EIR-Bericht Nr. 379

Eidg. Institut für Reaktorforschung Würenlingen Schweiz

A liquid organic carrier of hydrogen as a fuel for automobiles

(Nuclear power as a motive power for cars)

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Würenlingen, September 1979

EIR Report 379

A LIQUID ORGANIC CARRIER OF HYDROGEN

AS A FUEL FOR AUTOMOBILES

(Nuclear power as a motive power for cars)

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September 1979

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Abstract

A system of storing energy in a hydrogen containing fuel for the motor car is discussed. The recyclable liquid chemical carrier is:

The reverse reaction, the hydrogenation of toluene, occurs in a regional plant connected to a source of hydrogen (electrolysis of water) with a significant by-product being heat at 200 $^{\circ}$ C for district heating. The system is able to store hydrogen in liquid form under ambient temperature and pressure even in a small motor car. The concentration of hydrogen is 5.1 \sharp by weight. The release of gaseous hydrogen from the liquid methylcyclohexane needs a chemical catalytic reactor having a temperature of 300 $^{\circ}$ C and a pressure of some bars. Thes reaction has been well studied. The thermal energy for the dehydrogenation is taken from the exhaust gases at 700 $^{\circ}$ C.

A layout of the most important processes of the system is given. As a reference case a "motor car of tomorrow" is taken, having a total power of 20 kW (me) and an average power of 10 kW (me). Distances travelled on the open road are 320 km, in the city 170 km. The mean efficiency of the hydrogen engine is 25 \pm . For this car the proposed system consists of a 120 litre tank for the liquid organic carrier, and a catalytic tank of 15 litres. During a longer parking period the continuous heating of the insulated reactor requires less than 100 watts taken from a battery and can be supplied for some hours. Starting the cold engine and acceleration makes it necessary to burn small amounts of the toluene directly less than 2 \pm of the amount carried. The local regional hydrogen filling station in supplying a population of approx 10,000 people having 1000 hydrogen driven vehicles, includes an electrolysis plant, hydrogenation plant for converting toluene into methylcyclohexane and the tanks for storing the two liquids.

The hydrogen is produced during periods of low electrical load (8 hours). The electrical power required is 6 MW(el): the daily production 1140 kg hydrogen which provides annually for 1000 small vehicles the power to travel 15'000 km. For the economic solution in Switzerland one has the following parameters (1 SFr = Swiss France $\frac{2}{2}$ 0.5 U\$ (1979):

Price of electrici- ty during low load		Price of heating oil and gaso- line without taxes; SFr/litre
Year	SFr/kWh (el)	
1979	0.042	0.40
2000	0.061	0.55

1. Liquid organic carrier of hydrogen as an automobile fuel.

One of the important aims of future energy planners is to try and couple the nuclear power plant with the fuel system of the motor car. One possible scheme might be:

Nuclear fuel --- nuclear power station ---- electricity ---- electrolysis of water --- hydrogen ---- fuel for the motor car.

The best known system is to use solid metal hydrides as the storage for automobile fuel:

From source
$$(H_2) + [Me] \xrightarrow{10ad} [NeH_x]$$

(filling station) unloaded loaded storage loaded storage Used by engine $(H_2) + [Me] \xrightarrow{unload} [MeH_x]$

As an alternative Sultan and Shaw, 1975 (26) have proposed the following system using liquid "organic hydrides":

From source $(H_2) + (TO) \xrightarrow{load} (MC)$ (filling station) unloaded loaded loaded loaded WC in the car

The chemical reaction is:

 $(3H_2) + \{C_7H_8\} \xrightarrow{load}_{unload} \{C_7H_1\} + \sim 210 \text{ kJ/mol MC}$

The enthalpy $Q_{de} = 210 \text{ kJ/mol MC comes from the hydrogenation of toluene and is not connected with the enthalpy of hydrogen burning.$

The system could be used as a storage method for hydrogen fuel for the motor car as shown in figure 1. The reference car system is given in table 1.

The system proposed is called MTH:

(Methylcyclohexane, Toluene, Hydrogen).

The symbols used in this paper are:

gaseous	() or g
liquid	{	} or £
solid	1] or s
x		= stoichiometric coefficient
Me		= metal
н ₂		= gaseous hydrogen

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кс	* methylcyclohexane: $C_6 H_{11} CH_3 = C_7 H_{14}$
T0	* toluene: C ₆ H ₅ CH ₃ * C ₂ H _g
50	* exhaust gases
DK	<pre>s district heating</pre>
не	= heat exchanger
(to)	= total
(el)	= electrical
(ch)	- chemical
(th)	= thermal
(<u>a</u> e)	* mechanical
đ	= day = 24 h; y = year = 360 days
h	* hour
de	# dehydrogenation
ity	hydrogenation
co	- combustion
SFr.	= Swiss Francs = 0.5 US \$

2. Physical and chemical properties of the components.

Both the components of the proposed system, that is methylcyclohexane (MC) and toluene (TO) are liquid at ambient temperatures. Their freezing points lie around - 100 $^{\circ}$ C and boiling points \sim + 100 $^{\circ}$ C. Table 2 gives their most important properties.

