called MTH:

(Methylcyclohexane, Toluene, Hydrogen).

The symbols used in this paper are:

1» -

2. Physical and chemical properties of the components.

Both the components of the proposed system, that is methylcyclohexane (MC) and toluene (TO) are liquid at ambient temperatures. Their freezing points lie around - 100 $^{\circ}$ C and boiling points \sim + 100 $^{\circ}$ C. Table 2 gives their most important properties.

A rather important problem involving the storage of energy for the motor car is the fact that the dehydrogenation of methylcyclohexane requires heat G_{de} from an external source:

```
Methylcyclohexane + Q<sub>de</sub> --- Toluene + (34<sub>2</sub>)The value of Q_{\text{de}} is:
                                   210 kJ/mol MC
                                   1.25 MJ/kg EC
                                  35.2 MJ/kg H<sub>2</sub>
```
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Table 1: The reterence motor car.

Mote for Table 1; * The operation of the hydrogen burning engine at an equivalence ratio β 0.5 - 0.6 improves the brake thermal ef**ficiency by 10** *%* **to 20** *%* **compared to gasoline burning at** $\beta = 0.91$.

**** see appendix 9»8**

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FIG LA SIMPLIFIED SCHEME FOR THE RTH FUELED CAR SYSTEM

Since the heat of combustion $Q_{\alpha\alpha}$ of hydrogen is 121 MJ/kg H_2 the relation to the heat of dehydrogenation is

$$
Q_{de}/Q_{co} = 35.2/121 \ge 0.28
$$

Such a large amount of heat at the required temperature could be obtained from the following sources

- exhaust gases from the engine
- additional burning of hydrogen

This paper proposes the use of the heat contained in the exhaust gases which equals 45 *%* of the total energy balance at a temperature of 780 ^OC which is very high but by no means unreasonable (see appendix 9-1 and 9-4 for an exact calculation).

Additional serious problems of the system proposed are:

- the catalytic dehydrogenation of methylcyclohexane is a rather complex process
- the catalysts from this process have been extensively and intensively investigated. Not all technological details have been pi oven
- the best known catalysts for the dehydrogenation of methylcyclohexane are:
	- platinum on $A1_2O_3$ (ref. 24, 25)
	- vanadium oxide on $A1_2O_3$
	- copper-nickel
	- cobalt alloys
	- aluminium-cobalt-chromium-zinc
	- aluminium-chromium-molybdenum
	- nickel-phosphorus
	- aluminium-titanium-vanadium
	- chromium
	- ruthenium-silica
	- aluminium-molybdenum-nickel

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- the life of the catalysts should be at least ICC h. This penaits the reactivation of the catalysts every R^2 km, that is 3 times per year. To be competitive the cost of regeneration must be below some tens of dcllars
- the denydrogenation reaction produces some unwanted by-products. These could adversely affect the life of the catalyst but also total recycling process, that is the hydrogenation and storage of toluene.

3. Layout of the main components in the motor car.

The most important and crucial element of the system is the catalytic dehydrogenation unit. Table 3 shows the principle calculations giving the material flows for the car. Table *k* gives the layout calculations for the appropriate hydrogenation unit.

One can conclude from the thermodynamic calculations that to achieve the hydrogenation a thermal flux of \sim 15.2 kW(th) at more tnan 315 °C is needed.

The question arises whether it is possible to extract this amount of heat from the only easily available source in the car, that is the exhaust gases.

Appendix 9.3 and *9.^* giv? the calculation of the heat capacity of tne exhaust gases.

Figure 2 shows the values obtained from the calculations referred to above.

Table 2: Physical, chemical and toxic properties of components of the MTH-System.

Table 3: The calculation of the material flow for the reference motor car.

*) The higher efficiency for the hydrogen engine of 25 *%,* rather than the 20 *%* efficiency of gasoline engine is based on ref. 20, 29. (see also 9.8)

4. Environmental problems

As can be seen from table 2 the toxicity of both liquid components MC and TO are the same as for gasoline.

Air pollution from the hydrogen engine compared to the rasoline engine has been extensively discussed.

The thermal pollution of the proposed motor car is essentially lower than that of the petrol driven vehicle. For the later the heat losses are

The car based on the MTH system has losses in the cooling water and in direct radiation of approx 15 kW(th) plus in the exhaust gases only 3 kW(th). The thermal pollution is thereby approximately halved.

