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COOLOD-N2 : A COMPUTER CODE, FOR THE  
ANALYSES OF STEADY-STATE THERMAL-  
HYDRAULICS IN RESEARCH REACTORS

March 1994

Masanori KAMINAGA

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COOLOD-N2: A Computer Code, for the Analyses  
of Steady-state Thermal-hydraulics  
in Research Reactors

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The COOLOD-N2 code provides a capability for the analyses of the steady-state thermal-hydraulics of research reactors. This code is revised version of the COOLOD-N code, and is applicable not only for research reactors in which plate-type fuel is adopted, but also for research reactors in which rod-type fuel is adopted. In the code, subroutines to calculate temperature distribution in rod-type fuel have been newly added to the COOLOD-N code. The COOLOD-N2 code can calculate fuel temperatures under both forced convection cooling mode and natural convection cooling mode as well as COOLOD-N code. In the COOLOD-N2 code, a "Heat Transfer package" is used for calculating heat transfer coefficient, DNB heat flux etc. The "Heat Transfer package" is subroutine program and is especially developed for research reactors in which plate-type fuel is adopted. In case of rod-type fuel, DNB heat flux is calculated by both the "Heat Transfer package" and Lund DNB heat flux correlation which is popular for TRIGA reactor. The COOLOD-N2 code also has a capability of calculating ONB temperature, the heat flux at onset of flow instability as well as DNB heat flux.

Keywords: COOLOD-N, COOLOD-N2, DNB, Flow Instability, Forced Convection, Natural Convection, ONB, Plate-type Fuel, Research Reactor, Rod-type Fuel, Steady-state, Thermal-hydraulic

研究炉の定常熱水力解析コード COOLOD-N2

日本原子力研究所東海研究所研究炉部  
神永 雅紀

(1994年2月15日受理)

本報告書は、研究炉の定常熱水力解析コード COOLOD-N2 について述べたものである。本コードは、板状燃料を使用する研究炉の定常熱水力解析コード COOLOD-N の改良版であり、棒状燃料を使用した研究炉の解析が行えるように棒状燃料の温度計算サブルーチンを新たに COOLOD-N コードに組込んだものである。COOLOD-N2 は、強制循環冷却及び自然循環冷却のいずれの場合にも適用可能である。本コードにおいても COOLOD-N コードと同様に板状燃料を用いた研究炉用に開発された熱伝達相関式、DNB 熱流束相関式等からなる「熱伝達パッケージ」が組込まれているが、棒状燃料の DNB 熱流束計算では、熱伝達パッケージの他に TRIGA 炉の解析で用いられている Lund の相関式による値も合わせて計算するようにした。この他には、熱水力設計限界等の判定に重要な、ONB 温度、流動不安定（Flow Instability）などの計算機能も有している。



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**Nomenclature**

<i>A</i>	Flow area ( $\text{m}^2$ )
<i>A<sub>H</sub></i>	Heated area ( $\text{m}^2$ )
<i>C<sub>p</sub></i>	Specific heat ( $\text{kJ/kg K}$ )
<i>D<sub>H</sub></i>	Equivalent heated diameter (m)
<i>D<sub>e</sub></i>	Equivalent hydraulic diameter (m)
<i>F, f</i>	Friction loss coefficient (-)
<i>F<sub>b</sub></i>	Bulk temperature rising factor (-)
<i>F<sub>B</sub></i>	Bond temperature rising factor (-)
<i>F<sub>f</sub></i>	Film temperature rising factor (-)
<i>F<sub>F</sub></i>	Fuel meat temperature rising factor (-)
<i>F<sub>W</sub></i>	Clad temperature rising factor (-)
<i>G</i>	Mass flow rate ( $\text{kg/m}^2\text{s}$ )
<i>G'</i>	Dimensionless mass flow rate = $\frac{G}{\sqrt{\lambda \rho_s g (\rho_t - \rho_s)}}$
<i>g</i>	Acceleration of gravity ( $\text{m/s}^2$ )
<i>h</i>	Heat transfer coefficient ( $\text{kW/m}^2 \text{K}$ )
<i>h<sub>fg</sub></i>	Heat of vaporization ( $\text{kJ/kg}$ )
$\Delta h_i$	Inlet subcooled enthalpy ( $\text{kJ/kg}$ )
<i>k</i>	Thermal conductivity ( $\text{kW/m K}$ )
<i>L</i>	Flow channel length (m)
<i>L<sub>H</sub></i>	Heated length (m)
<i>Nu</i>	Nusselt number
<i>P</i>	Pressure ( $\text{kg/cm}^2 \text{ abs}$ )
<i>P<sub>c</sub></i>	Critical pressure ( $\text{kg/cm}^2 \text{ abs}$ )
<i>Pe</i>	Peclet number (-)
<i>P<sub>H</sub></i>	Heated perimeter (m)
<i>Pr</i>	Prandtl number (-)
<i>q</i>	Heat flux ( $\text{kW/m}^2$ )
<i>q'</i>	Dimensionless heat flux = $\frac{q}{h_{fr} \sqrt{\lambda \rho_s g (\rho_t - \rho_s)}}$
<i>q̄</i>	Heat generation rate (kW)
<i>Re</i>	Reynolds number (-)
<i>T</i>	Temperature ( $^\circ\text{C}$ )
<i>V, v</i>	Velocity (m/s)
<i>W</i>	Width of channel (m)
<i>x</i>	Quality (-)
<i>y</i>	Thickness (m)
<i>Z</i>	Distance from inlet of channel (m)
<i>β</i>	Volumetric expansion coefficient ( $1/\text{K}$ )
<i>ε</i>	Surface roughness (m)
<i>σ</i>	Surface tension (N/m)
<i>λ</i>	Characteristic length = $\sqrt{\frac{\sigma}{(\rho_t - \rho_s)g}} \text{ (m)}$
<i>μ</i>	Dynamic viscosity ( $\text{Pa}\cdot\text{s}$ )
<i>ν</i>	Kinematic viscosity ( $\text{m}^2/\text{s}$ )
<i>ρ</i>	Density ( $\text{kg/m}^3$ )

$\zeta$	Resistance coefficient due to geometry change (-)
$\eta$	Bubble detachment parameter (-)
<b>Subscript</b>	
<i>b</i>	Bulk
<i>B</i>	Bond
<i>c</i>	critical heat flux condition
<i>DNB</i>	Departure from Nucleate Boiling
<i>f</i>	Film
<i>g</i>	Steam
<i>l</i>	Liquid
<i>in</i>	Inlet
<i>out</i>	Outlet
<i>ONB</i>	Onset of Nucleate Boiling
<i>s</i>	Saturated
<i>sub</i>	Subcooled
<i>U</i>	Fuel meat
<i>W</i>	Clad or wall

## 1. Introduction

In Japan Atomic Energy Research Institute (JAERI), COOLOD-N code was developed for steady state thermal-hydraulic analysis of research reactors in which plate type fuel is employed<sup>[1]</sup>, especially for steady state thermal-hydraulic analysis under natural convection cooling, based on COOLOD code<sup>[2]</sup>. Thermal-hydraulic analyses of the JRR-3M<sup>[3]</sup>, RSG-GAS in Indonesia<sup>[4]</sup>, MEX-15 planning in Mexico<sup>[5]</sup>, etc. have been performed, using the COOLOD-N code. COOLOD-N2 code is a revised version of the COOLOD-N code. The COOLOD-N2 is developed based on the COOLOD-N code and provides a capability for the analysis of the steady-state thermal-hydraulics of research reactors. The COOLOD-N2 is applicable not only for research reactors in which plate-type fuel is adopted, but also for research reactors in which rod-type (pin-type) fuel is adopted. In the code, subroutines to calculate temperature distribution in rod-type fuel have been newly added to the COOLOD-N code. The COOLOD-N2 code can calculate fuel temperatures under both forced convection cooling mode and natural convection cooling mode as well as COOLOD-N code. In the COOLOD-N2 code, a "Heat Transfer Package"<sup>[6]</sup> which is subroutine program to calculate heat transfer coefficient and DNB heat flux etc., and was especially developed for research reactors in which plate type fuel is adopted, is also adopted, but in case of rod-type fuel, DNB heat flux is also calculated by Lund<sup>[7]</sup> correlation which is popular for TRIGA type fuels. The COOLOD-N2 code also has a capability of calculating ONB temperature, the heat flux at onset of flow instability (for plate-type fuel only) as well as DNB heat flux.

## 2. Description of the COOLOD-N2 code

### 2.1 Fuel plate temperature calculation

Fuel plate temperatures are calculated by assuming that the heat generation in fuel meat is constant along the radial direction and considering one dimensional heat conduction. An axial fuel plate temperature distribution is calculated from local bulk temperatures of the coolant and axial peaking factors. In case of some kinds of fuel plates which have different heat generation rate one another, exist in a fuel element, or right-hand side and left-hand side of the fuel plate cooling conditions are different due to different configuration of coolant channels or different coolant velocities, the code can calculate temperature distribution of each fuel plate. In case of some kinds of fuel elements exist in a core, the code is also able to calculate temperature distribution of each fuel element by using power distribution factors etc..

Given the fuel meat material (choice U-Al-alloy, U-Al<sub>x</sub>-Al) and the uranium density, the code calculates thermal conductivities of the fuel meat. Thermal conductivities of the fuel meat can be also inputted by data table. The properties of light water, heavy water and aluminum alloy are already given in the code.

### 2.2 Fuel rod temperature calculation

Fuel rod (pin) temperatures are calculated by assuming that the heat generation in fuel meat (pellet) is constant along the radial direction and considering one dimensional heat conduction. An axial fuel rod (pin) temperature distribution is calculated from local bulk temperatures of the coolant and axial peaking factors. In case of some kinds of fuel rods (pins) which have different heat generation rate one another, exist in a fuel element, the code can calculate temperature distribution of each fuel rods (pins). In case of some kinds of fuel elements exist in a core, the code is also able to calculate temperature distribution of each fuel element by using power distribution factors etc..

Thermal conductivities of the fuel meat (pellet) must be inputted by data table. Thermal conductivities of the cladding must be also inputted by data table. The properties of light water, heavy

water are already given in the code as described above

### **2.3 Cooling system temperature calculation**

In addition to the fuel plate temperature calculation, coolant temperatures of the primary and the secondary cooling system can be calculated by the COOLOD-N2 code. In this calculation, heat loss from the surface of piping, heat exchanger and so on are neglected.

Counter flow type cooling tower, and heat exchanger of counter flow type, parallel flow type and shell & tube type are treated in the code.

### **2.4 ONB temperature, Flow instability, DNB heat flux and Pressure drop**

The code has capabilities of calculating the ONB temperature, heat flux at onset of flow instability and DNB heat flux which are important to confirm safety of the reactor. The code also has a capability of calculating pressure drops and local pressures in the core which are required to calculate above value. As flow direction in the core, downward flow, upward flow and horizontal flow are treated in the code.

### **2.5 Natural convection cooling**

In general, pool type research reactors have a natural convection cooling mode as well as a forced convection cooling mode. In the natural convection cooling mode, the core flow is an upward flow, which is supplied by the downflow through a natural circulation valve, through a core bypass and so on. The driving force for the natural circulation is calculated by the difference between the outlet water density of the core flow heated by core power and the inlet water density through a core bypass or through a natural circulation valve, in the COOLOD-N2 code. See section 3.6.

### **2.6 Heat transfer package**

A "Heat Transfer Package" is a sub-program for calculating heat transfer coefficient, ONB temperature, heat flux at onset of flow instability and DNB heat flux. The "Heat transfer package" was especially developed for research reactors which are operated under low pressure and low temperature conditions using plate-type fuel, just like as the JRR-3M<sup>[1]</sup>. Heat transfer correlations adopted in the "Heat Transfer Package" were obtained or estimated based on the heat transfer experiments in which thermal-hydraulic features of the upgraded JRR-3 core were properly reflected. The "Heat Transfer Package" is applicable to, not only upward flow, but also downward flow. See section 3.3.

## **3. Calculation models**

### **3.1 Calculation model for temperature distribution in fuel plates**

Assuming that the heat generation in fuel meat is constant along the radial (thickness) direction ( $\dot{q}_U = q_U / v_U = \text{constant}$ ), and considering one dimensional heat conduction, temperature distribution in fuel plates are calculated as follows. Figure 1 shows calculation model of temperature distribution in fuel plates.

#### **(1) Coolant bulk temperature $T_b$**

$$T_b = T_{in} + F_b \frac{1}{G \cdot A \cdot C_p} \int_0^L Q(Z) dZ \quad (3.1.1)$$

(2) Clad outer surface temperature :  $T_{W1}$ 

$$\bar{T}_{W1} = T_b + F_{W1} \frac{q_w}{h_w} \quad (3.1.2)$$

$q_w = q_r$

(3) Clad inner surface temperature :  $T_{WB1}$ 

$$T_{WB1} = T_{W1} + F_{WB1} \frac{q_w \cdot y_w}{k_w} \quad (3.1.3)$$

(4) Fuel meat surface temperature :  $T_{BU1}$ 

$$T_{BU1} = T_{WB1} + F_{BU1} \frac{q_w \cdot y_B}{k_B} \quad (3.1.4)$$

(5) Fuel meat maximum temperature :  $T_{t\eta}$ 

$$T_{t\eta} = T_{BU1} + F_{t\eta} \frac{\dot{q}_r}{2k_r} y_{t\eta}^2 \quad (3.1.5)$$

$\dot{q}_r = \dot{q}_r(y_{t\eta})$

If the cooling condition of right hand side and left hand side of the fuel plate are different, the COOLOD-N2 code calculates a fuel meat maximum temperature until the fuel meat maximum temperature of right hand side and left hand side are equal by changing the location of maximum temperature point. If the cooling conditions of right hand side and left hand side of the fuel plate are equal, then the fuel maximum temperature appears center of the fuel meat.

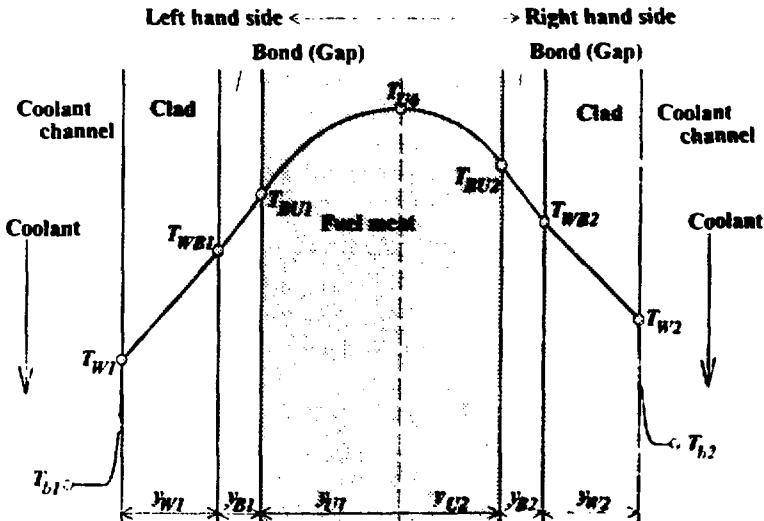


Fig. 1 Fuel plate temperature calculation model

### 3.2 Calculation model for temperature distribution in fuel rods

Assuming that the heat generation in fuel meat is constant along the radial direction ( $\dot{q}_t = 2q_t / y_t = \text{constant}$ ), and considering one dimensional heat conduction, temperature distribution in fuel rods are calculated as follows. Figure 2 shows calculation model of temperature distribution in fuel rods.