A rather important problem involving the storage of energy for the motor car is the fact that the dehydrogenation of methylcyclohexane requires heat Q_{de} from an external source:

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Methylcyclohexane + Q_{de} ----- Toluene + (3H_2)
The value of Q_{de} is: 210 kJ/mol MC
1.25 MJ/kg MC
35.2 MJ/kg H<sub>2</sub>
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Property	Unit	Yalue
Weight of car without fuel	kg	900
Peak engine power	kV(me)	20
Sean engine power	kĽ(me)	19
Engine volume	L	1
Mechanical (thermal) efficiency	\$	25 \$ (hydrogen) 20 \$ (gasoline)
Equivalence ratio for hydrogen	\$	0.46*
Exhaust gas temperature	°C	•• • 777
Operation distance		
open road city	ka. ka	320 170
	-	.,
Fuel rating		
gasoline open road	£/100 km	6.45
" city	1/100 km	12.8
gasoline in tank	kg	16.01
hydrogen gas		
open road	kg/100 km	1,53
city	kg/100 km	1.90
hydrogen in tank	kg	5
Organic liquid cerrier		
(o.) veignt x; y) x efficiency;	#/100 h-	
open road	1/100 Km	54.2
	14/10/0 KOM	
organic carrier in tank	KG 0	0) (see appendix)
	τ.	
Organic/gasoline weight ratio		05/16=5.3

Table 1: The reference motor car.

Note for Table 1: The operation of the hydrogen burning engine at an equivalence ratio 30.5 - 0.6 improves the brake thermal officiency by 10 % to 20 % compared to gasoline burning at $\beta = 0.91$.

** see appendix 9.8

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FIG LA SIMPLIFIED SCHEPE FOR THE HTH FUELED CAR SYSTEM



Since the heat of combustion Q_{co} of hydrogen is 121 MJ/kg H₂ the relation to the heat of dehydrogenation is

$$Q_{de}/Q_{co} = 35.2/121 = 0.28$$

Such a large amount of heat at the required temperature could be obtained from the following sources

- exhaust gases from the engine
- additional burning of hydrogen

This paper proposes the use of the heat contained in the exhaust gases which equals $45 \ \text{m}$ of the total energy balance at a temperature of 780 $^{\circ}$ C which is very high but by no means unreasonable (see appendix 9.1 and 9.4 for an exact calculation).

Additional serious problems of the system proposed are:

- the catalytic dehydrogenation of methylcyclohexane is a rather complex process
- the catalysts from this process have been extensively and intensively investigated. Not all technological details have been proven
- the best known catalysts for the dehydrogenation of methylcyclohexane are:
 - platinum on Al₂03 (ref. 24, 25)
 - vanadium oxide on Al₂0₃
 - copper-nickel
 - cobalt alloys
 - aluminium-cobalt-chromium-zinc
 - aluminium-chromium-molybdenum
 - nickel-phosphorus
 - aluminium-titanium-vanadium
 - chromium
 - ruthenium-silica
 - aluminium-molybdenum-nickel

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- the life of the catalysts should be at least 100 h. This permits the reactivation of the catalysts every for km, that is 3 times per year. To be competitive the cost of regeneration must be below some tens of dollars
- the dehydrogenation reaction produces some unwanted by-products. These could adversely affect the life of the catalyst but also total recycling process, that is the hydrogenation and storage of toluene.

3. Layout of the main components in the motor car.

The most important and crucial element of the system is the catalytic dehydrogenation unit. Table 3 shows the principle calculations giving the material flows for the car. Table 4 gives the layout calculations for the appropriate hydrogenation unit.

One can conclude from the thermodynamic calculations that to achieve the hydrogenation a thermal flux of \sim 15.2 kW(th) at more than 315 °C is needed.

The question arises whether it is possible to extract this amount of heat from the only easily available source in the car, that is the exhaust gases.

Appendix 9.3 and 9.4 give the calculation of the heat capacity of the exhaust gases.

Figure 2 shows the values obtained from the calculations referred to above.

Property	Unit	Sydrogen fu	el 2º -System	Convertional Carl
		tolucar	methylcyclo- bezane	ortane, gasoline
Formula	-	c ¹ R ¹	C., H 14	
Punction		only as they	tier of brirgen	for direct burning
Kolecular weight	kg/kuol	32.15	9 6. 19	114.3
Boiling point	°c	110.6	101.2	125.0
Preezing point	•c	- 95	- 126.6	- 5613
Hest of vaporization	w/e	3Å7	321	310+350
	kJ/mol	31.9	31.5	-
Vapour pressure (20 ⁰)	ber	010 <u>5</u>	0.07	0.03
Density (20 ⁰)	xc/L	0.867	c769	0172
	7/801	0.106	0_127	-
Viscosity	c ?	0.590		
Heat capacity	J/mol K	160. T	150.0	170
liquid	J/g K	1.738	1.53	1.7
gas (250°)	J/¢ %	1.64	2.45	•
gas	J/mol K	170	24 3	-
Heat of dehydrogena- tion	kā/mol HC	-	(20%);204.3 (427%C);215.8	-
liest of burning	ж/:	-	only tydrogen	total burning
			5.64	32.5
			per litre HC	per litre gason Line
Toxicity	part per Mega	100	¥00	30%
ļ	mg/m ³ air	360	1+200	1300
Price, 1919 March	US \$/milon	9.7 -	:	0.7

Table 2: Physical, chemical and toxic properties of components of the MTM-System.