These thermal differences, amounting to approx 15 kW per vehicle can be realised at the local hydrogenation station at \sim 300 $\rm{^{\circ}C}$ and be used for local district heating, (see below).

5. The local filling station

This chapter considers the feasibility of 'he local hydrogen filling station which has the following complex functions.

- production of hydrogen by weans of the electrolysis of water, during the low load period (8 h in the night). Supplying approx 1000 hydrogen driven cars covering each 15'000 km per year.

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Table 4: Simplified calculation of the catalytic dehydrogenation unit.

* Note: The power needed for acceleration (see appendix 9.6) is not included.

 \bullet

 $\hat{\mathbf{r}}$

Table 5: Exhaust gases balance and tank-volume (see appendix 9.2)

- use of this hydrogen in the hydrogenation of toluene transforming it to methylcyclohexane, the hydrogen carrier in the tanks of the motor cars.
- recovery of the "thermal waste" from the electrolysis and hydrogenation plant to supply local district heating for some thousand people.

Figure 3 shows a simplified scheme of the proposed KTH system. Table 6 gives details of the calculations.

6. Some economic considerations

On the basis of established sources such as the "Schlussbericht der Eidg. Kommission fur Gesamtenergiekonzeption, Bern 1978" some economic calculations have been made and are presented in Table 7. See also Figure 5.

Figure 6 also shows the partial cost of the hydrogen fuel produced in the form of liquid methylcyclohexane and the bonus in the recovery of the waste heat from electrolysis (\sim 130 $^{\circ}$ C) and hydgrogenation *(^* 300 °C).

FIG 3 SCHEPE OF THE REGIONAL FILLING STATION

Table 5: Energy balance of the local hydrvgen filling station.

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*) According to the literature the future modular unit for electrolysis will **have** a power of approximately 10 MM (el).

FIG 'UENERGY FLOW IN THE REGIONAL FILLING STATION

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Table 7: Economic calculations

| Object Operation | | Unit | Value |
|---|--|------------------------------------|------------------|
| Price of electricity | low load period | SFr/kWh(e1) | 0.040 |
| $-$ $H =$ | | SFr/GJ(e1) | 11.11 |
| Efficiency of electro- assumption lysis | | percent | 80 \bullet) |
| Amount of electri- | theoretical (100%) | $GJ(e1)/kgH_2$ | 0.121 |
| city | pratical: 0.121/C.8 | | 0.151 |
| Annual hydrogen pro- duction | see table 6 | GJ(ch)/y | 50'000 |
| Electricity consump- tion | $(50'000/0.8)$ × x(109/3.6x103) | kWh(e1)/y | 1.73 x 10^{7} |
| Electricity cost, annualy | 1.73 x 10^{7} x 0.04 | SFr/y | 690'000 |
| Electricity cost for hydrogen production | 0.151×11.11 | $SFr/kg H_2$ $SFr/GJ(c\bar{h})$ | 1.68 13.89 |
| Capital cost of elec- trolysis plant | roughly calculated (see also ref 3) | Srr/y | 2'600'000 |
| Load factor | <u>8 h x 360 d</u> $24h \times 365 d$ | factor | 0.30 |
| Life in service | according ref. 3 | У | 15 ₁ |
| Annuity | arbitrary (also ref 3) | $%$ per y | 10.3 |
| Annuity, specific | calculated | SFr/kg h_0 | 0.66 |
| | | SFr/GJ(ch) | 5.45 |
| Operation and main- | roughly calculated | $SFr/kg H_2$ | 0.20 |
| tenance | | SFr/GJ(ch) | 1.65 |
| Total cost of H_2 pro- | calculated her | SFr/kg H_0 | 2.54 |
| duction | | SFr/GJ(ch) | 21.00 |
| Capital cost of a hy- drogenation plant | according ref. 3) roughly assumed | SFr/Q(ch) STr/kg R_0 | 6.58.5 0.46 |
| (methylcyclohexane from toluene) | specific cost | Srr/GJ (ch) | 3.8 |
| | capital cost, total | SPr. | 1'800'000 |

 \bullet

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*) See appendix 9-9

FIG 7. PRICE OF HYDROGEN FUEL IN MTH-SYSTEM VERSUS PRICE OF ELECTRICAL ENERGY

FIG 8. HEAT BALANCE IN THE HYDROGENATION-FILLING STATION

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7. Comparison of energetics with other car propulsive systems.