(1) Coolant bulk temperature :  $T_b$

$$T_b = T_m + F_b \frac{1}{G A C_p} \int_0^L Q(Z) dZ \quad (3.2.1)$$

(2) Clad outer surface temperature :  $T_w$

$$\begin{aligned} T_w &= T_b + F_f \frac{q_w}{h_{bf}} \\ q_w &= q_t \frac{2\pi y_t}{2\pi y_w} \end{aligned} \quad (3.2.2)$$

(3) Clad inner surface temperature :  $T_{wB}$

$$\begin{aligned} T_{wB} &= T_w + F_w \frac{q_t y_t}{k_w} \ln \frac{y_w}{y_B} \\ &= T_w + F_w \frac{q_t y_t^2}{2k_w} \ln \frac{y_w}{y_B} \end{aligned} \quad (3.2.3)$$

(4) Fuel meat surface temperature :  $T_{Bt}$

$$T_{Bt} = T_{wB} + F_B \frac{q_t}{h_{wB}} \quad (3.2.4)$$

(5) Fuel meat maximum temperature :  $T_{t0}$

$$\begin{aligned} T_{t0} &= T_{Bt} + F_t \frac{\dot{q}}{4k_t} y_t^{-2} \\ q_t &= \frac{\dot{q}_t y_t}{2} \end{aligned} \quad (3.2.5)$$

### 3.3 Heat transfer calculation model (Heat transfer correlations)

In the COOLOD-N2 code, the COOLOD code original heat transfer correlations as well as the "Heat Transfer Package" which was developed for thermal-hydraulic analysis of research nuclear reactors in which plate-type fuel is employed, can be selected by the input data. Table I shows the COOLOD code original heat transfer correlations.

The "Heat Transfer Package" used in the COOLOD-N2 code has been modified taking into account of the characteristics of rod type fuels and is shown as follows.

(1) Single-phase forced-convection flow  
Downward flow ( $G < 0$ )

$$\text{Nu} = \frac{h D_e}{k} = 4.0 \quad \text{for laminar flow (Re} < 2000) \quad (3.3.1)$$

$$\text{Nu} = 0.023 \text{Re}_b^{0.8} \text{Pr}_b^{0.4} \quad \text{for turbulent flow (Re} \geq 2500) \quad (3.3.2)$$

(Dittus-Boelter correlation<sup>[8]</sup>)

Nusselt number is evaluated by interpolation with Eq.(3.3.1) and (3.3.2) for transition region ( $2000 \leq \text{Re} < 2500$ ).

#### Upward flow ( $G > 0$ )

$\text{Nu} = \max[\text{Eq.(3.3.1), Collier correlation}]$  for laminar flow ( $\text{Re} < 2000$ ) (3.3.3)  
where Collier correlation<sup>[9]</sup> is given as follows.

$$\text{Nu} = 0.17 \text{Re}_f^{0.33} \text{Pr}_f^{0.43} \left\{ \frac{(\text{Pr}_t)_f}{(\text{Pr}_t)_w} \right\}^{0.25} \left\{ \frac{\rho^2 \beta D e^3 g_c (T_w - T_t)}{\mu^2} \right\}_f^{0.1} \quad (3.3.4)$$

Nusselt number is evaluated by Eq.(3.3.2) for turbulent flow region ( $\text{Re} \geq 2500$ ).

Nusselt number is evaluated by interpolation with Eq.(3.3.4) and (3.3.2) for transition region ( $2000 \leq \text{Re} < 2500$ ).

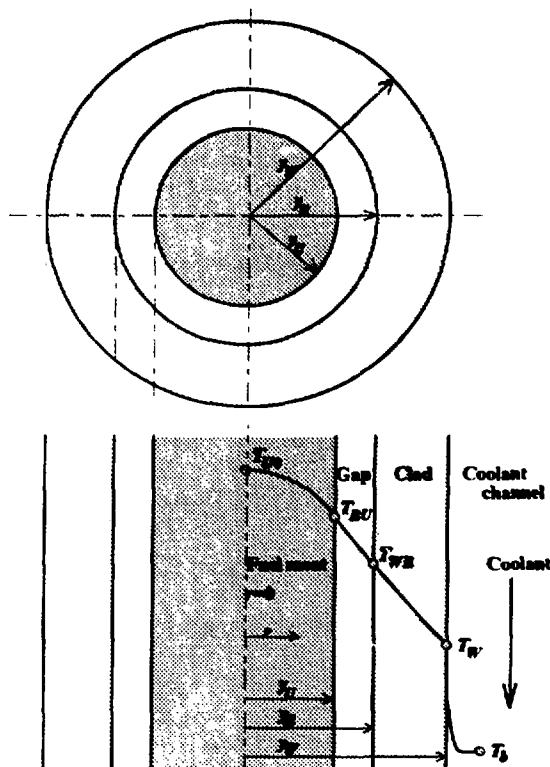


Fig. 2 Fuel rod temperature calculation model

Table 1 COOLOD code original heat transfer correlations

CGS unit

Heat transfer mode		Heat transfer correlation	Note
Single-phase forced-convection		$Nu = H_1 \times (\text{Re}^A - H_2) \times \text{Pr}^B \left[ 1.0 + H_3 \left( \frac{De}{Z} \right)^C \right] \times \left( \frac{\mu_c}{\mu_w} \right)^D$	$A, B, C, D$ and $H_1, H_2, H_3$ are given by input data
ONB temperature		$q = 0.025293 P^{1.156} \left[ \frac{9}{5} (T_{ONB} - T_{sat}) \right]^{\frac{2.1615}{P^{0.0234}}}$	Bergles-Rosenow
Nucleate boiling	Subcooled	$q_t = q_c + q_b$ $q_c = 0.023 \frac{k}{De} \text{Re}^{0.8} \text{Pr}^{0.4} (\Delta T_{sat} + \Delta T_{sub})$ $q_b = 4.50 e^{\frac{P}{20}} \frac{\Delta T_{sat}^{3.6}}{36000}$	Sato-Matsumura
	Saturated	$\Delta T_{sat} = 11.2951 q_t^{0.25} e^{\frac{P}{630}}$	Jens-Lottes
DNB heat flux		$q_{DNB} = 478800 (1 + 0.0365 v) (1 + 0.00507 \Delta T_{sub}) \times (1 + 0.0131 P)$	Mirshak, Durant and Towell IHTC = 2 (CARD G1)
DNB heat flux		$q_{DNB} = \left( 10890 \frac{De}{De + \frac{P_H}{\pi}} + 48 \frac{v}{De^{0.6}} \right) \times (102.6 \ln P - 97.2 \frac{P}{P+15} - \frac{v}{2.22} + 32 - (T_b)_{DNB})$	Bernath IHTC = 3 (CARD G1) $De$ : (ft) $P_H$ : (ft) $(T_b)_{DNB}$ : ( $^{\circ}$ F))
		$q_{DNB} = 145.4 \theta_{(P)} \left[ \frac{1 + 2.5v^2}{\theta_{(P)}} \right]^{\frac{1}{4}} \times \left( 1 + 15.1 \frac{C_p \Delta T_{sub}}{\lambda \sqrt{P}} \right)$ $\theta_{(P)} = 0.99531 P^{\frac{1}{4}} \left( 1 - \frac{P}{P_c} \right)^{\frac{3}{4}}$	Labuntsov IHTC = 1 (CARD G1) $q_{DNB}$ : (w/cm <sup>2</sup> ) $P$ : (bar) $P_c$ : (bar) $v$ : (m/s) $C_p$ : (kJ/kg K)

## (2) Nucleate boiling heat transfer

ONB Temperature (Bergles-Rohsenow correlation<sup>[10]</sup>)

$$q = 911 P^{1.156} \left\{ \frac{9}{5} (T_{ONB} - T_s) \right\}^{\frac{2.16}{P^{0.0234}}} \times \frac{1.163}{1000} \quad (3.3.5)$$

Subcooled nucleate boiling (Modified Chen correlation<sup>[9],[11]</sup>)

$$q = 0.023 \text{Re}_b^{0.8} \text{Pr}_b^{0.4} \frac{k}{De} (T_w - T_f) + 0.00122 \frac{k_f^{0.79} C p_f^{0.46} \rho_f^{0.49} (T_w - T_s)^{1.24} (P_w - P)^{0.75}}{\sigma^{0.5} \mu_f^{0.29} h_{fg}^{-0.24} \rho_g^{0.24}} \quad (3.3.6)$$

where

$$\begin{aligned}
 S &= \frac{1}{1 + 0.12 \text{Re}^{0.14}} & \text{Re}' < 32.5 \\
 S &= \frac{1}{1 + 0.42 \text{Re}^{0.78}} & 32.5 \leq \text{Re}' < 70.0 & \text{Re}' = \frac{|G|De}{\mu_f} \times 10^{-4} \\
 S &= 0.1 & 70.0 \leq \text{Re}' 
 \end{aligned}$$

### Saturated nucleate boiling (Chen correlation<sup>[9],[11]</sup>)

$$q = F 0.023 \left\{ \text{Re}_f (1-x) \right\}^{0.8} \Pr_f^{0.4} \frac{k_f}{De} (T_w - T_s) + S 0.00122 \frac{k_f^{0.79} C p_f^{0.45} \rho_f^{0.49} (T_w - T_s)^{1.24} (P_w - P)^{0.75}}{\sigma^{0.5} \mu_f^{0.29} h_{fr}^{0.24} \rho_g^{0.24}} \quad (3.3.7)$$

where

$$F = 1.0 \quad \frac{1}{x_n} \leq 0.1$$

$$F = 2.35 \left( \frac{1}{x_n} + 0.213 \right)^{0.736} \quad \frac{1}{x_n} > 0.1$$

$$\frac{1}{x_n} = \left( \frac{x}{1-x} \right)^{0.9} \left( \frac{\rho_f}{\rho_g} \right)^{0.5} \left( \frac{\mu_f}{\mu_g} \right)^{0.1}$$

$$S = \frac{1}{1 + 0.12 \text{Re}^{0.14}} \quad \text{Re}' < 32.5$$

$$S = \frac{1}{1 + 0.42 \text{Re}^{0.78}} \quad 32.5 \leq \text{Re}' < 70.0 \quad \text{Re}' = \frac{|G(1-x)|De}{\mu_f} \times 10^{-4}$$

$$S = 0.1 \quad 70.0 \leq \text{Re}'$$

### (3) DNB heat flux<sup>[12],[13]</sup>

$$q_{DNB,1}^* = 0.005 |G^*|^{0.611} \quad (3.3.8)$$

$$q_{DNB,2}^* = \frac{A}{A_H} \frac{\Delta h_L}{h_{fr}} |G^*| \quad (3.3.9)$$

$$q_3^* = 0.7 \left( \frac{A}{A_H} \right) \frac{\sqrt{W/\lambda}}{\left\{ 1 + (\rho_g / \rho_t)^{1.4} \right\}^2} \quad (3.3.10)$$

#### Downward flow ( $G < 0$ )

DNB heat flux is evaluated by min[Eq.(3.3.8), max[Eq.(3.3.9), Eq.(3.3.10)]]

#### Upward flow ( $G > 0$ )

DNB heat flux is evaluated by max[Eq.(3.3.8), Eq.(3.3.10)]

#### DNB heat flux for rod type fuels

In subcooled boiling, the DNB heat flux is a function of the coolant velocity, the degree of subcooling, and the pressure. The correlation used to predict DNB is Lund which was developed from empirical data gathered from an experiment conducted on a test assembly that confirmed to actual fuel

bundle in terms of dimension, flow and heat flux. The critical heat flux is given by<sup>[7]</sup>.

$$q_{Lund} = 0.5 f_c \rho V_{inter} C_p (T_c - T_{out}) \quad (3.3.11)$$

where	$f_c$	: Friction factor for the channel between fuel rods (-).
		= $0.55 Re_{inter}^{-0.37}$
	$Re_{inter}$	: Reynolds number for the interrod channel (-)
		= $2 \rho V_{inter} D_r (S-1) / \mu_{sat}$
	$V_{inter}$	: Interrod channel velocity (m/s).
		= $V [1.0 - 0.98 e^{-2.2(S-1)}]$
	$S$	: Pitch-to-diameter ratio (-)
	$D_r$	: Rod diameter (m)
	$V$	: Average velocity (m/s)
	$\rho$	: Density (kg/m <sup>3</sup> )
	$\mu_{sat}$	: Viscosity at saturation temperature (Pa·s)
	$C_p$	: Constant pressure specific heat (kJ/kg)
	$T_{out}$	: Temperature at outlet of cooling channel (°C)
	$T_c$	: Critical wall temperature (°C)

The critical wall temperature is given by

$$T_c = T_{sat} (1 + 6\sqrt{\theta_c})$$

where	$T_{sat}$	: Saturation temperature (°C)
	$\theta_c$	= $q_c \sigma_{sat} / p \mu_{sat} g h_{fg}$
	$\sigma_{sat}$	: Saturation surface tension (N/m)
	$p$	: Absolute pressure (kg/m <sup>2</sup> abs.)
	$h_{fg}$	: Heat of vaporization (kJ/kg)

#### (4) Heat flux at Onset of Flow Instability

The criterion for the onset of flow instability (flow excursion) has been obtained for rectangular channels by Whittle and Forgan<sup>[14]</sup>.

$$\frac{T_{out} - T_{in}}{T_s - T_{in}} = \frac{1}{1 + \eta \frac{D_H}{L_H}} \quad (3.3.12)$$

Energy balance is given by

$$q A_H = C_p (T_{out} - T_{in}) |G| \quad (3.3.13)$$

From Eq.(3.3.12) and (3.3.13), a following correlation was obtained.

$$q = \frac{1}{A_H} \frac{C_p (T_s - T_{in})}{1 + \eta \frac{D_H}{L_H}} |G| = \frac{C_p \Delta T_{sub}}{A_H + 4 \eta A} |G| \quad (3.3.14)$$

The bubble detachment parameter  $\eta$  was determined empirically to be 25<sup>[14]</sup>.