Property	Obtained by	Unit	Value
Reference car			see table 1
Average mechanical power	postulation	kW(me)	10
Efficiency	estimation	%	25 •)
Equivalence ratio	see appendix	$\phi = \lambda^{-1}$	0.46
Average thermal power	10 kW(me) x $\frac{1}{0.25}$	kW(th)	ኳሳ
Average flow of heat		kJ∕s	40
Heat of combustion of hydrogen	from litersture	MJ/kg MJ/molH ₂	121
Hydrogen fluy for power given above	(46 kJ/s) +(242kJ/mol)	mol H ₂ /s	0.164
Methylcyclohexane as source of hydrogen	MC → 3 H ₂ +TO	mol K ₂ mol TO	0.333 1.0
MC flux for power given above	(0.164)x(0.333)	mol MC/a	0.0555
Efficiency of deby- drogenstion pro- cess	estimation	×.	95
MC flux including efficiency	(0.055)x(1,0.95)	mol MC/s	0.76

Table 3: The calculation of the material flow for the reference motor car.

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*) The higher efficiency for the hydrogen engine of 25 %, rather than the 20 % efficiency of gasoline engine is based on ref. 20, 29. (see also 9.8)

4. Environmental problems

As can be seen from table 2 the toxicity of both liquid components MC and TO are the same as for gasoline.

Air pollution from the hydrogen engine compared to the casoline engine has been extensively discussed.

The thermal pollution of the proposed motor car is essentially lower than that of the petrol driven vehicle. For the later the heat losses are

-	cooling water and radiation	12	kW(th)
	Total	30	kW(th)

The car based on the MTH system has losses in the cooling water and in direct radiation of approx 15 kW(th) plus in the exhaust gases only 3 kW(th). The thermal pollution is thereby approximately halved.

These thermal differences, amounting to approx 15 kW per vehicle can be realised at the local hydrogenation station at \sim 300 $^{\circ}$ C and be used for local district heating. (see below).

5. The local filling station

This chapter considers the feasibility of the local hydrogen filling station which has the following complex functions.

- production of hydrogen by means of the electrolysis of water, during the low load period (8 h in the night). Supplying approx 1000 hydrogen driven cars covering each 15'000 km per year.

Property	Obtained from	Unit	Value
Methylcyclohexane flux average	see table 3	mol MC/s	0.06
Specific volume of catelyst	according Faith (1972)	m ³ ·s/(mol MC)	0.06
Volume of reactor	(0.06)x(0.06 mol NC/s)	m 3	0.004 calculated 0.013 for further calculation (arbitrary)
Temperature of catalyst	according Faith (1972)	к	590
Temperature of ex- haust gases - from engine - after reartor	estimation postulation	к к	1050 see table 5 610 and appendix
Logarithmic medium temperature difference	LMTD⇒ (1050-590)-(610-59 1n 1050 - 590 610 - 590	<u>о)</u> к	∿ 140
Overall beat trans- fer coefficient	rough estimation	₩/m ² •K	150
Heat of dehydrogena- tion of MC	literature	kJ∕≊ol MC	∿ 210
Heat flux to reac- tor	see table 3	kW(th)	∿ 15.2
Heating surface needed	15.2x10 ³ W .(150 W/m·K)x(140 K)	m ²	∿ 0.73
Length of tube	arbitrary	ш	0.5
Diameter of tube	$D=4 \times \frac{4 \times 10^{-3} m^3}{0.73}$ m	m	0,022
Surface of 1 tube	(0.5)x(0.022)	m ²	0.034 calcu-
Number of tubes	(0.73)/(0.034)	number	21 lated
Volume of 1 tube	(0.5)x(0.022 ² x3.14/4)	" ³	0.0002 for
Length of tubes	(0.73)/(0.034)	number	21 4 litre
Number of tubes	$(4 \times 10^{-3} \text{m}^3) / (0.002 \text{ m}^3)$	number	~ 20 reactor

<u>Table 4</u>: Simplified calculation of the catalytic dehydrogenation unit.

Table 4: Continuation

Property	Obtained from	Unit	Value
Wall thickness	arbitrary	M	1,2 · 10 ⁻³
Wall unit, sur- face	(0.022) ² x(3.14/4)	m ²	n.042
Wall unit, volume	(0.042) x $(1.2$ x $10^{-3})$	m ³	5 x 10 ⁻⁵
Density of steel	literature	kg/m ³	7500
Mass of tubes	(14)x(0.5)x(5x10 ⁻⁵)x x(7500)	kg	2.6
Catalyst, volume	from above	m ³	0.004 =)
Specific weight of catalyst	arbitrary	kg/m ³	2000
Catalyst weight	(0.004)x(2000)	kg	8
Reactor pitch	arbitrary (length/dia- meter)	L/D ratio	1
Cross section	21x(0.022x2) ²	2 2	0.04 see
Diameter	(0.04-4/3.14) ^{1/2}	ш	0.22 also
Reactor volume	(0.C4)x(0.5)	m ³	0.020 appen-
Reactor surface	(0.03x2)+(0.5x0.22)	 2	0.19 dix
Insulation, thick- ness	arbitrary	л	0.1
Heat conductivity	literature	W/m•K	0.05
Temperature dif- ference	estimated	К	300
Heat losses	(0.19)x(0.05/0.1)x(300)	W (th)	∿ 30 arbitrary
Relative heat los- ses	(30 W)/(40000 W)	,	∿ 0.2
Electrical battery for heat losses	60 W x 10 h equi val ent to 12 V x 50 Ah	Wh (el)	600
Heat capacity of reactor	(0.02 m ³)x(0.5 kJ/kg·K) x (2000 kg/m ³)	MJ (th)	6

* Note: The power needed for acceleration (see appendix 9.6) is not included.