The following comparison gives some interesting values for various car propulsive systems.

Normalised for 1 Joule of energy in the motor-car (see figure 9).

*) Assuming additional energy for district Leating (or equivalent of 2 Joules (th)

- **) Corrected for the increase in weight of the 900 kg car due to the 100 kg battery (56 kWh (el)). This is an advanced battery sulphur/sodium type having 150 Wh (el)kg.
- Note: The cost of the battery is assumed to be 120 SFr/kWh (el). The optimistic cost of the battery is therefore (120 SFr/kWh (el) x (56 kWh(el)) = 5600 SFr. For an efficient system at least one battery must be in the filling station for every 3 batteries on the road. The cost of the battery for this reference car is 9000 SFr. The cost

FI6 9. COMPARISON OF ENERGY BALANCE OF SELECTED MOTOR-CAR SYSTEMS

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of the car without battery 7500 SFr. Result; the battery driven car is twice as expensive as the hydrogen *or* **gasoline driven car.**

8. Conclusions

A motor car system fuelled with gaseous hydrogen stored on board by means of a liquid organic carrier of methylcyclohexane-toluene **seems to be a feasable system.**

The prjs and contras are as follows:

- **Pro: A liquid carrier of hydrogen is very attractive even for small private vehicles**
	- **The beat of denydrogenation could be extracted from the exhaust gases**
	- **The fuel tank and catalytic reactors are not large: 120 (tank, 13** *I* **dehydrogenatian unit and a rather small heat exchanger for a car having a weight of 950 kg and a radius of operation of 320 km on the open road or 170 km in the city**
	- **The start up for a cold engine at intervals shorter than 10 hours can be covered by a battery of 12 V and 50 Ah**
	- **The start after a longer parking time can be achieved with electricity from the grid i.e. 10 kW(e) for 10 minutes, or by burning the methylcyclchoxane directly**
	- **The thermal pollution is 2.5 times lower than that of conventional engines**
	- **The toxicity of both liquid carriers MC and TO is the same as for gasoline**
	- **The loading of hydrogen into the toluene at the local plants**

seems to be feasable at the scale of about 1000 cars. The electrical power needed for electrolytic production of hydrogen for 8 h in the night is only 6 MW(el)

- Based on an electricity price of 0.042 SFr/kWh(el) and an oil and gasoline price without taxes of 0.4 SFr/1 the system can be economically self supporting. Even with an electricity price for the year 2000 (according to the GEK) of 0.061 SFr/kWh(el) the corresponding price of oil and gasoline would only be 0.55 SFr/£.
- Contra: $-$ The use of catalytic reactors is a limiting factor with their short lifetime, price and complexity of recharging
	- The by-products of dehydrogenation could be harmful for the environment even though they will be only 1 *%* of the emmis:;ion on one cycle
	- Much more development work remains to be done
	- The proposed filling station is a complex plant and includes
		- the electrolytic plant 6 MW(el); 1.14 t H₂/day
		- hydrogenation plant: 20 t methylcyclohexane/day
		- filling station with 2 tanks of \sim 25 m^3
		- *-* filling of 1000 cars each 15'000 km/y
		- $-$ a capital cost \sim 4.4 x 10⁶ SFr.

9. Appendix

- 9.1 Chemical_reactors_in_the_equilibrium_state
- a) Hydrogen production

 $(c_{6}H_{11}CH_{3})_{g} \longrightarrow$ ($c_{6}H_{5}CH_{3}/g$ * (3H₂)^g)