### 3.4 Pressure drop calculation model

#### 3.4.1 Friction loss coefficient<sup>[15]</sup>

(1) Friction loss coefficient for laminar flow ( $Re \leq 2500$ )

$$F = \frac{Cb}{Re} \quad (3.4.1)$$

where  $Cb$  is a factor which depends on the configuration of the channel.

$Cb = 64.0$	for tube
$Cb = 56.9$	for square
$Cb = 96.0$	for rectangular

(2) Friction loss coefficient for turbulent flow ( $Re > 2500$ )

Following correlations can be selected.

Blasius correlation

$$F = 0.3164 Re^{-0.25} \quad (3.4.2)$$

Kármán-Nikuradse correlation

$$\frac{1}{\sqrt{F}} = 2.0 \log_{10} \left( Re \sqrt{F} \right) - 0.8 \quad (3.4.3)$$

Cole-Brook correlation

$$\frac{1}{\sqrt{F}} = 2.0 \log_{10} \left[ \frac{\varepsilon / De}{3.71} + \frac{2.51}{Re \sqrt{F}} \right] \quad (3.4.4)$$

#### 3.4.2 Pressure drop calculation model

A pressure drop calculation model for the COOLOD-N2 code is shown in Figure 3. In this calculation model, a pressure drop due to friction loss is calculated as a pressure drop inside the segment. A pressure drop due to geometry change is calculated as a pressure drop between segment  $n$  and segment  $n+1$ . A local pressure  $P_{n,1}$  and  $P_{n,2}$  of  $n$ -th segment is calculated as follows by using Bernoulli's theorem.

$$P_{n,1} = P_{n-1,2} + \frac{1}{2g} \left( \bar{\rho}_{n-1} v_{n-1}^2 - \bar{\rho}_n v_n^2 - \rho_n \zeta_n \bar{v}_{n+1}^2 \right) \quad (3.4.5)$$

$$P_{n,2} = P_{n,1} + \bar{\rho}_n \left( L \Delta Z_n - F_n \frac{\Delta Z_n}{De_n} \frac{\bar{v}_n^2}{2g} \right) \quad (3.4.6)$$

where

$\bar{\rho} = \frac{\rho_n + \rho_{n+1}}{2}$	: Average density of the segment $n$
$L$	: Flow direction flag
	= -1 : Upward flow
	= 0 : Horizontal flow
	= 1 : Downward flow

$$\bar{v} = \max(v_n, v_{n-1})$$

and  $P_{0,2} = P_{in}$  is given by input data. In the non-heated channel,  $\rho_n = \rho_{n-1} = \bar{\rho}_n$ .

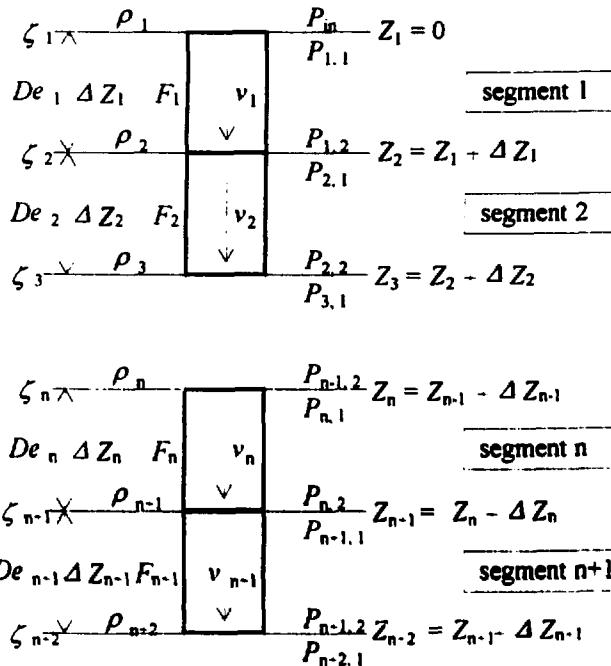


Fig. 3 Pressure drop calculation model for COOLOD-N2 code

### 3.5 Cooling tower and heat exchanger calculation model

#### 3.5.1 Cooling tower temperature calculation model<sup>[14]</sup>

In case of considering a heat exchange between air and water at the cooling tower, transfer unit  $U$  of the cooling tower is expressed as follows.

$$U = \frac{K_a}{G} V \quad (3.5.1)$$

where

$K_a$  : Overall volumetric heat transfer coefficient based on enthalpy difference (kcal/m<sup>3</sup>hΔt)

$G$  : Air flow rate (kg/h)

$V$  : Volume of the cooling tower (m<sup>3</sup>)

Transfer unit  $U$  is also expressed as follows.

$$U = N \int_{in}^{out} \frac{-dT_b}{h_b - h} \quad (3.5.2)$$

where

$N$	: Water air ratio
$h$	: Enthalpy of air (kcal/kg)*
$h_b$	: Enthalpy of saturated air at water temperature $T_b$
$T_b$	: Temperature at the cooling tower (°C)

\* kg' means weight of dry air in the wet air

Inlet and outlet temperatures of the cooling tower are calculated from Eq. (3.5.1) and Eq. (3.5.2) by using a wet-bulb temperature, a water-air ratio  $N$  and dummy inlet and outlet temperature of the cooling tower  $T_{b,in}$ ,  $T_{b,out}$  until Eq. (3.5.1) equal to Eq. (3.5.2) where  $dT_b (T_{b,in} - T_{b,out}) = \text{constant}$ .

### 3.5.2 Heat exchanger temperature calculation model<sup>[17]</sup>

Inlet and outlet temperatures of the primary coolant in a heat exchanger are calculated from the temperature  $T'$  of the secondary coolant.

$$T_{in} = T' + \frac{\Delta T}{E_A} \quad (3.5.3)$$

$$T_{out} = T_{in} - \Delta T \quad (3.5.4)$$

where

$\Delta T$	: Temperature difference between inlet and outlet temperature of primary coolant (°C)
$T_{in}$	: Inlet temperature of primary coolant (°C)
$T_{out}$	: Outlet temperature of primary coolant (°C)
$E_A$	: Exchanger effectiveness

If a heat exchanger type is different, then  $E_A$  has also different value.  $E_A$  is calculated as follows.

#### (1) Counter flow type heat exchanger

$$E_A = \frac{1 - \exp(-(NTU)_A (1 - R_A))}{1 - R_A \exp(-(NTU)_A (1 - R_A))} \quad (3.5.5)$$

#### (2) Parallel flow type heat exchanger

$$E_A = \frac{1 - \exp(-(NTU)_A (1 - R_A))}{1 + R_A} \quad (3.5.6)$$

#### (3) Shell and tube type heat exchanger (Shell side $m$ pass, tube side $2m$ pass)

1)  $m = 1$

$$E_A = \frac{2}{(1 + R_A) + \sqrt{1 + R_A^2} \frac{1 + \exp(-\Gamma)}{1 - \exp(-\Gamma)}} \quad (3.5.7)$$

where

$$\Gamma = (NTU)_A \sqrt{1 + R_A}$$

2)  $m > 2$

$$E_A = \frac{\left( \frac{1 - E_a R_A}{1 - E_a} \right)^m - 1}{\left( \frac{1 - E_a R_A}{1 - E_a} \right)^m - R_A} \quad (3.5.8)$$

where

$R_A$  : Capacity rate ratio of primary coolant and secondary coolant =  $\frac{W_1}{W_2}$

$(NTU)_A$  : Number of transfer unit =  $\frac{UA_H}{W}$

$U$  : Overall heat transfer coefficient ( $\text{kcal}/\text{m}^2\text{h}^\circ\text{C}$ )

$W$  : Heat capacity =  $G A C_p$  ( $\text{kcal}/\text{h}^\circ\text{C}$ )

$A_H$  : Heat transfer area of the heat exchanger ( $\text{m}^2$ )

### 3.6 Natural convection cooling calculation model

In the natural convection cooling model,  $m$  kinds of heated channels and  $n$  kinds of core bypass channels (non-heated channel) are considered in the COOLOD-N2 code. A basic equation used in this calculation model is a equation of conservation of mass between heated channels and non-heated channels.

A sum of mass flow rates  $G_i$  for core bypass channels is equal to a sum of mass flow rates  $G_j$  for heated channels.

$$\sum_{i=1} G_i = \sum_{j=1} G_j = G_0 \quad (3.6.1)$$

On the other hand, the relation between a pressure drop of the heated channel in the core  $\Delta P_{ci}$  ( $i=1$  to  $i_{\max}$ ), a pressure drop of the non-heated channel (core bypass)  $\Delta P_{bj}$  ( $j=1$  to  $j_{\max}$ ), and a driving force  $\Delta P_{di}$  ( $i=1$  to  $i_{\max}$ ) are expressed as shown below.

$$\Delta P_{ci}(G_i) + \Delta P_{bj}(G_j) = \Delta P_{di}(G_i) \quad (3.6.2)$$

$$\Delta P_{bj}(G_j) = \Delta P_b \quad (\text{constant}) \quad (3.6.3)$$

The driving force  $\Delta P_{di}$  for the natural circulation is expressed with the difference between the water density  $\rho'_i$  of heated channel and the water density  $\rho$  through non-heated channel (core bypass), and is shown below.

$$\begin{aligned} \Delta P_{di} &= \int_0^{L_i} (\rho - \rho'_i) dx \\ &= \sum_{m=1}^{m_{\max}} (\rho'_{im} - \rho_{im}) L_{im} \\ &= \rho L_i - \sum_{m=1}^{m_{\max}} \rho'_{im} L_{im} \end{aligned} \quad (3.6.4)$$

where

$$\begin{aligned} L_i &: \text{Heated length of } i\text{-th channel (m)} = \sum_{m=1}^{n_i} \ell_{im} \\ \ell_{im} &: \text{Heated length of } m\text{-th segment of } i\text{-th channel (m)} \end{aligned}$$

The driving force is calculated by the coolant temperature distribution of the heated channel which depends on the core power.

If nucleate boiling would occur in the core, the right hand side of Eq.(3.6.4) will be replaced by following equation.

$$\rho'_{tim} \ell_{im} = (1 - \alpha_{im}) \rho_{tim} \ell_{im} \quad (3.6.5)$$

where

$$\begin{aligned} \rho_{tim} &: \text{Saturated water density of } m\text{-th segment of } i\text{-th heated channel} \\ &\quad (\text{kg/m}^3) \\ \alpha_{im} &: \text{Void fraction of } m\text{-th segment of } i\text{-th heated channel} \end{aligned}$$

In this calculation model, the condition of onset of nucleate boiling is defined as follows<sup>[18]</sup>.

$$Nu_B = \frac{q De}{K_b(T_s - T_b)} \geq 455 \quad ; \quad Pe \leq 7000 \quad (3.6.6)$$

$$St_B = \frac{q}{GCp_b(T_s - T_b)} \geq 0.0065 \quad ; \quad Pe > 7000 \quad (3.6.7)$$

The void fraction is calculated by following correlation.

(1) Void fraction under subcooled boiling region (AHMAD correlation<sup>[19]</sup>)

$$\alpha = \frac{x}{x + s(1-x)\rho_g/\rho_t} \quad (3.6.8)$$

$$s = \left( \frac{\rho_t}{\rho_g} \right)^{0.205} \left( \frac{G D}{\mu_t} \right)^{-0.016} \quad (3.6.9)$$

(2) Void fraction under subcooled boiling region (Zuber correlation<sup>[20]</sup>)

$$\alpha = \frac{x}{1.13 \left( x \frac{\rho_t - \rho_g}{\rho_t} + \frac{\rho_g}{\rho_t} \right) + C_b \frac{\rho_g}{G} \left[ \frac{d(\rho_t - \rho_g)g}{\rho_t^2} \right]^{1/4}} \quad (3.6.10)$$

(3) Void fraction under subcooled boiling region (Combination of Eq.(3.6.9) and Eq.(3.6.10)).

$G < G_{LM}$  then Eq.(3.6.9)

$G \geq G_{LM}$  then Eq.(3.6.10)

where,  $G_{LM}$  (kg/s) is given by input data. The range of  $G_{LM}$  is 500 to 1500 (kg/m<sup>2</sup>s)

(4) Void fraction under nucleate boiling region (Zuber correlation<sup>[21]</sup>)

$$\alpha = \frac{x}{1.13 \left( x_{eq} \frac{\rho_f - \rho_g}{\rho_f} + \frac{\rho_g}{\rho_f} \right) + C_b \frac{\rho_g}{G} \left[ \frac{\sigma(\rho_f - \rho_g)g}{\rho_f^2} \right]^{1/4}} \quad (3.6.11)$$

where

$$x = \frac{x_{eq} - x_{eqB} e^{x_{eq}/x_{eqB}-1}}{1 - x_{eqB} e^{x_{eq}/x_{eqB}-1}} \quad (3.6.12)$$

$$x_{eq} = \frac{q_w p_h Z / (G \cdot A) - C p_b (T_s - T_{bB})}{h_{fr}}$$

$x_{eqB}$	Quality at the point of onset of nucleate boiling
$T_{bB}$	Coolant temperature at the point of onset of nucleate boiling (°C)
$Z$	Distance from the point of onset of nucleate boiling (m)
$p_h$	Heated perimeter (m)
$C_b$	Zuber's coefficient = 1.18 or 1.41

## 4. Properties used in the code

### 4.1 Thermal conductivities of fuel meat (plate-type-fuel)

Given the fuel meat material (choice U-Al-alloy, U-Al-Al) and the uranium density, the thermal conductivity of the fuel meat is calculated by the code. Thermal conductivities used in the code are shown below.

#### (1) Thermal conductivity of U-Al alloy<sup>[22]</sup> : $k_{u0}$

$$k_{u0} = 0.415 - 1.0 \times 10^{-4} T_u \quad ; \quad (20 < T_u < 640 \text{ °C})$$

$$k_{u0} = 0.135 \quad ; \quad (T_u > 640 \text{ °C})$$

where

$$k_{u0} \quad \text{Thermal conductivity of U-Al alloy (cal/s cm°C)}$$

$$T_u \quad \text{Temperature of U-Al alloy ( °C)}$$

#### (2) Thermal conductivity of U-Al<sub>x</sub> dispersion fuel<sup>[23]</sup> : $k_{u1}$

$$k_{u1} = k'_{u1} (1-P)^{3/2}$$

$$k'_{u1} = 2.16546 - 2.765x$$

where

$$k_{u1} \quad \text{Thermal conductivity of U-Al<sub>x</sub> dispersion fuel (W/cm°C)}$$

$$x \quad \text{Weight fraction of uranium in the fuel meat} = \frac{\rho}{0.8 \rho + 2.7(1-P)}$$

$$\rho \quad \text{Uranium density of U-Al<sub>x</sub> dispersion fuel}$$

$$P \quad \text{Porosity}$$

Thermal conductivities of the fuel meat can be also inputted by data table.