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Parameter	Obtained from	Unit	Value
Exhaust gases	·····	• • ₩ -	
Hydrogen flux	from above given cal- culation	mol/s	0.17
Equivalence rating maximum lean	according Karim (1966)	$\phi \ (\lambda = 0^{-1})$	0.46 (λ= 2.17)
Relative mass of exhaust gases	according Finegold (1978)	<u>g exhaust</u> mol H ₂	151
Mass of exhaust gases	(151 g air/mol H ₂)x x (0.17 mol H ₂ /s)	g/s	26
Specific heat of exhaust gases	estimated	J∕g•K	1.2 (see appendix)
Temperature of ex- haust gases: in- put	see appendix 9.4	°C K	777 1050
Output temperature	see appendix 9.4	°C K	337 610
Temperature dif~ ference	(T input)-(T output) = 4T	°C	եր0
Heat capacity of ex- haust gases	(26 g/s)x(1.2 J/g·K)x x (44о K)	J/s kW (th)	15'500 15.5
<u>Tank volume</u> (simplified)			
Hydrogen flux	from above	mol H ₂ /s	0.17
Methylcyclohexane storage, load	(0.17x1/3)x(0.127 ½/mol x (4.2 hr)) e	110
Toluene storage unload	(100 1) x 0.83	L	92
Equivalent gasoline value	estimated	Ł	20
Tank dimension	arbitrary	e	120
	side of tank	m	(0.66)x(0.66)x x (0.25)
Tank weight (empty)	approximately	kg	20

<u>Table 5</u>: Exhaust gases balance and tank-volume (see appendix 9.2)



- use of this hydrogen in the hydrogenation of toluene transforming it to methylcyclohexane, the hydrogen carrier in the tanks of the motor cars.
- recovery of the "thermal waste" from the electrolysis and hydrogenation plant to supply local district heating for some thousand people.

Pigure 3 shows a simplified scheme of the proposed MTH system. Table 6 gives details of the calculations.

6. Some economic considerations

On the basis of established sources such as the "Schlussbericht der Eidg. Kommission für Gesamtenergiekonzeption, Bern 1978" some economic calculations have been made and are presented in Table 7. See also Figure 5.

Pigure 6 also shows the partial cost of the hydrogen fuel produced in the form of liquid methylcyclohexane and the bonus in the recovery of the waste heat from electrolysis (\sim 130 $^{\circ}$ C) and hydgrogenation (\sim 300 $^{\circ}$ C).



FIG 3 SCHEME OF THE REGIONAL FILLING STATION

Object	Operation	Unit	Value
Region	erbitrary	population	10°000
Notor cars, total	essumption	cars/region	4*000
Motor cars, hydro- gen	essumption	"	1*000
Electricity consumption	according GEX 2000	$\frac{kW - y (el)}{y - capita}$	1.1
Ration of low/high load		ratio	1.75
"Free" capacity, low load	calculated	kü (el) average	0.6
"Free" capacity in the region	0.6 x 10*000	НЫ (el)	6 •)
Daily "free" energy	6000 kW(el) x 8 h	GJ (el)/d	173
Electrolysis of wate r	efficiency, assumed	\$	50
Chemical energy in H ₂	173 GJ(el) x 0.80	€€(H ₂)/a	138
Amount of hydrogen	138 GJ/(0.121 GJ/kg)	kg H ₂ /d	1*140
Heat waste, electro- lysis	173 GJ(el) x 0.20	GJ(th)/d	 ◆ 35
Heat waste, hydroge- nation	138 GJ(eh) x 0.28	GJ(th)/d	~ 38
Total heat "waste"	35 + 38	GJ(th)/d	* 7 <u>3</u>
Ratio waste/elec- tricity	73 /173	ratio	0.42
Total amount of H ₂	$1140 \frac{kg}{d} \times 366 \frac{d}{y}$	t H ₂ /y	¥10
Chem. energy in H ₂	3110 t/y x 121 GJ/t	GJ(ch)/y	50*000

Table 5: Energy balance of the local hydrogen filling station.

Object	Operation	Unit	Value
Energy per car	50*000/1*000	GJ(ch)/y	50
Thermal power/car	during 300 h/y	k¥ (th)	~ 45
Mechanical power/ car	efficiency 0.25	kW (me)	~ 11
Efficiency of car with gasoline	assumption	ž	20
Equivalent of gaso- line	calculated	≎ /y	1.56x10 ⁶

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*) According to the literature the future modular unit for electrolysis will have a power of approximately 10 MW (el).



FIG 4 ENERGY FLOW IN THE REGIONAL FILLING STATION

Table 7: Economic Calculations

Object	Operation	Unit	Value
Price of electricity	low load period	SFr/kWh(el)	0.040
"		SFr/CJ(el)	11.11
Efficiency of electro- lysis	assumption	percent	80 •)
Amount of electri-	theoretical (100%)	GJ(el)/kg H ₂	0.121
city	pratical: 0.321/0.8	77	0.151
Annual hydrogen pro- duction	see table 6	GJ(ch)/y	50'000
Electricity consump- tion	(501000/0.8) × ×(109/3.6x103)	kWh(el)/y	1.73 x 10 ⁷
Electricity cost, annualy	1.73 x 10 ⁷ x 0.04	SFr/y	690'000
Electricity cost for hydrogen production	0.151 x 11.11	SFr/kg H ₂ SFr/GJ(ch)	1.68 13.89
Capital cost of elec- trolysis plant	roughly calculated (see also ref 3)	SFr/y	2*600*000
Load factor	<u>8 h x 360 a</u> 24h x 365 a	factor	0.30
Life in service	according ref. 3	У	15
Annuity	arbitrary (also ref 3)	🖇 per y	10.3
Annuity, specific	calculated	SFr/kg H ₂	0.66
		SFr/GJ(ch)	5.45
Operation and main-	roughly calculated	SFr/kg H ₂	0.20
tenance		SFr/GJ(ch)	1.65
Total cost of H ₂ pro-	calculated her	SFr/kg H ₂	2.54
duction		SFr/GJ(ch)	21.00
Capital cost of a hy-	according ref. 3)	SFr/GO(ch)	6.5 8.5
(methylcyclohexane	specific cost	SFr/GJ(ch)	3.8
irom (ofuene)	capital cost, total	SPr.	1*800*000