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```
Catalyser : Pt/Al<sub>2</sub>O<sub>3</sub> in powder form
Temperature: 610 K 
Pressure : 1 bar 
Enthalpy of hydrogenation: 210 kJ/mol MC (endothermic)
Efficiency of the hydrogenation reaction: 95 %
```
b) Hydrogen burning with excess air equal to X. (the equivalence ration ϕ equals λ^{-1})

$$
H_2 \div (\frac{1}{2} O_2 + 1.88 N_2) \longrightarrow H_2O \div (\lambda - 1) \frac{1}{2} O_2 + 1.88 N_2
$$

Enthalpy of burning: 240 kJ/mol H²

The value of X is selected as 2.17 since a hydrogen/air mixture with 2 *%* **Hp lies below the explosion limit and thereby a "normal" burning of hydrogen can be achieved. At the same time the burning has a high efficiency. Thus for X = 2.17 the following equation is obtained:**

$$
H_2 + 2.17 \left(\frac{1}{2} 0_2 + 1.88 N_2 \right) \longrightarrow H_2 0 + 0.58 0_2 + 4.07 N_2
$$

The maximum theoretical combustion temperature for this reaction can be calculated as (specific heats see table 8).

1'600 K (constant pressure) 1*959 K (constant temperature)

Note: According to (1) the engine exhaust gas temperature decreases significantly with operation at a low equivalence ratio *6.*

9.2 Heat_exchanger_sy-stem

The dehydrogenation of methylcyclohexane takes place in the gaseous form. To obtain a good energy extraction, a heat exchanger is used in which the enthalpy (heat) of the products (exhaust gas

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Another heat exchanger provides the reaction heat of iehydrogenar. ion .

9.3 Energy flow in the system.

The total energy requirements of the dehydrogenation system is made up **as follows:**

$$
\Delta H_{(to)}^{\rho} : \int_{298}^{T_{B(NC)}} c_{P[\kappa c]}^{\Delta t} + \Delta H_{B[\kappa c]} \rightarrow \int_{T_{B(KC)}}^{610} c_{P(KC)}^{\Delta T} + \Delta H_{D(KC)}
$$

From this: ΔH_{E}^{f} = 317 kJ/mol MC

This energy requirement ΔH_f is provided by the cooling of the dehydrogenation products $(C_6H_5CH_3 + 3H_7)$ and the exhaust gases (EG).

The cooling of the gaseous toluene and hydrogen gives the following amount of heat

$$
\Delta H_{(t_0)}^{\pi} : \int_{610}^{T_{B(TO)}} c_{P} (TO) dt + \Delta H_{B(TO)} * \int_{T_B}^{327} c_{P[TO]} dr + 3 \int_{610}^{327} c_{P(H_2)} dr
$$

$$
\Delta H_{(to)} = 101 \text{ kJ/mol} (T0 + 3H_2)
$$

The remaining energy is provided by the exhaust gases as follows:

$$
\Delta(\Delta H)
$$
 = 317 - 101 = 216 kJ/mol MC
\n $\Delta(\Delta H)$ = 3 $\int_{60}^{1050} Cp$ (EJ) df

Table 3: Some thermodynamic values.

£ii£Hi§£ion_of_s£ecific_heat C

 \sim

T s temperature (K) $C_{\mathbf{n}} \rightarrow \mathbf{a} + \mathbf{b}$ **T** + CT^2 ΔH ***** $\int C \mathbf{p} dT$ *** a**T ***** $\frac{1}{2}T^2$ ***** $\frac{1}{3}T^3$

| | Coefficients | | | Cp at 1050 K | |
|---------------------|--------------|-----------------------|-----------------------|--------------|-----------|
| | | | c | J/gK | J/mol K |
| H ₂ O | 34.39 | 0.62×10^{-3} | 5.61×10^{-6} | 2.307 | 41.5 |
| 0 ₂ | 28.23 | 2.54×10^{-3} | $0.54x10^{-6}$ | 1.095 | 35.0 |
| N_{2} | 28.3 | 2.54×10^{-3} | $0.54x10^{-6}$ | 1.181 | $33 - 0$ |
| H_{2} | 27.7 | $3.39x10^{-3}$ | | 14.655 | 29.3 |

Enthalpy of boiling, combustion and denydrogenation

- **B = Boiling**
- **V = Combustion**
- **D = Dehydrogenation**

Specific heat of KC, TO

| | Cp ${MC}$ = 180 J/mol ${MC}$ \cdot K | | |
|--|---|--|--------------------|
| | c_p (MC) = 240 J/mol (MC) \cdot K | | Boiling points: |
| | $C_{\rm P}$ $\left\{TO\right\}$ = 160 J/mol $\left\{TO\right\}$ \cdot K | | T_R (MC) = 374 K |
| | C_P (TO) = 170 J/mol (TO) \cdot K | | T_R (TO) = 383 K |
| | C_{P} [Fe] = 400 J/mol [Fe] \cdot K | | |

FIG **I D SCHFJC OF HEAT** FLOW

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9.4 Energy_balance_and_temperature_distribution

To simplify the presentation and calculation the system of heat exchanger and reactor is divided into six sections A to P. (In reality only two parts - heat exchanger and reactor).

It is assumed that the heat exchangers operate in counterflow. To calculate the surface area the following formula are used

 $\Delta T = T_{in} - T_{out}$

$$
T_m = \frac{(\Delta_{TO} - T_{MC})}{\ln(\Delta T_{TO}/T_{MC})}
$$

 \dot{Q} = AH x \dot{m} (\dot{m} = mass flow; $\frac{mol}{a}$) \dot{Q} = power [W] s

$$
F = \frac{\dot{Q}}{k \cdot \Delta T_m} \left[m^2 \right]
$$
 $F = \text{surface of the heat exchanger } [m^2]$
 $k = \text{heat transfer coefficient } [W/m^2 K]$

For the reference case it was assumed that

 $\hbar = 0.055$ mol MC/s

The total surface area of the complete heat exchange system is $\Sigma F = 2.03 m^2$

The Carnot efficiency for an Otto engine with hydrogen is

(V = const; T_{max} = 1959 K)

$$
n_{\text{carnot}} = \frac{T_{\text{max}} - T_{\text{out}}}{T_{\text{max}}} = \frac{1950 - 1050}{1950}
$$

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Table 9: Calculation of the surface area

* See "WSrme-Atlas" 1974.

System components (see toble 9)

For this paper it is assumed for the hydrogen Otto engine: n_{eff} = 0.25

On the relationship of n_{off} to the Carnot efficiency is: 6Ii ${\mathfrak{n}}_{\tt eff}$ / ${\mathfrak{n}}_{\tt Carnot}$ = 0.25/0.462 = 0.54

which appears reasonable,

9.5 Possible_layou of the heat exchanger and reactor.

The surface area of the heat exchanger is about 2 m^2 . With a tube cross section of 0.002 m² (ϕ 5 cm) a total length of 12.8 m is needed. The heat exchanger could possibly be dimensioned as follows (in a cube of 0.7 m) (Figure 11 a shows a proposed layout).

The inner cube has a side 0.5 m long. To that is added 0.1 m of insulation with a thermal conductivity $\lambda = 0.1$ W/m.K.