#### 4.2 Thermal conductivities of aluminum<sup>[24]</sup> (plate-type-fuel) : $k_{Al}$

$$k_{Al} = 0.390 + 2.22 \times 10^{-4} T_{Al} - 3.79 \times 10^{-7} T_{Al}^2 + 2.42 \times 10^{-10} T_{Al}^3$$

. ( $20 < T_{Al} < 649$  °C)

$$k_{Al} = 0.170$$

. ( $T_{Al} > 649$  °C)

where

$T_{Al}$  : Temperature of aluminum clad (°C)

#### 4.3 Thermal conductivities of bond layer<sup>[25]</sup> (plate-type-fuel) : $k_B$

$$k_B = 0.123804 \times 10^{-4} - 0.593896 \times 10^{-7} - 0.37228 \times 10^{-10} T_B$$

; ( $18 < T_B < 520$  °C)

where

$T_B$  : Temperature of bond layer (°C)

As for the thermal conductivity of bond layer, the thermal conductivity of Xe is used in the code

#### 4.4 Properties of light water and heavy water<sup>[26],[27]</sup>

The properties of light water, heavy water used in the code are listed in Table 2 and Table 3

**Table 2 Properties of Light Water**

Temp (°C)	Specific weight (kg/m <sup>3</sup> )	Specific heat (kcal/kg°C)	Kinematic viscosity x 10 <sup>-6</sup> (m <sup>2</sup> /s)	Thermal conductivity (kcal/mh°C)	Thermal diffusivity x 10 <sup>-7</sup> (m <sup>2</sup> /h)	Dynamic viscosity x 10 <sup>-3</sup> (kg s/m <sup>2</sup> )	Surface tension x 10 <sup>-3</sup> (kg/m)	Saturated pressure (kg/cm <sup>2</sup> )	Enthalpy (kcal/kg)
								Saturated water	Saturated vapor
0	999.9	1.008	1.79	0.489	4.85	1.829	7.72	0.006228	0.001 <sup>1</sup> 597.49 <sup>1</sup>
10	999.7	1.002	1.31	0.505	5.04	1.336	7.56	0.012512	10.030 601.87
20	998.2	0.999	1.00	0.518	5.08	1.022	7.39	0.023826	20.030 606.23
30	995.7	0.998	0.803	0.531	5.34	0.816	7.24	0.043251	30.014 610.57
40	992.3	0.998	0.668	0.543	5.48	0.676	7.08	0.075204	39.995 614.88
50	988.1	0.999	0.555	0.552	5.59	0.559	6.90	0.12578	49.980 619.13
60	983.2	1.000	0.480	0.562	5.72	0.482	6.74	0.20313	59.972 623.32
70	977.8	1.001	0.417	0.571	5.85	0.416	6.55	0.31776	69.975 627.43
80	971.8	1.003	0.368	0.578	5.93	0.365	6.37	0.48294	79.993 631.45
90	965.3	1.005	0.328	0.583	6.01	0.323	6.19	0.71491	90.031 635.36
100	958.4	1.007	0.297	0.586	6.08	0.290	6.00	1.03323	100.092 639.15
120	943.1	1.014	0.247	0.589	6.16	0.238	5.55	2.0246	120.311 646.31
140	926.1	1.023	0.209	0.588	6.21	0.197	5.10	3.6850	140.705 652.78
160	907.3	1.037	0.186	0.585	6.22	0.172	4.65	6.3025	161.334 658.43
180	886.9	1.054	0.168	0.578	6.25	0.152	4.17	10.224	182.267 663.10
200	864.7	1.075	0.155	0.568	6.11	0.137	3.70	15.855	203.585 666.60
220	840.3	1.102	0.146	0.544	5.98	0.125	3.24	23.656	225.393 668.75
240	814	1.136	0.139	0.537	5.81	0.115	2.78	34.138	247.827 669.30
260	784	1.183	0.133	0.517	5.57	0.106	2.32	47.869	271.076 667.91
280	751	1.250	0.128	0.493	5.25	0.098	1.85	65.486	295.414 664.09
300	712	1.36	0.13	0.462	4.77	0.091	1.40	87.621	321.261 657.07
320	667	1.54	0.13	0.423	4.12	0.083	0.95	115.12	349.337 645.76

<sup>1</sup> 1.001°C**Table 3 Properties of Heavy Water**

Temp (°C)	Specific weight (kg/m <sup>3</sup> )	Specific heat (kcal/kg°C)	Kinematic viscosity x 10 <sup>-6</sup> (m <sup>2</sup> /s)	Thermal conductivity (kcal/mh°C)	Thermal diffusivity x 10 <sup>-7</sup> (m <sup>2</sup> /h)	Dynamic viscosity x 10 <sup>-3</sup> (kg s/m <sup>2</sup> )	Surface tension x 10 <sup>-3</sup> (kg/m)	Saturated pressure (kg/cm <sup>2</sup> )	Enthalpy (kcal/kg)
								Saturated water	Saturated vapor
0	1105	1.015	0.7444	0.4782	4.266	0.7556	7.72	0.006954	1.201 <sup>1</sup> 554.65 <sup>1</sup>
10	1105	1.012	1.278	0.4882	4.364	1.319	7.56	0.01063	6.270 556.73
20	1105	1.009	1.135	0.5031	4.515	1.168	7.39	0.02067	16.373 560.83
30	1103	1.006	0.9044	0.5159	4.651	0.928	7.24	0.03827	26.483 564.87
40	1100	1.003	0.7297	0.5268	4.774	0.746	7.08	0.06780	36.477 568.86
50	1196	1.001	0.6034	0.5360	4.885	0.616	6.90	0.1153	46.494 572.78
60	1091	0.9991	0.5105	0.5434	4.985	0.520	6.74	0.1890	56.492 576.60
70	1085	0.9974	0.4405	0.5493	5.076	0.448	6.55	0.2994	66.473 580.37
80	1078	0.9959	0.3864	0.5537	5.157	0.392	6.37	0.4600	76.440 584.07
90	1071	0.9946	0.3438	0.5568	5.229	0.349	6.19	0.6871	86.393 587.68
100	1062	0.9937	0.3095	0.5586	5.291	0.314	6.00	1.001	96.331 591.22
120	1044	0.9932	0.2584	0.5587	5.387	0.262	5.55	1.984	116.189 598.00
140	1024	0.9959	0.2225	0.5547	5.438	0.226	5.10	3.641	136.061 604.38
160	1003	1.003	0.1963	0.5470	5.343	0.201	4.65	6.266	156.036 610.28
180	981.5	1.018	0.1765	0.5359	5.361	0.183	4.17	10.22	176.237 615.65
200	959.6	1.044	0.1611	0.5216	5.209	0.172	3.70	15.92	196.833 620.35
220	938.1	1.083	0.1489	0.5044	4.966	0.164	3.24	23.84	219.016 623.27
240	917.1	1.140	0.1390	0.4841	4.630	0.144	2.78	34.51	239.985 623.46
260	897.0	1.220	0.1368	0.4607	4.210	0.163	2.32	48.47	262.938 623.01
280	878.0	1.328	0.1339	0.4339	3.722	0.168	1.85	66.31	287.036 619.85
300	860.3	1.470	0.1180	0.4034	3.191	0.177	1.40	88.72	312.704 612.14
320	844.1	1.652	0.1129	0.3688	2.645	0.190	0.95	116.50	340.860 598.43

<sup>1</sup> 1.50°C

## 5. Input data information for COOLOD-N2

<CARD A> Title card (A72)

TITL Title for the calculation

<CARD B1> Control card (Free format)

INFORM : Index for input data format (I)

- = 0 : COOLOD original type input data (Plate-type-fuel only)
- = 1 : COOLOD-N, N2 original type input data  
FZ (CARD F4) are defined as points
- = 2 : COOLOD-N, N2 original type input data  
FZ (CARD F4) are defined as segments.

<CARD B2> Control card (Free format)

IAMAX : Number of calculation cases (I) (1=<IAMAX=<10)  
(Number of <Card C>)

IMAX : Number of calculation points in fuel meat radial direction (I) (1=<IMAX=<5)

JMAX : Number of calculation points for fuel plate axial direction (I)  
(1=<JMAX=<21, INFORM = 0, 1 (CARD B1))  
(1=<JMAX=<20, INFORM = 2 (CARD B1))  
(Number of CARD F4)

NMAX : Number of different fuel elements in the core (I) (1=<NMAX=<5)

NPLOT : Plot option of calculation results (I)  
= 0 : No plot  
= 1 : Plot of calculation results

KEY(1) : Option for coolant temperature calculation (I)  
= 0 : Cooling Tower, Heat Exchanger and Fuel temperature calculation  
= 1 : Fuel temperature calculation only (Input data 'Tin' (primary coolant core  
inlet temperature) is required for calculation)  
= -1 : Fuel temperature calculation skip

KEY(2) : Index for flow direction in the core (I)  
= -1 : Upflow  
= 0 : Horizontal flow  
= 1 : Downflow  
= 5 : Natural circulation cooling mode

KEY(3) : Index for coolant (I)  
= 0 : Light water ( $H_2O$ )  
= 1 : Heavy water ( $D_2O$ )

IDMAX : Number of division in cladding region (If IDMAX is positive value, rod type  
fuel calculation model will be selected.) <New>

<CARD C> Thermal-hydraulic parameter (Free format)

QRR : Reactor thermal power (MW) (R)

PFLOW : Primary coolant flow rate or average coolant velocity in the core (R)

- \* If KVELO (CARD G1)=0, then PFLOW is Volumetric flow rate  
( $m^3/min$ )
- \* If KVELO (CARD G1)=1, then PFLOW is Mass flow rate (kg/s)
- \* If KVELO (CARD G1)=2, then PFLOW is Average coolant velocity  
in the core (cm/s)
- \* If INFORM (CARD B1)=0, then PFLOW is Volumetric flow rate  
( $m^3/min$ )

TIN : If KEY(1)=1 then the Primary coolant core inlet temperature (°C) (R)  
       If KEY(1)=0 or -1 then the Wet bulb temperature (°C) (R)  
 DT : Increment of inlet temperature "TIN" (°C) (R)  
 JAMX : Number of calculation cases for "DT" (I) (Normally =1)

## &lt;CARD D&gt; Cooling Tower and Heat Exchanger data (Free format)

SFLOW : Secondary coolant flow rate ( $\text{m}^3/\text{min}$ ) (R)  
 AFLOW : Air flow rate of the cooling tower ( $\text{m}^3/\text{min}$ ) (R)  
 CTKI : Overall heat transfer coefficient of the cooling tower (kcal/ $\text{m}^2\text{h}^\circ\text{C}$ ) (R)  
 HEKI : Heat transportation coefficient of the heat exchanger (kcal/ $\text{m}^2\text{h}^\circ\text{C}$ ) (R)  
 SSCT : Cross sectional area of the cooling tower ( $\text{m}^2$ ) (R)  
 ZCT : Effective height of the cooling tower (m) (R)  
 SSHE : Heat transfer area of the heat exchanger ( $\text{m}^2$ ) (R)  
 IHE : Heat exchanger type (I)  
       = -1 : Counter flow type  
       = 0 : Parallel flow type  
       = m : Shell side m pass and tube side 2\*m pass type

\* CARD D is only used in case of KEY(1)<1(CARD B1)

## &lt;CARD E1&gt; Heat transfer correlation (Free format)

$H_1, H_2, H_3, A, B, C, D, \text{ITWC}$

$H_1-H_3$  and A-D (R) and ITWC (I) are shown below

$$\text{Nu} = <H_1> \times \left( \text{Re}^{A} - <H_2> \right) \times \text{Pr}^{B} \times \left[ 1.0 + <H_3> \left( \frac{Dc}{Z} \right)^C \right] \times \left( \frac{\mu_b}{\mu_w} \right)^D.$$

(Single phase heat transfer correlation)

Nu : Nusselt number (-)  
 Re : Reynolds number (-)  
 Pr : Prandtl number (-)  
 De : Equivalent hydraulic diameter (cm)  
 Z : Distance from inlet of channel (cm)  
 $\mu_b$  : Dynamic viscosity at bulk water temperature (dynes/cm<sup>2</sup>)  
 $\mu_w$  : Dynamic viscosity at wall water temperature  
       (Surface temperature of fuel plate) (dynes/cm<sup>2</sup>)

ITWC : Standard temperature for property (I)  
       = 0 : Properties are evaluated by TWC(0)  
           TWC(0) = (Core inlet temperature + core outlet temperature)/2.0  
       = 1 : Properties are evaluated by TWC(1)  
           TWC(1) = Bulk coolant temperature at Z  
       = 2 : Properties are evaluated by TWC(2)  
           TWC(2) = (TWC(0) + Fuel surface temperature at Z)/2.0  
       = 3 : Properties are evaluated by TWC(3)  
           TWC(3) = (TWC(1) + Fuel surface temperature at Z)/2.0

\* CARD E1 is only used for the case of IHTC = 1-3 (CARD G1), if  
 IHTC = 4, then CARD E1 is not used in the calculation, but dummy  
 data are required even in the case of IHTC = 4.