Object	Operation	Unit	Va lue
Total cost of H ₂ in the methylcyclohexand	calculated	SFr/kg H ₂ SFr/GJ(ch)	3.0 24.9
Heat recovery from the electrolysis and hydrogenation	from table 6 (73 GJ/d x 360)	GJ(th)/y	26*000
Heat recovered equi- velent to oil	oil: 30 MJ(th)/£	m ³ oil/y	870
Efficiency of burning of oil	assumption	z	70
Effective oil equi- valent	870 x 1/0.7	m ³ oil/y	1"230
Price of oil (low sulphur)	arbitrary	SFr/100 £	40
Value of oil equi- valent	1230 x 10 x 40	SFr/y	500*000
Price of gasoline	assumption	SFr/L	0.4
Value of gesoline equivalent	from table 6 (1.56 x 10 ⁶ x 0.4)	SFr/y	620*000
Total cash	calculated	SFr/y	430*000

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*) See appendix 9.9





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FIG 7. PRICE OF HYDROGEN FUEL IN MTH-SYSTEM VERSUS PRICE OF ELECTRICAL ENERGY



FIG 8. HEAT BALANCE IN THE HYDROGENATION-FILLING STATION

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7. Comparison of energetics with other car propulsive systems.

The following comparison gives some interesting values for various car propulsive systems.

Normalised for 1 Joule of energy in the motor-car (see figure 9).

Primary Fuel	Transformation Plant	Total pri- mary ener- gy (Joule)*	Engine, type	Heat for district heating (Joule (th))
Nuclear fuel	Light water reactor efficiency ~33%	15	Internal combus- tion en- gine n = 25 \$	2
		8.8	Fuel cell plus elec- tromotor	1
		9+5**	Elect bat- tery plus motor	0
Crude oiJ	Refinery efficiency ∿80%	10.0	Internal- combustion engine n = 20 \$	0

 Assuming additional energy for district Leating (or equivalent of 2 Joules (th)

- **) Corrected for the increase in weight of the 900 kg car due to the 400 kg battery (56 kWh (el)). This is an advanced battery sulphur/sodium type having 150 Wh (el)kg.
- Note: The cost of the battery is assumed to be 120 SFr/kWh (el). The optimistic cost of the battery is therefore (120 SFr/kWh (el) x (56 kWh(el)) = 5600 SFr. For an efficient system at least one battery must be in the filling station for every 3 batteries on the road. The cost of the battery for this reference car is 9000 SFr. The cost



FIG 9. COMPARISON OF ENERGY BALANCE OF SELECTED MOTOR-CAR SYSTEMS

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of the car without battery 7500 SFr. Result; the battery driven car is twice as expensive as the hydrogen or gasoline driven car.

8. Conclusions

A motor car system fuelled with gaseous hydrogen stored on board by means of a liquid organic carrier of methylcyclohexane-toluene seems to be a feasable system.

The pros and contras are as follows:

- <u>Pro</u>: A liquid carrier of hydrogen is very attractive even for small private vehicles
 - The heat of dehydrogenation could be extracted from the exhaust gases
 - The fuel tank and catalytic reactors are not large: 120 f tank, 13 f dehydrogenation unit and a rather small heat exchanger for a car having a weight of 950 kg and a radius of operation of 320 km on the open road or 170 km in the city
 - The start up for a cold engine at intervals shorter than 10 hours can be covered by a battery of 12 V and 50 Ab
 - The start after a longer parking time can be achieved with electricity from the grid i.e. 10 kW(e) for 10 minutes, or by burning the methylcyclohexane directly
 - The thermal pollution is 2.5 times lower than that of conventional engines
 - The toxicity of both liquid carriers MC and TO is the same as for gasoline
 - The loading of hydrogen into the toluene at the local plants

seems to be feasable at the scale of about 1000 cars. The electrical power needed for electrolytic production of hydrogen for 8 h in the night is only 6 MW(el)

- Based on an electricity price of 0.042 SFr/kWh(el) and an oil and gasoline price without taxes of 0.4 SFr/1 the system can be economically self supporting. Even with an electricity price for the year 2000 (according to the GEK) of 0.061 SFr/kWh(el) the corresponding price of cil and gasoline would only be 0.55 SFr/1.
- <u>Contra</u>: The use of catalytic reactors is a limiting factor with their short lifetime, price and complexity of recharging
 - The by-products of dehydrogenation could be harmful for the environment even though they will be only 1 % of the emmission on one cycle
 - Much more development work remains to be done
 - The proposed filling station is a complex plant and includes
 - the electrolytic plant 6 MW(el); 1.14 t H₂/day
 - hydrogenation plant: 20 t methylcyclohexane/day
 - filling station with 2 tanks of \sim 25 m³
 - filling of 1000 cars each 15'000 km/y
 - a capital cost $\sim 4.4 \times 10^6$ SFr.