```
This gives a power loss of
```
 \mathbf{S}^{\perp}

loss = 150 W calculated loss of heat exc langer

The catalytic reaction unit can be designed as follows

 $(R = Radius, (m), L = length, (m)).$ $R_i = 0.1$ m $\Delta R = R_a - R_i$ $T_a =$ temperature at A $\Delta T = T_i - T_a$ $R_a = 0.2$ m $\Delta R =$ wall the $L = 0.4 m$

FIG 12 . GEOrETRY AND ISOLATION OF DEHYDROGENATION REACTOR

a) The geometry of the dehydrogenation reac+or

b) The isolation of the dehydrogenation reactor

The reactor volume without isolation is about 13 litres (see also table 4).

The power loss of the reaction unit is

$$
\dot{Q}_{\text{loss}} = \frac{T_a - T_i}{2n (R_a/R_i)} \cdot 2\pi \cdot \lambda \cdot L + 2 \left[\frac{R_a + R_i}{2} \right] \pi \frac{\lambda \cdot \Delta T}{R} \approx 30 \text{ W}
$$

The total losses of the combined system is therefore \sim 180 W.

9.6 Dynamic behaviour (Acceleration and cold starts).

Acceleration requires a power of 20 kW(me) and could be provided by

- a) direct injection of toluene into the engine (excess air!) or
- b) by additional batteries which are charged up during the journey.