## &lt;CARD E2&gt; Core flow condition (Free format)

FRATE : FRATE = (Effective flow rate for fuel plates cooling)  
       /(Primary coolant flow rate) (-) (R)  
 VIN : Coolant velocity in the inlet plenum (cm/s) (R)

VOUT : Coolant velocity in the outlet plenum (cm/s) (R)  
 PRESSIN : Core inlet pressure (kg/cm<sup>2</sup>abs) (R)  
 RAMF : Index for straight pipe friction loss for turbulent flow (R)  
     = -1.0 : Blasius correlation  
     = 0.0 : Karman-Nikuradse correlation  
     =  $\varepsilon/D_e$  : Cole-Brook correlation  $\varepsilon/D_e$  is a relative roughness

## &lt;CARD F1&gt; Fuel element title card (A40)

TITLN Title for fuel element

## &lt;CARD F2&gt; Fuel element data (Free format)

NPMX : Number of different fuel plates in this kind of fuel element (I)  
     (Different cooling condition, different configuration) (1=<NPMX=<15)  
     (Number of CARD F51-CARD F53)  
 NFUEL : Number of this kind of fuel elements in the core (R)  
 MA : Index for fuel meat material (I)  
     = 0 : U-Al alloy  
     = 1 : U-Al<sub>x</sub> dispersion type  
     = 2 : Fuel meat properties are inputted by data table (CARD F22)  
 UDENST : Uranium density in meat (g/cm<sup>3</sup>) (R) (For U-Al and U-Al<sub>x</sub> dispersion type fuel)  
 POROTY : Porosity (-) (R) (For U-Al<sub>x</sub> dispersion type fuel)  
 IDPMX : Number of different configuration fuel plates in this kind of fuel element (I)  
     (1=<IDPMX=<5) (Number of CARD F6)  
 IDC MX : Number of different configuration flow channels in this kind of fuel element (I)  
     (1=<IDCMX=<5) (Number of CARD F70, CRAD F74 or CARD F76)  
 EAREA : Effective flow area for this kind of fuel element (cm<sup>2</sup>) (R)  
 FRATEN : Flow rate distribution factor for this kind of fuel element (-) (R)  
 FRATEN = (Flow rate of this kind of fuel element)/(Average flow rate of fuel element)

## &lt;CARD FNEW&gt; Rod type fuel element equivalent hydraulic diameter, rod-pitch ratio data card (Free format) &lt;New&gt;

WID0 : Equivalent hydraulic diameter or D (diameter)/2.0 (cm) (R)  
     (for DNB heat flux calculation)  
 RD0 : Rod diameter (cm) (R)  
 SSR0 : Rod-pitch ratio (Rod-pitch/Rod diameter) (-) (R)  
     (for DNB heat flux calculation for Lund correlation)

## &lt;CARD F221&gt; Fuel pellet thermal conductivity data card. (If rod-type fuel is selected (IDMAX on CARD B2 &gt; 0 and MA =2 on CARD F2, then this card is required.) &lt;New&gt;

N221 : Number of data points (-) (I)  
 T221 : Temperature (°C) (R)  
 K221 : Thermal conductivity for fuel pellet ((W/cm K) (R))

## &lt;CARD F222&gt; Gap heat transfer data card. (If rod-type fuel is selected (IDMAX on CARD B2 &gt; 0 and MA =2 on CARD F2, then this card is required.) &lt;New&gt;

N222 : Number of data points (-) (I)  
 T222 : Temperature (°C) (R)  
 K222 : Thermal conductivity for fuel pellet ((W/cm<sup>2</sup>K))

## &lt;CARD F223&gt; Cladding thermal conductivity data card. (If rod-type fuel is selected (IDMAX on CARD B2 &gt; 0 and MA =2 on CARD F2, then this card is required.) &lt;New&gt;

N223 : Number of data points (-) (I)  
 T223 : Temperature (°C) (R)  
 K223 : Thermal conductivity for fuel pellet ((W/cm K)) (R)

## &lt;CARD F22&gt; Fuel meat data table (Free format)

NUAL : Number of data sets (I)  
 TUAL : Temperature (°C) (R)  
 UAL : Thermal conductivity of the fuel meat (W/cm K)  
 \* If MA<>2(CARD F21), then this card is not required.

## &lt;CARD F3&gt; Hot channel factors (Free format)

FR : Radial peaking factor ( $F_R$  (radial)  $\times F_E$  (uncertainty)) (R)  
 FCCOL : Engineering peaking factor for bulk coolant temperature rise (R) ( $F_b$ )  
 FHFLX : Engineering peaking sub-factor for heat flux (R)  
 (This sub-factor is used in the calculation of DNBR)  
 FFILM : Engineering peaking factor for film temperature rise (R) ( $F_f$ )  
 FCLAD : Engineering peaking factor for clad temperature rise (R)  
 FBOND : Engineering peaking factor for bond temperature rise (R)  
 FMEAT : Engineering peaking factor for fuel meat temperature rise (R)

## &lt;CARD F4&gt; Axial peaking factors (Free format)

FZ : Axial peaking factor (R)  
 \* If INFORM = 0 or 1 (CARD B1), then FZ is defined as a point ( $f(M_i)$ ).  
 \* If INFORM = 2 (CARD B1), then FZ is defined as a segment ( $f(S_i)$ ).  
 \* If INFORM = 0 (CARD B1), then following data are not required  
 In this case, DDZ is calculated as follows.

$$DDZ = HB / (JMAX-1) \quad HB : \text{CARD F6}$$

DDZ : Distance from point, ( $M_i$ ) to point, ( $M_{i-1}$ ) or a segment length (R)  
 \* If INFORM = 1 (CARD B1), then DDZ is distance from  $M_i$  to  $M_{i-1}$   
 ( $DDZ = \Delta Z_i$ ). In this case  $DDZ_{JMAX}$  ( $\Delta Z_{JMAX}$ ) is dummy data.  
 \* If INFORM = 2 (CARD B1), then DDZ is a segment length  
 ( $DDZ = \Delta Z_i$ )  
 \* INFORM = 1 :  $\sum_{j=1}^{JMAX-1} DDZ_j = HB \quad HB : \text{CARD F6}$   
 \* INFORM = 2 :  $\sum_{j=1}^{JMAX} DDZ_j = HB$

ZET : Resistance coefficient at point, ( $M_i$ ). (R) (Normally : = 0.0)  
 \* If INFORM = 2 (CARD B1), then  $f(M_i)$  are calculated as follows.  
 using  $f(S_i)$

$$f(M_1) = 2f(S_1) - f(M_2)$$

$$f(M_2) = f(S_1) + \frac{\Delta Z_1}{\Delta Z_1 + \Delta Z_2} [f(S_2) - f(S_1)]$$

$$f(M_3) = f(S_2) + \frac{\Delta Z_2}{\Delta Z_2 + \Delta Z_3} [f(S_3) - f(S_2)]$$

$$f(M_n) = f(S_{n-1}) + \frac{\Delta Z_{n-1}}{\Delta Z_{n-1} + \Delta Z_n} [f(S_n) - f(S_{n-1})]$$

$$f(M_{n, max}) = 2f(S_{n, max-1}) - f(M_{n, max-1})$$

## &lt;CARD F51&gt; Fuel plate title card (A20)

TITLP : Title for fuel plate

## &lt;CARD F52&gt; Fuel plate data (Free format)

NPLATE : Number of this kind of fuel plates in this kind of fuel element (R)  
 FLOCL : Local peaking factor (R)  
 IDPL : Identity number of fuel plate configuration (I) (See CARD F6)  
 KMX : Index for cooling condition of fuel plate (I)  
     = 1 Right hand side of fuel plate cooling condition and left hand side of fuel plate cooling condition are equal  
     = 2 Right hand side of fuel plate cooling condition and left hand side of fuel plate cooling condition are not equal  
 IPLOT : Plot option for the calculation results (I)  
     = 0 : No plot  
     = 1 : Channel No.1 side calculation results are plotted  
     = 2 : Channel No.2 side calculation results are plotted  
     = 3 : Both of channel No.1 and No.2 sides calculation results are plotted  
           \* Channel No. means ICHL of CARD F53.  
 IOUT : Print out option for pressure, ONB, DNB and Heat flux at onset of Flow instability calculation results (I)  
     = 0 : No print  
     = 1 : Print out of pressure, ONB and DNB calculation results  
           \* If INFORM = 0 (CARD B1), then meaning of IOUT is as follows.  
     = 0 : No print  
     = 1 : Print out of pressure, ONB, DNB and Heat flux at onset of Flow instability calculation results. DNB heat flux is calculated by LABNTSOV correlation  
     = 2 : Print out of pressure, ONB, DNB and Heat flux at onset of Flow instability calculation results. DNB heat flux is calculated by MIRSHAK correlation  
     = 3 : Print out of pressure, ONB, DNB and Heat flux at onset of Flow instability calculation results. DNB heat flux is calculated by BERNATH correlation

## &lt;CARD F53&gt; Coolant channel data (Free format)

ICHL : Identity number of channel configuration (I)  
      (See CARD F70, CARD F74 or CARD F76)  
 NHEAT : Coolant condition (R)  
     = 1.0 : Coolant is heated from one side  
     = 2.0 : Coolant is heated from both sides  
 FRATEC : Flow rate distribution factor for this kind of channel (R)  
     FRATEC = (Flow rate of this kind of channel)/(Average flow rate of channel in this kind of fuel element)  
     \* This card is required KMX (CARD F52) sets.  
     \* CARD F51-CARD F53 are required NPMX (CARD F21) sets.

## &lt;CARD F6&gt; Fuel plate configuration data (Free format)

XA : Half thickness of fuel meat for plate-type fuel (cm) (R)  
     Radius of fuel pellet for rod-type fuel (cm) (R)  
 XB : Distance between fuel meat center and clad inner surface for plate-type fuel (cm) (R) (For plate-type fuel, normally : XA = XB)  
     Gap thickness between pellet and cladding for rod-type fuel (cm) (R)

XC	Distance between fuel meat center and clad outer surface for plate-type fuel (cm) (R) (Half thickness of fuel plate)
YA	Cladding thickness for rod-type fuel (cm) (R)
HA	Width of fuel meat for plate-type fuel (cm) (R)
HB	= 0.0 for rod-type fuel (R)
HC	Distance between inlet of channel and top(bottom) of fuel meat (pellet) (cm) (R)
	Length of fuel meat (fueled region) (cm) (R)
	Distance between outlet of channel and bottom(top) of fuel meat (pellet) (cm) (R)

<CARD F70> Coolant channel configuration data (Free format) (If INFORM = 0 (**CARD B1**), then this card is required.)

YCHI	Gap(thickness) of coolant channel (cm) (R)
XCHI	Width of coolant channel (cm) (R)

<CARD F71> Pressure loss calculation data (Fuel element entrance - plate entrance) (Free format) (If INFORM = 0 (**CARD B1**), then this card is required.)

ZETA(1)	Resistance coefficient of fuel element entrance (STRETCH(1)) (R)
DH(1)	Distance between fuel element entrance and fuel plate entrance (cm) (R)
HDE(1)	Equivalent hydraulic diameter of this region (cm) (R)
AR(1)	Cross sectional area of this region (Flow area) (cm <sup>2</sup> ) (R)

<CARD F72> Pressure loss calculation data (Fuel plate exit - fuel element plug entrance) (Free format) (If INFORM = 0 (**CARD B1**), then this card is required.)

ZETA(2)	Resistance coefficient of fuel element plug entrance (STRETCH(3)) (R)
DH(2)	Distance between fuel plate exit and fuel element plug entrance (cm) (R)
HDE(2)	Equivalent hydraulic diameter of this region (cm) (R)
AR(2)	Cross sectional area of this region (Flow area) (cm <sup>2</sup> ) (R)

<CARD F73> Pressure loss calculation data (Fuel element plug entrance - fuel element exit) (Free format) (If INFORM = 0 (**CARD B1**), then this card is required.)

ZETA(3)	Resistance coefficient of fuel element plug exit (STRETCH(3)) (R)
DH(3)	Distance between fuel element plug entrance and fuel element exit (cm) (R)
HDE(3)	Equivalent hydraulic diameter of this region (cm) (R)
AR(3)	Cross sectional area of this region (Flow area) (cm <sup>2</sup> ) (R)

\* CARD F70 - CARD F73 are required IDC MX (**CARD F21**) sets

\* CARD F1 - CARD F73 are required NMAX (**CARD B2**) sets

<CARD F74> Coolant channel configuration data (Free format) (If INFORM<>0 (**CARD B1**) and KEY(2)<>5 (**CARD B2**), then this card is required.)

YCHI	Gap (thickness) of coolant channel for plate-type fuel (cm) (R)
	Equivalent hydraulic diameter for rod-type fuel (cm) (R)
XCHI	Width of coolant channel for plate-type fuel (cm) (R)
	Effective flow area for one fuel rod for rod-type fuel (cm <sup>2</sup> ) (R)
MSFLW	Number of segments, except fuel plate region <sup>11</sup> . (Number of <b>CARD F75</b> )

<CARD F75> Pressure loss calculation data (Free format) (If INFORM<>0 (**CARD B1**) and KEY(2)<>5 (**CARD B2**), then this card is required.)

ZETA	Resistance coefficient of this region entrance (R)
DH	Length of flow area (cm) (R)

ZLAM : Friction loss coefficient for laminar flow  $C_b^2$  (R)  
 HDE : Equivalent hydraulic diameter of this region (cm) (R)  
 AR : Cross sectional area of this region (Flow area) ( $\text{cm}^2$ ) (R)  
     \* CARD F74 - CARD F75 are required IDC MX (CARD F21) sets  
     \* CARD F1 - CARD F75 are required NMAX (CARD B2) sets

<CARD F76> Coolant channel configuration data (Free format) (If INFORM<>0 (CARD B1) and KEY(2)=5 (CARD B2), then this card is required.)

YCHI : Gap (thickness) of coolant channel for plate-type fuel (cm) (R)  
 . Equivalent hydraulic diameter for rod-type fuel (cm) (R)  
 XCHI : Width of coolant channel for plate-type fuel (cm) (R)  
 . Effective flow area for one fuel rod for rod-type fuel ( $\text{cm}^2$ ) (R)  
 MSFLW : Number of segments, include fuel plate region<sup>1)</sup> (In this case number of fuel plate region must be 1) (Number of CARD F77)  
 MSFUEL : Fuel plate region segment number (From top of segment) (I)

<CARD F77> Pressure loss calculation data (Free format) (If INFORM<>0 (CARD B1) and KEY(2)=5 (CARD B2), then this card is required )

ZETA : Resistance coefficient of this region entrance (R)  
 DH : Length of flow area (cm) (R)  
 ZLAM : Friction loss coefficient for laminar flow  $C_b^2$  (R)  
 HDE : Equivalent hydraulic diameter of this region (cm) (R)  
 AR : Cross sectional area of this region (Flow area) ( $\text{cm}^2$ ) (R)  
     \* CARD F76 - CARD F77 are required IDC MX (CARD F21) sets  
     \* CARD F1 - CARD F77 are required NMAX (CARD B2) sets.

<CARD G1> Control card ( Free format) (If INFORM<>0 (CARD B1), then this card is required )

KVELO : Index for primary coolant flow rate (I)  
     = 0 . Volumetric flow rate ( $\text{m}^3/\text{min}$ )  
     = 1 . Mass flow rate ( $\text{kg/s}$ )  
     = 2 . Average coolant velocity in the core (cm/s)  
 JUMAX : Number of non-heated flow segment of channel inlet side (I)  
 JLMAX : Number of non-heated flow segment of channel outlet side (I)  
     \* If KEY(2)<>5 (CARD B2), then JUMAX + JLMAX = MSFLW  
           (CARD F74)  
     \* If KEY(2)=5 (CARD B2), then JUMAX + JLMAX + 1 = MSFLW  
           (CARD F76)  
 IHTC : Index for heat transfer correlation (I)  
     = 1-3 : COOLOD code original heat transfer correlation. See Table 1.  
           (Single-phase heat transfer correlation is defined by CARD E1.)  
     = 1 : DNB heat flux is calculated by LABUNTSOV correlation  
     = 2 : DNB heat flux is calculated by MIRSHAK correlation  
     = 3 : DNB heat flux is calculated by BERNATH correlation  
     = 4 : "Heat Transfer Package"  
 KBFLG : Index for void fraction calculation in the natural circulation cooling mode (I)  
     = 0 : Void fraction is calculated in only nucleate boiling region (Zuber correlation)  
     > 0 : Void fraction is calculated in both nucleate boiling and subcooled boiling region. In subcooled boiling region, void fraction correlation is as follows.  
           = 1 : AHMAD correlation  
           = 2 : Zuber correlation

= 3 : If flow rate in the core G(kg/s) < GLIM (**CARD G5**), then AHMAD correlation

If flow rate in the core G(kg/s) >= GLIM (**CARD G5**), then Zuber correlation

\* If forced convection cooling mode, then KBFLG = 0

NCMAX Number of non-heated channel(Core bypass) (!)