9. Appendix

- 9.1 Chemical reactors in the equilibrium state
- a) Hydrogen production

 $(c_6H_{11}CH_3)_g \longrightarrow (c_6H_5CH_3)_g + (3H_2)_g$

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```
Catalyser : Pt/Al<sub>2</sub>O<sub>3</sub> in powder form
Temperature: 610 K
Pressure : 1 bar
Enthalpy of hydrogenation: 210 kJ/mol MC (endothermic)
Efficiency of the hydrogenation reaction: 95 $
```

b) Hydrogen burning with excess air equal to λ . (the equivalence ration ϕ equals λ^{-1})

$$H_2 + (\frac{1}{2} O_2 + 1.88 N_2) \longrightarrow H_2 O_2 + (\lambda - 1) \frac{1}{2} O_2 + 1.88 N_2$$

Enthalpy of burning: 240 kJ/mol H₂

The value of λ is selected as 2.17 since a hydrogen/air mixture with 2 \$ H₂ lies below the explosion limit and thereby a "normal" burning of hydrogen can be achieved. At the same time the burning has a high efficiency. Thus for $\lambda = 2.17$ the following equation is obtained:

$$H_2 + 2.17 (\frac{1}{2} O_2 + 1.88 N_2) \longrightarrow H_2 O + 0.58 O_2 + 4.07 N_2$$

The maximum theoretical combustion temperature for this reaction can be calculated as (specific heats see table 8).

1'600	K	(constant	pressure)
1'959	ĸ	(constant	temperature)

<u>Note</u>: According to (1) the engine exhaust gas temperature decreases significantly with operation at a low equivalence ratio **6**.

9.2 Heat exchanger system

The dehydrogenation of methylcyclohexane takes place in the gaseous form. To obtain a good energy extraction, a heat exchanger is used in which the enthalpy (heat) of the products (exhaust gas

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Another heat exchanger provides the reaction heat of dehydrogenation.

9.3 Energy flow in the system.

The total energy requirements of the dehydrogenation system is made up as follows:

$$\Delta H_{(to)}^{\prime} = \int_{298}^{T_{B}(NC)} C_{p} \left\{ NC \right\}^{dt} + \Delta H_{B} \left\{ NC \right\}^{-1} + \int_{T_{B}(NC)}^{610} C_{p} \left(NC \right)^{-dT} + \Delta H_{D}(NC)$$

Prom this: $\Delta H_{\Sigma}^{f} = 317 \text{ kJ/mol MC}$

This energy requirement ΔH_t is provided by the cooling of the dehydrogenation products ($C_5H_5CH_3 + 3H_2$) and the exhaust gases (EG).

The cooling of the gaseous toluene and hydrogen gives the following amount of heat

$$\Delta H_{(to)}^{T} = \int_{610}^{T_{B}(TO)} C_{P} (TO) dt + \Delta H_{B}(TO) + \int_{T_{B}}^{327} C_{P} (TO) dT + 3 \int_{610}^{327} C_{P} (H_{2}) dT$$

$$\Delta H_{(to)} = 101 \text{ kJ/mol} (TO + 3H_2)$$

The remaining energy is provided by the exhaust gases as follows:

$$\Delta(\Delta H) = 317 - 101 = 216 \text{ kJ/mol MC}$$

$$\Delta(\Delta H) = 3 \int_{-606}^{1050} \text{Cp} (\text{EJ}) \text{ dT}$$

Table 8: Some thermodynamic values.

<u>Calculation of specific heat Cp</u>

_

T = temperature (K) C_p = a + bT + CT² $\Delta H = \int Cp dT = aT + \frac{b}{2}T^2 + \frac{c}{3}T^3$

		Coefficient	Cp at	1050 K	
	2	b	e	J/gK	J/mol K
H ₂ O	34.39	0.62x10	5.61x10 ⁻⁶	2.307	41.5
0 ₂	28.23	2.54x10 ⁻³	0.54x10 ⁻⁶	1.095	35.0
N2	28.3	2.54x10 ⁻³	0.54x10 ⁻⁶	1.181	33.0
H2	27.7	3.39x10 ⁻³	-	14.655	29.3

Enthalpy of boiling, combustion and dehydrogenation

н _в	(XC)	=	31.5	KJ/mol	MC
н _v	(MC)	*	4500	KJ/mol	MC
н _в	(TO)		31.9	KJ/mol	то
Hv	(H ₂)	=	240	KJ/mol	^H 2
HD	(MC)	±	210	KJ/mol	мс

- B = Boiling
- V = Combustion
- D = Dehydrogenation

Specific heat of MC, TO

Cp (MC)	1	180 J/mol	{MC}	·к	
ср (мс)	:	240 J/mol	(MC)	• к	Boiling points:
ср [то]	=	160 J/mol	(T0)	۰ĸ	т _в (MC) = 374 К
Ср (то)	2	170 J/mol	(TO)	- K	Т _в (то) = 383 к
Cp [Pe]	£	400 J/mol	[Fe]	·к	

Section	Temperature of HC(Kelvin)	Tenperature TO_or_exhau	of st(Kelvin)	Energy flow through the wall
A	HC ₁ 273	PRe Heater MC,		13.5
B	" 347 L	Heater MC	383 	4.7
c	⁷⁶ 9 379	MC Evaporator		27.2
Ù	۰ ۲	Evaporator	10 _g	4.3
ε		Superheater	•	51.6
F	»» 	Superheater	EG	5.8
6	610 610	Reactor		50 210
			TO	

FIG 10 SCHEME OF HEAT FLOW

Section	Function on th Methylcyclohexane	e side: Toluene + H ₂
A	Heat Exchange, liquid	Heat Exchange, liquid
B	Heat Exchange, liquid	Condensor
C	Evaporator	Condensor and separation H_2
D	Evaporator	Heat Exchanger, gaseous
E	Superheater	Heat Exchanger, gaseous
F	Superheater	Heat Exchanger (exhaust gases
G	Dehydrog	enation Catalyser

-

9.4 Energy balance and temperature distribution

To simplify the presentation and calculation the system of heat exchanger and reactor is divided into six sections A to F. (In reality only two parts - heat exchanger and reactor).

It is assumed that the heat exchangers operate in counterflow. To calculate the surface area the following formula are used

 $\Delta T = T_{in} - T_{out}$

$$T_{m} = \frac{(\Delta_{TO} - T_{MC})}{\ln (\Delta T_{TO}/T_{MC})}$$

 $\dot{Q} = \Delta H \times \dot{m} (\dot{m} = mass flow; \frac{mol}{s}) \dot{Q} = power [W]$

$$F = \frac{\dot{Q}}{k \cdot \Delta T_{m}} \begin{bmatrix} m^{2} \end{bmatrix}$$

$$F = \text{surface of the heat exchanger } \begin{bmatrix} m^{2} \end{bmatrix}$$

$$k = \text{heat transfer coefficient } \begin{bmatrix} W/m^{2} & K \end{bmatrix}$$

For the reference case it was assumed that

m = 0.055 mol MC/s

The total surface area of the complete heat exchange system is $\Sigma F = 2.03 \text{ m}^2$

The Carnot efficiency for an Otto engine with hydrogen is

 $(V = const; T_{max} = 1959 K)$

$$n_{carnot} = \frac{T_{max} - T_{out}}{T_{max}} = \frac{1950 - 1050}{1950}$$

	İ			System co	nponents			······································
	Unit	A liq/liq	B liq/cond	C vap/cond	D vap/gas	E gas/gas	F gas/gas	(]• gas∕gas
Δ ^T TO	к	56	0	condensa- tion	17	210	34	140
∆T _{MC}	ĸ	74	27	0 evaporation	0	212	24	
∆T _m	к	64				211	28	
ĸ	W/m ² K*	340	600	2500	600	14	14	150
$\mathtt{T}_{\mathtt{t}}$	К			9				
٩,	kW	0.75	0.26	1.5	0.24	2,84	0,32	15.2
F	m ²	0,03	0.02	0.06	0.03	1.03	0,86	0.73

÷

Table 9: Calculation of the surface area

* See "Wärme-Atlas" 1974.





System components (see table 9)

For this paper it is assumed for the hydrogen Otto engine: $n_{eff} = 0.25$

On the relationship of n_{eff} to the Carnot efficiency is: $n_{eff}/n_{Carnot} = 0.25/0.462 = 0.54$

which appears reasonable.

9.5 Possible layou of the heat exchanger and reactor.

The surface area of the heat exchanger is about 2 m^2 . With a tube cross section of 0.002 m^2 (ø 5 cm) a total length of 12.8 m is needed. The heat exchanger could possibly be dimensioned as follows (in a cube of 0.7 m) (Figure 11 a shows a proposed layout).

The inner cube has a side 0.5 m long. To that is added 0.1 m of insulation with a thermal conductivity $\lambda = 0.1$ W/m.K.