```
a) TO Injection
```
- Burning during normal travel

 H_2 + 2.17 (0.50₂ + 1.88 N₂) \longrightarrow H_2 0 + 0.580 0₂ + 4.08 N₂ + AH $\Delta H = 240$ kJ

- Burning during acceleration with addition of 0.06 mol TO (e.g. 5.9 g TO) H_2 + 2.17 (0_2 + 1.88 N₂) + 0.06 TO \longrightarrow 1.42 H_2 0 + 0.42 CO₂ + $+ 4.08 N_2$

 $\Delta H = 504$ kJ

i.e. with the injection of only 5.9 g of TO the thermal power is more than doubled!

However the mechanical power does not increase by the same amount since the efficiency falls.

The advantages of this system lies in the availability c^* TO from the tank. (Note: With metal hydrides this advantage is not present).

The disadvantages of TO injections also for cold starts are as follows

- larger environmental impact from $CO₂$ and perhaps NO_v (nevertheless still fifty times smaller than with the gasoline engine)
- lower efficiency of the engine
- increased corrosion of the heat exchanger system
- burning of the hydrogen carrier which comes from liquid fossile fuel at a rate of about 2 *%* per fuel tank filling. That means after about 50 fillings the contents of the tank which is supposed to be recycled is completely exhausted and requires replacement.
- b) Cold starts

The energy required is to supply the heat for: the heating of the heat exchange (HE) system $Q_{HF} = \Sigma CP \cdot \Delta T \cdot m_{HF}$ and the thermal energy necessary to start the release of hydrogen.

It is arbitrarily assumed that
$$
m_{tot} = 1.5 m_{HE}
$$
.
\n m_{tot} = $(F \cdot d \cdot p)x2.5$; $p = 7800 kg/m^3$; $d = 2 mm$
\nF = surface (m^2)
\nCp = 0.4 kJ/kg (iron)
\n $\Delta T = \sqrt{T_{max} \times T_{min}} - T_u = \sqrt{1000 \times 273} - T_u = 250 °C$
\n T_u = temperature of the surroundings

The heat needed for a cold start is:

 Q_{HF} = 78 kJ

This heat can be produced either by burning toluene of by external electrical heating.

By burning 0.36 mol or 35 g TO the heat exchanger is brought to operating temperature and 3 mols of hydrogen produced. Thus a cold start needs about 0.04 *%* of the fuel tank contents. After 50 starts 2 *%* of the tank's contents are directly burnt. This amount must be replaced continuously in the MTH system.

c) Warm start

Energy required 317 kJ/mol MC thus 0.28 mol TO or 28 g TO must be burnt.

The loss of the heat transfer system of 180 W (see above) can be covered by a normal car battery. Additional equipment is necessary

- TO exhaust (as well as the hydrogen exhaust)
- process controller (microprocessor)

d) Electrically driven acceleration (using batteries)

Advantages:

- constant efficiency
- higher engine usage
- small environmental impact

Disadvantages:

- the car weighs more, requires a high fuel consumption
- additional equipment is necessary (batteries, electrical motor)

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9•7 Calculation_of_the_increased_fuel^r^guired

The following assumptions have been made (see tabl? 1)

In relation to the weight of the car (900 kg)

 $\frac{900 + 146}{900} = 116$ \$

Assuming 1 *%* **increase in weight corresponds to an increase 0.9 ? in fuel, required i.e. 16** *% x* **0.9 = 15** *%•* **After iterating one obtains an increase in the fuel requirements of 18** *%.*

*9***.8 Ef£iciency>_of_the_hj£drogen_engine.**

Based on numerous theoretical and practical studies the hydrogen engine is much more efficient than the equivalent gasoline engine.

Van Vorst (1974) writes: "Comparison of results (for the hydrogen and gasoline engine) indicates that thermal efficiency is significantly higher. An increase in efficiency from 25 *%* **to 100** *%* **were noted; the greatest increase being obtained at the lower power outputs. At the highest gasoline engine efficiency the efficiency of the hydrogen engine is 25** *t* **greater".**

H. May (1976) writes: "For the higher hydrogen mixtures a higher

efficiency is obtained as is obtained from the lower hydrogen throughput, which can be expected from the theoretical calculations".

9.9 Electrolysis_of_water

One of the aims of this paper is to discuss the feasability of small scale production of hydrogen for the hydrogenation of toluene to methylcyclohexane. This hydrogen is produced from the electrolysis of water using electrical energy from a nuclear power station.

At the present time (according to ref. 20) all the large scale water electrolysis plants are built as multi-module systems with a module representing 1-2 MW (el).

 $T = 80$ °C, investment cost 250 $\sqrt{s/W(e)}}$

Today's modern electrolysers have an electric efficiency of 75-80 *%.*

A promising method of water electrolysis is based on a solid polymer electrolite membrane, unfortunately having a high price per unit cell but nevertheless a high energy efficiency (targets of 90 *%* are aimed at). Operating temperature is 150 °C; the module system with 8-10 MW(el).

According to ref. 18 the price of hydrogen produced by electrolysis is dependent on the price of electricity and the plant capacity factor (as shown in figure 13).

According to ref. 21 the future development of electrolysis of water has the following goals:

 $\sim 10^{-11}$

FIG 13. ELECTROLYSIS OF WATER

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