\* If KEY(2)<>5 (**CARD B2**), then NCMax must be 0

NATIP Option for flow rate calculation in the natural circulation cooling mode (1)

= 0 : Hot channel factors are not used in the calculation of flow rate in the natural circulation cooling mode.

= 1 : Hot channel factors are used in the calculation of flow rate in the natural circulation cooling mode.

\* If KEY(2)<>5 (**CARD B2**), then NATIP must be 0

<**CARD G2**> Core bypass data (1) (Free format) (If INFORM<>0 (**CARD B1**) and KEY(2) =5 (**CARD B2**), then this card is required.)

MSFLOW Number of core bypass segments (?)

<**CARD G3**> Core bypass data (2) (Free format) (If INFORM<>0 (**CARD B1**) and KEY(2) =5 (**CARD B2**), then this card is required.)

ZETA Resistance coefficient of this region entrance (R)

DH Length of flow area (cm) (R)

ZLAM Friction loss coefficient for laminar flow  $C_b^{2/3}$  (R)

HDE Equivalent hydraulic diameter of this region (cm) (R)

AR Cross sectional area of this region (Flow area) (cm<sup>2</sup>) (R)

\* This card is required MSFLOW (**CARD G2**) sets

\* **CARD G2** and **CARD G3** are required NCMax (**CARD G1**) sets.

<**CARD G4**> Coolant channel configuration identity data (Free format) (If INFORM<>0 (**CARD B1**), then this card is required.)

JMSH Flag for channel configuration (!)

\* ((JMSH(NP, k), NP = 1, NPMX), K = 1, KMX)

\* If KEY(2) <> 5 (**CARD B2**), then this card is required MSFLW x NMAX (**CARD B2**) sets. (MSFLW (**CARD F74**)= JUMAX (**CARD G1**) + JLMAX (**CARD G1**))

\* If KEY(2) = 5 (**CARD B2**), then this card is required MSFLW x (NMAX (**CARD B2**) + NCMax (**CARD G1**)) sets.

(MSFLW (**CARD F76**) = JUMAX (**CARD G1**) + JLMAX (**CARD G1**) + 1)

<**CARD G5**> Void fraction calculation data (Free format) (If INFORM<>0 (**CARD B1**) and KEY(2) =5 (**CARD B2**), then this card is required )

CB Zuber constant (R)

\* You had better to use CB = 1.18 or 1.41.

GLIM Standard flow rate for void fraction calculation (kg/s) (R)

\* GLIM is used only in the case of KBFLG = 3 (**CARD G1**)

\* You had better to use GLIM = 500 - 1500 (kg/m<sup>2</sup>s).

<**CARD G6**> Debug control card (!)

IDBG(I), I = 1, 25

IDBG(I), I = 26, 50

IDBG : If you need debug the subroutine l, please input IDBG >= 5  
 See Table 4. (Normally : =0)

## &lt;CARD P1&gt; Plot control card (1)

WITHX : Length of X axial (Maximum 200 mm) (mm) (R)  
 WITHY : Length of Y axial (Maximum 230 mm) (mm) (R)  
 TMIN : Minimum value of temperature scale (°C) (R)  
 TMAX : Maximum value of temperature scale (°C) (R)  
 PMIN : Minimum value of pressure scale (kg/cm<sup>2</sup>abs.) (R)  
 PMAX : Maximum value of pressure scale (kg/cm<sup>2</sup>abs.) (R)  
 HMIN : Minimum value of heat flux scale (W/cm<sup>2</sup>) (R)  
 HMAX : Maximum value of heat flux scale (W/cm<sup>2</sup>) (R)

## &lt;CARD P2&gt; Plot control card (2) (A4)

NEWI : = "NEW" Plot on new page  
 = "OLD" Plot on same page  
 \* In the first figure NEWI must be "NEW".

## &lt;CARD P3&gt; Figure title card (A40)

TITLE : Title of figure

\* If NEWI="OLD", then this card is not required.

## &lt;CARD P4&gt; Plot control card (3) (I)

IDPLOT(1)-(7), NSMBL(1)-(7)

Plot items are listed as follows.

- (1) Coolant temperature
- (2) Clad surface temperature
- (3) Meat maximum temperature
- (4) Saturation temperature
- (5) ONB temperature
- (6) Pressure
- (7) Clad surface heat flux

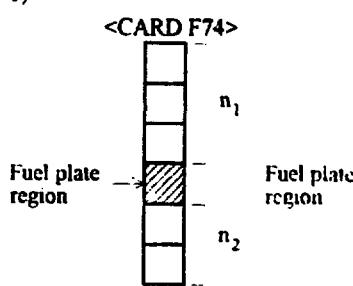
IDPLOT(I) : = 0 No plot  
 = 11-15 Solid line is used  
 = 21-25 Doted line is used

NSMBL(I) : = 0 No symbol  
 = 1 ○ is plotted on the line  
 = 2 △ is plotted on the line  
 = 3 + is plotted on the line  
 = 4 × is plotted on the line  
 = 11 \* is plotted on the line

Table 4 Debug flag for each subroutine

Subroutine name	IDBG No	Subroutine name	IDBG No
	1		26
	2	ONBTE	27
CALCTL	3	CLADTE	28
	4	BONDTE	29
	5	FUELTE	30
INITLZ	6	HEATBL	31
POWER	7	QHFPKG	32
TMPINL	8		33
	9	PRESS	34
DISPWZ	10	QDNB (=8)	35
QRATE	11		36
TMPCAL	12		37
	13		38
	14		39
	15	NATURE	40
VELOC, VELOC2	16	FLWGO	41
	17	DLTPD	42
	18	LOSTL	43
	19		44
NEWTON (=8)	20	GICAL	45
	21		46
	22	PBPH	47
	23	REN (=1)	48
COOLTE	24	UNITI	49
PRESDRP	25	UNITO	50

1)



$$MSFLW = n_1 + n_2$$

2)

$$F = \frac{C_b}{Re}$$

Friction loss coefficient

Reynolds number

Tube  $C_b = 64.0$ Square  $C_b = 56.9$ Rectangular  $C_b = 96.0$ 

(Channel of fuel element,

$$MSFLW = n_1 + n_2 + 1 \text{ (Fuel plate region)}$$

## 6. Concluding Remarks

In this report, information required for the COOLOD-N2 code users has been described. The COOLOD-N2 was developed based on the COOLOD-N code and provides a capability for the analysis of the steady-state thermal-hydraulics of research reactors. The COOLOD-N2 is applicable not only for research reactors in which plate-type fuel is adopted, but also for research reactors in which rod-type (pin-type) fuel is adopted. This work has been done as a part of the thermal-hydraulic analysis of the JRR-4 TRIGA fueled core. Thermal-hydraulic calculations for the JRR-4 TRIGA core were successfully conducted using COOLOD-N2 code.

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## References

- [1] M. Kaminaga, "COOLOD-N : A Computer Code, for the Analyses of Steady-State Thermal-Hydraulics in Plate-Type Research Reactors", JAERI-M 90-021, 1990.
- [2] S. Watanabe, "COOLOD : Thermal and Hydraulic Analysis Code for Research Reactors with Plate Type Fuel Elements (in Japanese)", JAERI-M 84-162, 1984.
- [3] M. Kaminaga, H. Ikawa, S. Watanabe, H. Ando and Y. Sudo, "Thermohydraulic Characteristics Analysis of Natural Convective Cooling Mode on the Steady State Condition of Upgraded JRR-3 Core, using COOLOD-N Code (in Japanese)", JAERI-M 87-055, 1987
- [4] M. Kaminaga, "Core Thermohydraulic Analysis of the Multi-Purpose Research Reactor RSG-GAS using COOLOD-N code", Internal Report, JAERI, 1990.
- [5] M.A. Lucatero, M. Kaminaga, "Thermal-Hydraulic Conceptual Design of the Multiple Purpose Research Reactor MEX-15", JAERI-M 94-006, 1994.
- [6] Y. Sudo, H. Ikawa and M. Kaminaga, "Development of Heat Transfer Package for Core Thermal-hydraulic Design and Analysis of Upgraded JRR-3", in Proceeding of the International Meeting of Reduced Enrichment for Research and Test Reactors, Petten, The Netherlands, October 14-16, 1985.
- [7] General Atomics, "10 MW TRIGA-LEU Fuel and Reactor Design Description", UZR-14 (Rev.), 1979.
- [8] F. W. Dittus and L. M. K. Boelter, Univ. Calif. Pubs. Eng., 2, 443, 1930.
- [9] J. G. Collier, "Convective Boiling and Condensation", McGraw-Hill Book Co., New York, 1972.
- [10] A. E. Bergles and W. H. Rohsenow, "The determination of forced-convection surface-boiling heat transfer", ASME, Ser.C, 86, 365-372, 1964.
- [11] J. C. Chen, "A correlation for boiling heat transfer to saturated fluids in convective flow", ASME paper No.63-HT-34.
- [12] Y. Sudo, K. Miyata, H. Ikawa, M. Kaminaga and M. Ohkawara, "Experimental Study of Differences in DNB heat Flux between Upflow and Downflow in Vertical Rectangular Channel", J. Nucl. Sci. Technol., Vol.22, No 8, 604-618, 1985.
- [13] K. Mishina, "Boiling burnout at low flow rate and low pressure conditions". Dissertation Thesis, Kyoto Univ., 1981.
- [14] R. H. Whittle and R. Forgan, "A correlation for the minima in the Pressure Drop versus Flow-rate Curves for Sub-cooled Water Flowing in Narrow Heated Channels", Nucl. Eng. Design, 6,

- 89-99, 1967.
- [15] The Japan Society of Mechanical Engineers. "Handbook of Mechanical Engineering (6th Edition) (in Japanese)", Section 8, 13-15.
  - [16] H. Uchida. "Wet air and Cooling tower (in Japanese)". Shoukado, 1963.
  - [17] H. Obana. "Handbook of Heat exchanger Design (in Japanese)". Kougaku-Tosho, 1977
  - [18] Saha, Zuber. "Point of net vapor generation and vapor void fraction in subcooled boiling". Proc 5th Int. Heat Trans. Conf., Tokyo, 4, 175-179, 1974.
  - [19] S. Y. Ahmad. "Axial distribution of bulk temperature and void fraction in a heated channel with inlet subcooling". Trans. ASME, Ser.C, 92-4, 595-609, 1970.
  - [20] N. Zuber, F. W. Stanb and G. Bijwaard. "Vapor void fraction in subcooled boiling system". 3rd Int. Heat Transfer Conf., 5, 24-38, 1966
  - [21] P. G. Kroeger and N. Zuber. "An analysis of the effects of various parameters on the average void fractions in subcooled boiling". Int. J. Heat Mass Transfer, 11, 211-233, 1968
  - [22] J. E. Houghtaling, Alain Sola and A. H. Spano. "Transient Temperature Distribution in the SPERT1D-12/25 Fuel Plates during Short-Period Power Excursions". IDO-16884, 1964.
  - [23] S. Nazare, G. Ondracek and F. Thummel. "Investigation on UAl<sub>x</sub>-Al Dispersion Fuels for High-Flux Reactors". J. Nucl. Materials, 56, 251-259, 1975.
  - [24] Y. S. Touloukian et al., "Thermophysical Properties of Matter, Vol.1, Thermal Conductivity". 1970
  - [25] Y. S. Touloukian et al., "Thermophysical Properties of Matter, Vol.3, Thermal Conductivity". 288-289, 1970.
  - [26] The Japan Society of Mechanical Engineers, "Dennetsu-kougaku shuryo (3rd Edition) (in Japanese)". 1976.
  - [27] J. N. Elliott, "Tables of the Thermodynamic Properties of Heavy Water". AECL-1673, 1963
  - [28] M. Kaminaga "Steady-State GA Benchmark calculation for JRR-4 TRIGA-16 Fueled Core (Steady-State Thermal-Hydraulic Analysis). (in Japanese)". private communication, 1993
  - [29] General Atomics, private communication
  - [30] M. Kaminaga "Steady-State Thermal-Hydraulic Analysis for JRR-4 TRIGA-25 Fueled Core (in Japanese)", private communication, 1993.
  - [31] M. Kaminaga "Steady-State Thermal-Hydraulic Analysis for JRR-4 TRIGA-16 Fueled Core (in Japanese)", private communication, 1993.
  - [32] General Atomics, private communication.

## Appendix A Sample calculation results

### A. 1 JRR-4 TRIGA-16 core GA (General Atomics) benchmark calculation, input data description

Thermal hydraulic analysis for the JRR-4 TRIGA-16 core<sup>[28]</sup> was carried out to verify the fuel rod temperature calculation model and Lund DNB heat flux correlation by comparing analysis results calculated by General Atomics using TIGER code<sup>[29]</sup>. This analysis has been done as a part of the thermal-hydraulic analysis of the JRR-4 TRIGA fueled core<sup>[28],[30],[31]</sup>.

Following input data were used for JRR-4 TRIGA-16 core thermal hydraulic analysis. Almost all of following input data used in the analysis were as same as those used by GA calculation<sup>[29]</sup>.

- a. Primary coolant flow rate through TRIGA fuel element is 81.12% ( $5.68 \text{ m}^3/\text{min}$ ) of the total primary coolant flow rate of  $7 \text{ m}^3/\text{min}$ .
- b. Hot pin factor (radial peaking factor  $\times$  local peaking factor) is 1.7.
- c. Engineering hot channel factors were not considered.
- d. Equivalent hydraulic diameter for TRIGA fuel element is calculated for a channel which is surrounded by 4 fuel rods.
- e. Form loss coefficient at the fuel element inlet is taken from "10 MW TRIGA-LEU Fuel and Reactor Design Description", General Atomics<sup>[7]</sup>.
- f. An axial power distribution is calculated by the following correlation which was used for the thermal hydraulic design of 14 MW TRIGA reactor, Romania<sup>[32]</sup>. Figure A.1 shows the axial power distribution used in the analysis.

$$APF = 1.35(1 + 1.275e^{-39.96(1-\xi)})\cos(1.325\xi) \quad (\text{A-1})$$

$$\xi = |1 - 2Z/L|$$

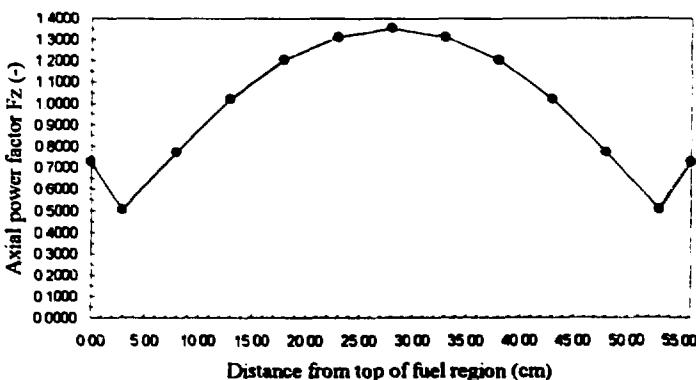


Figure A.1 Axial power distribution for JRR-4 TRIGA-16 core

- g. Thermal conductivities for fuel pellet and cladding, gap conductance are shown in Table A.1.