```
This gives a power loss of
```

ò	-	<u> F·λ·ΔT</u>
^w loss	-	

F ≠	$6a^2 \neq 3m^2$	surface of the insulated cube
λ =	0.1 W/mK	thermal conductivity of the insulation
d ≠	0.1 m	insulation thickness
ΔТ =	50 K	T _{TO_{out} - T_{air}}
Q _{loss}	= 150 W	calculated loss of heat exchanger

The catalytic reaction unit can be designed as follows

(R = Radius, (m), L = length, (m)). $R_i = 0.1 \text{ m}$ $\Delta R = R_a - R_i$ $T_a = temperature at A$ $R_a = 0.2 \text{ m}$ $\Delta R = wall thickness; T_i = temperature at I$ L = 0.4 m $\Delta T = T_i - T_a$

FIG 12. GEOMETRY AND ISOLATION OF DEHYDROGENATION REACTOR

a) The geometry of the dehydrogenation reactor



b) The isolation of the dehydrogenation reactor



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The reactor volume without isolation is about 13 litres (see also table 4).

The power loss of the reaction unit is

$$\dot{Q}_{loss} = \frac{T_a - T_i}{\ln (R_a/R_i)} \cdot 2\pi \cdot \lambda \cdot L + 2 \left[\frac{R_a + R_i}{2}\right] \pi \frac{\lambda \cdot \Delta T}{R} \stackrel{\sim}{=} 30 W$$

The total losses of the combined system is therefore \sim 180 W.

9.6 Dynamic behaviour (Acceleration and cold starts).

Acceleration requires a power of 20 kW(me) and could be provided by

- a) direct injection of toluene into the engine (excess air;) or
- b) by additional batteries which are charged up during the journey.

```
a) TO Injection
```

- Burning during normal travel

 $H_2 + 2.17 (0.50_2 + 1.88 N_2) \longrightarrow H_20 + 0.580 O_2 + 4.08 N_2 + \Delta H$ $\Delta H = 240 \text{ kJ}$

Burning during acceleration with addition of 0.06 mol T0 (e.g. 5.9 g T0)
 H₂ + 2.17 (0₂ + 1.88 N₂) + 0.06 T0 - 1.42 H₂0 + 0.42 CO₂ + 4.08 N₂

 $\Delta H = 504 \text{ kJ}$

i.e. with the injection of only 5.9 g of TO the thermal power is more than doubled!

However the mechanical power does not increase by the same amount since the efficiency falls.

The advantages of this system lies in the availability of TO from the tank. (Note: With metal hydrides this advantage is not present).

The disadvantages of TO injections also for cold starts are as follows

- larger environmental impact from CO_2 and perhaps NO_x (nevertheless still fifty times smaller than with the gasoline engine)
- lower efficiency of the engine
- increased corrosion of the heat exchanger system
- burning of the hydrogen carrier which comes from liquid fossile fuel at a rate of about 2 % per fuel tank filling. That means after about 50 fillings the contents of the tank which is supposed to be recycled is completely exhausted and requires replacement.
- b) Cold starts

The energy required is to supply the heat for: the heating of the heat exchange (HE) system $Q_{HE} = \Sigma C p \cdot \Delta T \cdot m_{HE}$ and the thermal energy necessary to start the release of hydrogen.

It is arbitrarily assumed that
$$m_{tot} = 1.5 m_{HE}$$
.
 $m_{tot} = (F \cdot d \cdot p) \times 2.5$; $p = 7800 \text{ kg/m}^3$; $d = 2 \text{ mm}$
 $F = \text{surface } (m^2)$
 $Cp = 0.4 \text{ kJ/kg} \text{ (iron)}$
 $\Delta T = \sqrt{T_{max} \times T_{min}} - T_u = \sqrt{1000 \times 273} - T_u = 250 \text{ °C}$
 $T_u = \text{temperature of the surroundings}$

The heat needed for a cold start is:

 $Q_{\rm HE} = 78 \, \rm kJ$

This heat can be produced either by burning toluene of by external electrical heating.

By burning 0.36 mol or 35 g TO the heat exchanger is brought to operating temperature and 3 mols of hydrogen produced. Thus a cold start needs about 0.04 % of the fuel tank contents. After 50 starts 2 % of the tank's contents are directly burnt. This amount must be replaced continuously in the MTH system.

c) Warm start

Energy required 317 kJ/mol MC thus 0.28 mol TO or 28 g TO must be burnt.

The loss of the heat transfer system of 180 W (see above) can be covered by a normal car battery. Additional equipment is necessary

- TO exhaust (as well as the hydrogen exhaust)
- process controller (microprocessor)

d) Electrically driven acceleration (using batteries)

Advantages:

- constant efficiency
- higher engine usage
- small environmental impact

Disadvantages:

- the car weighs more, requires a high fuel consumption
- additional equipment is necessary (batteries, electrical motor)

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9.7 Calculation of the increased fuel required

The following assumptions have been made (see tabl? 1)

Gasoline - ful. tank	16 kg	average	8 kg
MTH system " "	85 kg		84 kg
Difference			76 kg
Tank for MTH system	30 kg		
Catalyser reactor	20 kg		
Other extras	20 kg		70 kg
Total additions			146 kg

In relation to the weight of the car (900 kg)

 $\frac{900 + 146}{900} = 116$

Assuming 1 3 increase in weight corresponds to an increase 0.9 3 in fuel, required i.e. 16 3 x 0.9 = 15 5. After iterating one obtains an increase in the fuel requirements of 18 5.

9.8 Efficiency of the hydrogen engine.

Based on numerous theoretical and practical studies the hydrogen engine is much more efficient than the equivalent gasoline engine.

Van Vorst (1974) writes: "Comparison of results (for the hydrogen and gasoline engine) indicates that thermal efficiency is significantly higher. An increase in efficiency from 25 \$ to 100 \$ were noted; the greatest increase being obtained at the lower power outputs. At the highest gasoline engine efficiency the efficiency of the hydrogen engine is 25 \$ greater".

H. May (1976) writes: "For the higher hydrogen mixtures a higher

efficiency is obtained as is obtained from the lower hydrogen throughput, which can be expected from the theoretical calculations".

9.9 Electrolysis of water

One of the aims of this paper is to discuss the feasability of small scale production of hydrogen for the hydrogenation of toluene to methylcyclohexane. This hydrogen is produced from the electrolysis of water using electrical energy from a nuclear power station.

At the present time (according to ref. 20) all the large scale water electrolysis plants are built as multi-module systems with a module representing 1-2 MW (el).

T = 80 °C, investment cost 250 \$/kW(el)

Today's modern electrolysers have an electric efficiency of 75-80 %.

A promising method of water electrolysis is based on a solid polymer electrolite membrane, unfortunately having a high price per unit cell but nevertheless a high energy efficiency (targets of 90 % are aimed at). Operating temperature is 150 °C; the module system with 8-10 MW(el).

According to ref. 18 the price of hydrogen produced by electrolysis is dependent on the price of electricity and the plant capacity factor (as shown in figure 13).

According to ref. 21 the future development of electrolysis of water has the following goals:

Overall system «	efficiency	85	- 90 %
System capital o	cost	50	\$/kW(el)
Life		20	years
Scale up: Demo a	system	5	MW(el)

Temperature		150 °C	
Pressure		20 bar	
Estimated costs			
(1 US \$ = 2 SFr.)		
1976	Electricity	9.5 SFr./GJ(el)	
	Hydrogen without sale of O ₂	10.7 SFr./GJ(ch) in	^{n H} 2

.



FIG B. ELECTROLYSIS OF WATER



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