Table A.1 Thermal conductivities and gap heat transfer coefficient used in the analysis

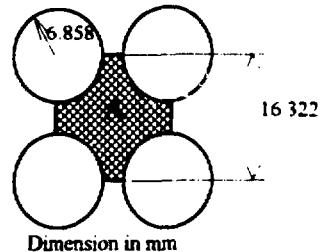
Fuel pellet thermal conductivity (W/m K)	0.196
Cladding thermal conductivity (W/m K)	0.162
Gap conductance (W/m <sup>2</sup> K)	0.804

h. Fuel pellet diameter	: 1.2903 cm
i. Fuel rod diameter	: 1.3716 cm
j. Equivalent hydraulic diameter	: 0.0406 cm
k. Sub-channel flow area	: 1.1898 cm <sup>2</sup>

(flow area per one fuel rod) Figure A.2 shows cross-sectional area of the channel.

#### 1. Pressure drop calculation model for TRIGA fuel element.

Figure A.3 shows a pressure drop calculation model for JRR-4 TRIGA-16 fuel element.



**Figure A.2 Cross sectional area of the sub-channel**

Form loss coefficient at the fuel element inlet

$$K = 3.2$$

Flow area and an equivalent hydraulic diameter of the sub-channel

$$A = 163322^2 - 0.6858^2 \pi$$

$$= 1.189849 \text{ cm}^2$$

$$De = \frac{4 \times 1.189849}{2 \times 0.6858 \pi} \\ = 1.1045219 \text{ cm}$$

Form loss coefficient of spacers (2 locations)

$$K = 0.4$$

Flow area of fueled region

$$A = 1.189849 \times 16 = 19.03758 \text{ cm}^2$$

Grid plate

$$K = 0.5$$

$$A = \frac{1.27 \pi}{4} \times 16 = 20.2683 \text{ cm}^2$$

$$De = 1.27 \text{ cm}$$

Flow channel under the grid plate

$$K = 1.0$$

$$A = \frac{6.808 \pi}{4} = 36.3809 \text{ cm}^2$$

$$De = 6.808 \text{ cm}$$

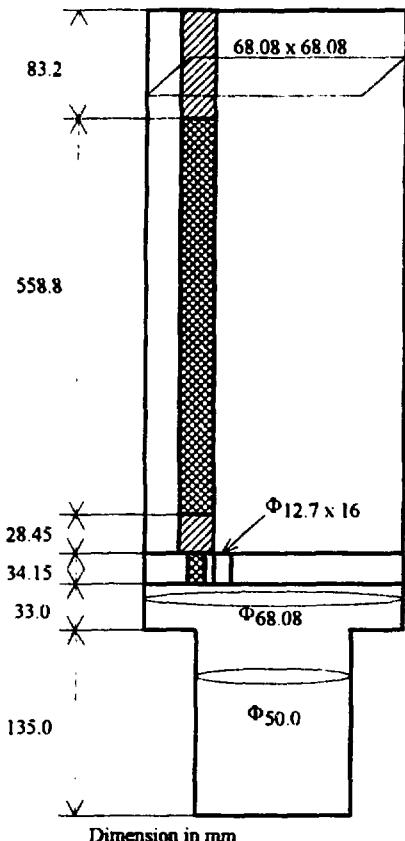
Nozzle section

$$Kin = 0.5$$

$$A = \frac{5.0 \pi}{4} = 19.6350 \text{ cm}^2$$

$$De = 5.00 \text{ cm}$$

$$Kout = 1.0$$



**Figure A.3 Pressure drop calculation model for JRR-4 TRIGA fuel element**

#### A. 2 JRR-4 TRIGA-16 core GA benchmark calculation, calculation results

Table A.2 shows calculation results of JRR-4 TRIGA-16 core including major input data. In the table calculation results calculated by Tiger code are also shown for a comparison. Figure A.4 shows axial temperature, pressure and heat flux distributions of the hot channel.

**Table A.2 Comparison of calculation results between COOLOD-N2 and Tiger code**

Thermal power = 3.5 MW		Calculation results, Tiger code	
		Hot channel	Hot channel
Fuel	Number of fuel elements	20	20
	Number of rod / element	16	16
	Pin diameter (m)	0.013716	0.013716
	Flow area / element ( $m^2$ )	$1.904 \times 10^{-3}$	$1.904 \times 10^{-3}$
Peaking factor	Fr (Radial peaking factor) (Hot pin factor)	1.7	1.7
	Fz (Axial peaking factor)	1.34	1.35
	Total	2.278	2.295
Coolant	Core inlet pressure ( $kg/cm^2$ )	2.074	2.074
	Core flow rate ( $m^3/min$ )	7.00	7.00
	Effective flow rate for fuel cooling ( $m^2/min$ )	5.68	5.68
	Average coolant velocity (m/s)	2.50	2.50
	Inlet temperature ( $^{\circ}C$ )	37.0	37.0
	Outlet temperature ( $^{\circ}C$ )	51.9	52.1
Results	Maximum fuel (cladding) surface temperature ( $^{\circ}C$ )	129.7	127.3
	Maximum fuel (pellet) temperature ( $^{\circ}C$ )	476.3	474.2
	Maximum fuel (cladding) surface heat flux ( $W/m^2$ )	$1.041 \times 10^6$	$1.042 \times 10^6$
	DNB heat flux ( $W/m^2$ )	$2.507 \times 10^6$	$2.657 \times 10^6$
	DNBR	2.41	2.55

From the calculation results, calculation results calculated by COOLOD-N2 code show good agreement with those calculated by Tiger code while there is a little difference of fuel temperatures between the two codes. The calculation results calculated by COOLOD-N2 code (including DNB heat flux) are a little bit higher than those calculated by Tiger code. Heat transfer calculation model of COOLOD-N2 code is considered to be the same with Tiger code, but at this moment, we do not have enough information of Tiger code and its input data. Therefore, it is difficult to investigate the reason of these difference furthermore.

The calculation model for rod type fuel of COOLOD-N2 code was successfully verified by comparing the calculation results calculated by Tiger code.

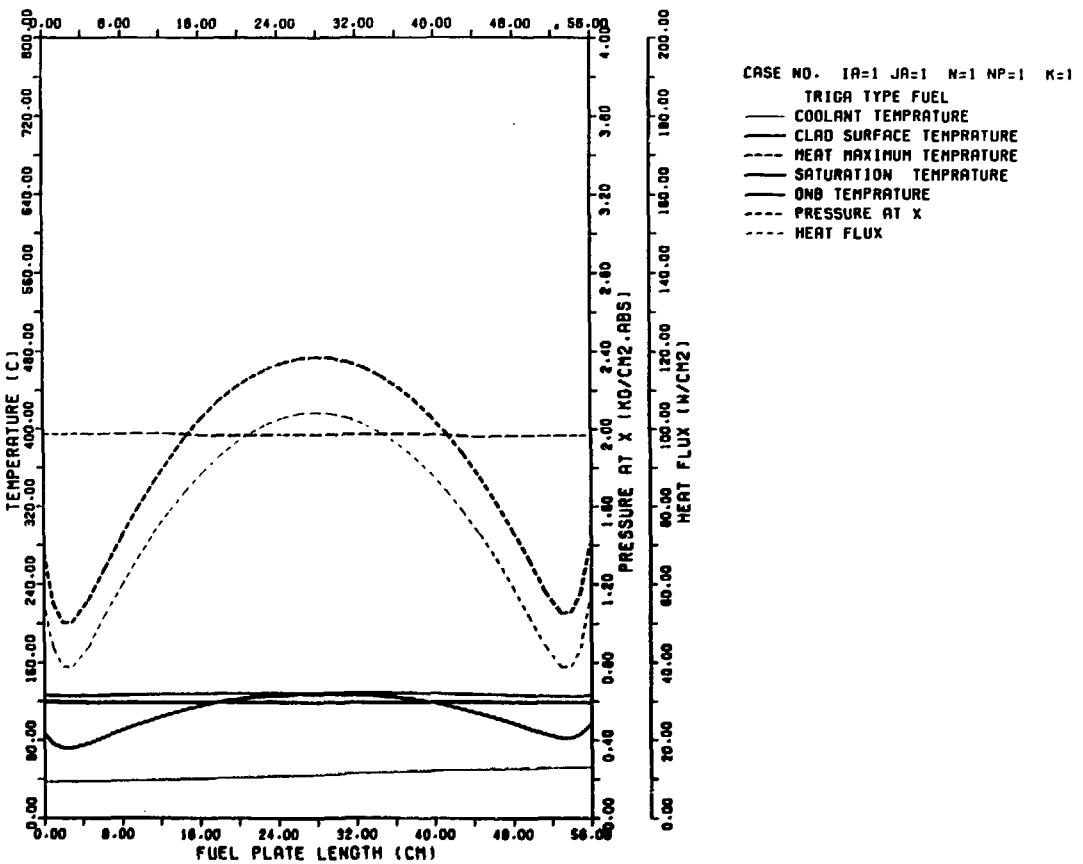


Figure A.4 Axial temperature, heat flux and pressure distribution in the hot channel of JRR-4 TRIGA-16 core

**A. 3 Input data for COOLOD-N2, JRR-4 TRIGA-16 core thermal-hydraulic analysis**

```

INPUT DATA CARD IMAGE
COOLOD-N THERMAL HYDRAULIC CALC. (JRR-4 TRIGA-16 FUEL CORE) 08/10/83 HECK
C CORE FLOW RATE 7.0 MC/MIN GA007 GA BENCHMARK CALCULATION
C <CARD B1> INFORM
    1
C <CARD B2> IIMAX IDMAX JMAX NMAX NPILOT KEY(1) KEY(2) KEY(3) IDMAX
    1      5     13    1    1     1    1     1    0     3
C <CARD C > QCR(MIN) PFLW(KG/SEC) TIN(DEG) DT(DEG) JMAX
    3.5     115.887   37.0     0.0     1
C <CARD E1> H1   H2   H3   A   B   C   D   ITMC
    0.023   0.0   0.0   0.800   0.400   0.0   0.0   1
C <CARD E2> FRATE VIM VOUT PREBIN RAMF
    0.011159   0.0   0.0   2.0760565   0.0
C <CARD F1> FUEL ELEMENT TITLE
TRIGA TYPE FUEL 16PIN
C <CARD F2> NMMAX NTFLW MA UDENST POROTY IDMAX IDCMX KAREA FRATION
    1     20.0   2   19.07   0.030   1   1 19.03758 1.000
C <CARD F3> DE(CM) D(CM) PITCH/D (-)
    1.1045218   1.3716   1.190741
C <CARD F221> ER-H W/CM.C
    3
        0.00     0.1963317
        378.00    0.1963317
        2000.00   0.1963317
C <CARD F222> GAP CON W/CM2.C DUMAY (GAP = 0.00230 CM)
C 1.8467E-03 (W/CM.C) / 0.00230 (CM) = 0.80353 W/CM2.C NOT PIN
    2
        0.0     0.00353
        2000.0   0.00353
C <CARD F223> CLAD INCOLLOY W/CM.C
    2
        0.0     0.16194
        2000.0   0.16194
C <CARD F3 > IR PCOOL PFLWK PFLWU PCFLD PFRDN PWEAR
    1.70000 1.000 1.000 1.000 1.000 1.000 1.000
C <CARD F4 > IR DDE INT (FROM CALC. RESULTS OF CREATRON)
    0.72320   2.84000   0.0
    0.50300   5.00000   0.0
    0.77100   5.00000   0.0
    1.02000   5.00000   0.4
    1.20000   5.00000   0.0
    1.31000   5.00000   0.0
    1.35000   5.00000   0.0
    1.31000   5.00000   0.0
    1.20000   5.00000   0.4
    1.02000   5.00000   0.0
    0.77100   5.00000   0.0
    0.50300   2.84000   0.0
    0.72320   0.00000   0.0
C <CARD F5 > NMMAX GROUPS
C <CARD F51> FUEL PLATE TITLE
TRIGA TYPE FUEL
C <CARD F52> NMFRD PFLW IRPL IRPL IRPL IRPL OUT
    16.0   1.000   1   1   1   1
C <CARD F53> ICNL(1) WHET(1) FRATEC(1) ICNL(2) WHET(2) FRATEC(2)
    1     1.0   1.000
C <CARD F6 > XA XB XC YA MA HD NC
    0.64516  0.00000  0.04064  0.0   8.320  55.000  2.645
C <CARD F7 > XCHI(DE) YCHI(AREA) NMFLW MDFUEL
    1.1045219 1.189849   5   0
C <CARD F74> KERA DR ELAN HDE AR
    3.20   0.000  0.0   0.932905 22.4942
    0.50   3.415 64.0   1.27000 20.2583
    1.00   3.300 64.0   6.80500 36.3809
    0.50   13.500 64.0   5.00000 19.6350
    1.00   0.000  0.0   5.00000 19.6350
C IF INFORM.NE.0, SET <CARD G> AS FOLLOWS
C <CARD G1> = ONE SHOOT
C <CARD G2><CARD G3> = 'NMMAX' GROUPS
C <CARD G4> = 'NMMAX-NMAX' SHOOT
C <CARD G1> IVELO JMAX JLMAX INTC KMFZL NMMAX NMAX
C     1     2     4     4     0     0     0
C     1     1     4     4     0     0     0
C <CARD G4 > ((JMAX(NF, NF, JJ, JLO, NF-1, NMMAX), JLO-1, NMMAX)
    1
    2
    3
    4
    5
C <CARD G4 > IDBG
    0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
    0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
C <CARD P1> WATMX WATRY THINX NMAX PFLWK NMFLW NMFLX
    140.0 200.0   0.0 800.0   0.0   4.0   0.0 200.0
C <CARD P2> NEWI
NEW
C <CARD P3> TITLE(M40)
JRR-4 TRIGA-16 CASE:GA BENCHMARK
C <CARD P4> ID1 ID2 ID3 ID4 ID5 ID6 ID7 MS1 MS2 MS3 MS4 MS5 MS6 MS7
    11   13   23   14   12   22   21   0   0   0   0   0   0   0

```

## A. 4 Output lists of COOLOD-N2, JRR-4 TRIGA-16 core thermal-hydraulic analysis

COOLOD-N THERMAL HYDRAULIC CALCULATION

CALCULATION DATE 94-01-11 PAGE 1

```
*****
*** COOLOD-N THERMAL HYDRAULIC CALC. (JRR-4 TRIGA-16 FUEL CORE) 08/10/93 ***
*** CALCULATION DATE 94-01-11 ***
*** INITIAL INPUT DATA ***
*****
```

INPUT CARD B1	INFORM										
INPUT CARD B2	I	IDMAX	JMAX	NMAX	NPLOT	KEY(1)	KEY(2)	KEY(3)	IDMAX		
NZ ***	5 6 9										
INPUT CARD C	QRR	PTLOW	TIN	D2	JMAX						
CASE 1	3.500	115.887	37.000	0.000	1						
INPUT CARD E1	M1	M2	M3	A	B	C	D	ITMC			
	0.023	0.000	0.000	0.800	0.400	0.000	0.000	1			
INPUT CARD E2	FTRATE	VIN	VOUT	FREZIN	ZMAX						
	0.8112	0.0000	0.0000	2.0741	0.0000						
INPUT CARD F1	TRIGA TYPE FUEL 16PIN										
INPUT CARD F21	NPAC	NTFUEL	NR	UDENST	POROTY	IDPK	IDSK	EAREA	FRATE		
	1	20.0	2	10.070	0.030	1	1	19.038	1.000		
INPUT CARD F3	SR	FCOOL	FPFLM	FPFLX	PCFLAD	PROND	PRNDL				
	1.700	1.000	1.000	1.000	1.000	1.000	1.000				
INPUT CARD F4	J	FE	DDE	SET							
	1	0.7232	2.940	0.000							
	2	0.8030	5.000	0.000							
	3	0.7710	5.000	0.000							
	4	1.0200	5.000	0.000							
	5	1.2000	5.000	0.000							
	6	1.3100	5.000	0.000							
	7	1.3800	5.000	0.000							
	8	1.3100	5.000	0.000							
	9	1.2000	5.000	0.000							
	10	1.0200	5.000	0.000							
	11	0.7710	5.000	0.000							
	12	0.8030	2.940	0.000							
	13	0.7232	0.000	0.000							
INPUT CARD F51	INPUT CARD F52								INPUT CARD F53		
PLATE NAME	NPAC	FLOC1	IDPL	IRK	IPLOT	IZOUT	K	ICNL	NPATR		
NP= 1	TRIGA TYPE FUEL	16.0	1.000	1	1	1	1	1	1.0 1.0000		
INPUT CARD F6	IDP	XAI	XBI	XCI	XAI	XBI	NCI	NCI			
	1	0.665	0.000	0.041	0.000	0.120	55.960	2.645			
INPUT CARD F74	IDC	XCII	YCII	MSFLN			INPUT CARD F75				
	1	1.105	1.190	5			IS	SETA	DR AREA		
							1	3.200	0.000 0.000 0.933 22.494		
							2	0.500	3.415 64.000 1.270 20.260		
							3	1.000	3.300 64.000 6.806 36.381		
							4	0.500	13.500 64.000 5.000 19.635		
							5	1.000	0.000 0.000 5.000 19.635		
INPUT CARD <GL>	KVEL0	JUMAX	JMAX	IDTC	KFLG	NCMAX	NPATP				
	1	1	4	4	0	0	0				
INPUT DATA FORMAT --> COOLOD ORIGINAL											
VELOCITY(=0) MASS FLOW RATE(=1)=> 1											
UPPER PLATE MESH-- 1 FUEL PLATE MESH-- 13 LOWER PLATE MESH-- 4											
JMAXH MNH 1 MFLMN 5 NMAXH 1 NCNAH 1 KEY(2)= 1											
1 1 2 2 3 3 4 4 5 5											
MNH 1 JMAXH 5 JMAXH 1 2 3 4 5											
IDBG											
1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25											
0 0											
26 .7 28 29 30 31 32 33 34 35 36 37 38 39 40 41 42 43 44 45 46 47 48 49 50											
0 0											

COOLOD-N THERMAL HYDRAULIC CALCULATION CASE = (IA- 1 JA- 1)

CALCULATION DATE 94-01-11 PAGE 2

```
*****
** COOLOD-N THERMAL HYDRAULIC CALC. (JRR-4 TRIGA-16 FUEL CORE) 08/10/93 **
** RESULTS OF CALCULATION AND USED VALUES **
*****
*** PRIMARY COOLANT ***
REACTOR INLET TEMPERATURE = 37.00 C
REACTOR OUTLET TEMPERATURE = 44.23 C
PRIMARY TEMPERATURE DIFFERENCE = 7.23 C
PRIMARY COOLANT FLOW RATE = 115.88 KG/S
*** REACTOR CORE ***
REACTOR THERMAL POWER = 3.50 MW
AREA OF TOTAL FUEL CHANNELS = 380.75 CM2
NUMBER OF FUEL ELEMENTS = 20.0 ELEMENTS
TRIGA TYPE FUEL 1SPIN = 20.0 (ELEMENTS)
AVERAGE HEAT GENERATION = 149.68 (W/CM2)
AVERAGE MASS FLUX = 2468.872 (KG/M2 SEC)
```

COOLANT TEMP. --(SEPARATED MODEL) KITE = 0

Average heat flux for plate type fuel  
= AVERAGE HEAT GENERATION x XA  
(XA : Half thickness of fuel meat (CARD F6))

Average heat flux for rod type fuel  
= AVERAGE HEAT GENERATION x  $\frac{XA}{2} \times \frac{2\pi XA}{2\pi(XA + XB + XC)}$   
(XA : Radius of fuel pellet (CARD F6))  
(XB : Gap thickness between pellet and cladding (CARD F6))  
(XC : Cladding thickness (CARD F6))

COOLED-N THERMAL HYDRAULIC CALCULATION CASE = (IA- 1 JA- 1)

CALCULATION DATE 94-01-11 PAGE 3

\*\*\*  
\*\* TRIGA TYPE FUEL 16PIN  
\*\*\*-----  
AVERAGE CHANNEL TEMPERATURE DISTRIBUTION  
-----

FLOW CHANNEL AREA = 19.06 CM2

NUMBER OF FUEL PLATES

TRIGA TYPE FUEL = 16.0

TEMPERATURE DISTRIBUTION					
J	COOLANT (DEG.C)	CLADDING SURFACE (DEG.C)	CLADDING INNER (DEG.C)	FUEL MEAT OUTER (DEG.C)	FUEL MEAT MAXIMUM (DEG.C)
1	37.00	65.94	74.46	117.90	175.25
2	37.29	57.36	63.29	93.50	133.39
3	37.80	68.42	77.50	123.01	184.06
4	38.51	78.76	80.77	152.04	232.93
5	39.39	86.36	100.49	172.57	267.73
6	40.38	91.21	106.63	165.32	289.20
7	41.45	93.37	109.27	190.36	297.41
8	42.51	92.46	107.88	186.57	290.45
9	43.51	88.08	103.01	175.09	270.26
10	44.40	82.68	94.69	185.95	236.64
11	45.11	73.87	82.95	129.26	190.40
12	45.62	64.30	70.22	100.63	140.32
13	46.91	72.70	81.21	124.65	182.00

\*\* HOT CHANNEL FACTORS (EXCEPT F1) \*\*  
F(COOLANT) = 1.000 F(FILM) = 1.000 F(CLAD) = 1.000 F(BOND) = 1.000 F(MEAT) = 1.000

HEAT TRANSFER CONDITION					
J	F1	TRANSFER COEFFICIENT (W/CM2.C)	HEAT FLUX IN PLATE SURFACE (WC/CM2.MU)	KAA (CM8)	HEAT GENERATION (W/CM2)
1	0.723	1.1350	32.851	0.38241E+06	0.000
2	0.503	1.1380	22.048	0.19642E+06	0.000
3	0.771	1.1434	35.022	0.30109E+06	0.000
4	1.020	1.1510	46.332	0.39831E+06	0.000
5	1.200	1.1607	54.509	0.46865E+06	0.000
6	1.310	1.1711	59.508	0.51254E+06	0.000
7	1.350	1.1811	61.322	0.52718E+06	0.000
8	1.310	1.1915	59.505	0.51158E+06	0.000
9	1.200	1.2014	54.509	0.46865E+06	0.000
10	1.020	1.2105	46.332	0.39831E+06	0.000
11	0.771	1.2179	35.022	0.30109E+06	0.000
12	0.503	1.2233	22.048	0.19642E+06	0.000
13	0.723	1.2263	32.851	0.38241E+06	0.000

\*\*\*\* ITERATION COUNT = 5\*\*\*\* CONVERGED.  
 \*\*\*\* ITERATION COUNT = 5\*\*\*\* CONVERGED.

H E A T F L U X = Average heat flux x FZ  
(FZ : Axial peaking factor (CARD F4))

F(COOLANT) = FR x FLOCL x FCOOL (CARD F3)

F(FILM) = FR x FLOCL x FFILM (CARD F3)

F(CLAD) = FR x FLOCL x FCLAD (CARD F3)

F(BOND) = FR x FLOCL x FBOND (CARD F3)

F(MEAT) = FR x FLOCL x FMEAT (CARD F3)

COOLOD-N THERMAL HYDRAULIC CALCULATION CASE = (IA- 1 JA- 1)

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## TRIGA TYPE FUEL (TRIGA TYPE FUEL 16PIN)

CHANNEL DIMENSION = 1.105 \* 1.180 (CM)  
CHANNEL VELOCITY = 249.30 (CM/SEC)

J	COOLANT (DEG.C)	TEMPERATURE DISTRIBUTION		FUEL MEAT MAXIMUM (DEG.C)
		SURFACE (DEG.C)	CLADDING (DEG.C)	
1	37.00	86.20	100.68	174.53
2	37.49	71.56	81.62	132.99
3	38.35	90.15	105.59	164.32
4	39.57	107.32	127.73	231.89
5	41.07	119.74	143.76	266.29
6	42.77	125.49	151.71	285.46
7	44.57	127.33	154.35	292.20
8	46.37	126.19	152.41	286.18
9	48.07	121.84	148.66	268.40
10	49.58	111.71	132.13	236.29
11	50.78	97.33	112.76	191.49
12	51.65	81.84	91.91	163.27
13	52.14	56.40	109.87	163.72

\*\* NOT CHANNEL FACTORS (EXCEPT FZ) \*\*

F(COOLANT)= 1.700 F(FILM)= 1.700 F(CLAD)= 1.700 F(BOND)= 1.700 F(MEAR)= 1.700

J	FZ	HEAT TRANSFER CONDITION			HEAT GENERATION (W/CM2)
		TRANSFER COEFFICIENT (W/CM2.C)	HEAT FLUX IN PLATE SURFACE (W/CM2)	XAA (%)	
1	0.723	1.1381	55.646	0.46010E+06	0.000
2	0.803	1.1402	58.642	0.33395E+06	0.000
3	0.771	1.1493	59.537	0.51182E+06	0.000
4	1.020	1.1626	78.768	0.67713E+06	0.000
5	1.200	1.1780	92.665	0.79661E+06	0.000
6	1.310	1.2229	101.180	0.86964E+06	0.000
7	1.350	1.2397	104.268	0.89202E+06	0.000
8	1.310	1.2678	101.189	0.86964E+06	0.000
9	1.200	1.2561	82.665	0.79661E+06	0.000
10	1.020	1.2676	78.768	0.67713E+06	0.000
11	0.771	1.2783	59.537	0.51182E+06	0.000
12	0.503	1.2868	34.842	0.33395E+06	0.000
13	0.723	1.2910	55.846	0.46010E+06	0.000

H E A T F L U X = Average heat flux x FR x FFILM x FZ

(FR : Radial peaking factor (CARD F4))

(FFILM : Engineering factor for film temperature rise

(CARD F4))

(FZ : Axial peaking factor (CARD F4))

COOLOD-N THERMAL HYDRAULIC CALCULATION CASE = (IA-1 JA-1)

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TRIGA TYPE FUEL ( PRESSURE , DNB &amp; DNB CONDITION )

	PRESSURE AT Z (KG/CMA)	PRESSURE LOSS (KG/CMA)	TOTAL LOSS (KG/CMA)	COOLANT VELOCITY (CM/SEC)	TEAT (C)	ZONE (C)	TCLAD (C)	DTONE (C)	CLAD DOME DOME ID	HEAT FLUX (W/CM <sup>2</sup> )	DNB
INLET	PLENUM	2.074	0.00								
STRETCH(1)	INLET	1.980	0.07176	0.07176	210.35						
STRETCH(1)	OUT	1.980	0.00000	0.07176	210.35						
PLATE	ENTRANCE	1.969	0.00241	0.07417	240.55						
FUEL PLATE ZONE 1		1.972	0.00522	0.07939	240.55	119.16	126.49	86.20	40.28	55.85	* 0.00
FUEL PLATE ZONE 2		1.973	0.00184	0.08122	240.59	119.16	125.35	71.96	53.80	38.84	* 0.00
FUEL PLATE ZONE 3		1.975	0.00313	0.08634	240.66	119.21	126.73	90.15	36.60	59.54	* 0.00
FUEL PLATE ZONE 4		1.976	0.00312	0.08744	240.77	118.24	127.84	107.32	20.52	78.77	* 0.00
FUEL PLATE ZONE 5		1.986	0.00311	0.09052	240.92	118.07	128.37	119.74	8.64	92.66	0.38
FUEL PLATE ZONE 6		1.988	0.00311	0.09355	240.10	118.10	128.68	125.49	3.19	101.16	41.73
FUEL PLATE ZONE 7		1.970	0.00310	0.09663	240.39	119.13	120.79	127.33	1.45	104.25	70.88
FUEL PLATE ZONE 8		1.971	0.00309	0.09965	240.48	118.16	120.71	126.19	2.53	101.16	51.12
FUEL PLATE ZONE 9		1.973	0.00308	0.10268	240.56	118.19	120.66	121.86	6.62	82.66	6.45
FUEL PLATE ZONE 10		1.983	0.00308	0.10570	240.62	118.02	127.65	111.71	15.93	78.77	* 0.00
FUEL PLATE ZONE 11		1.985	0.00307	0.10871	240.96	118.05	128.61	97.33	29.28	59.84	* 0.00
FUEL PLATE ZONE 12		1.987	0.00307	0.11174	250.07	118.08	128.26	81.86	63.43	38.84	* 0.00
FUEL PLATE ZONE 13		1.988	0.00180	0.11351	250.13	118.10	126.43	95.40	31.03	85.85	* 0.00
'WORST CONDITION'		1.963	0.00176	0.11826	280.13					106.25	* 0.00
PLATE	EXIT	1.969	0.00176	0.11826	280.13						
STRETCH(2)	INLET	1.972	0.00012	0.11837	234.94						
STRETCH(2)	OUT	1.976	0.00183	0.11699	234.94						
STRETCH(3)	INLET	1.979	0.01380	0.13080	130.89						
STRETCH(3)	OUT	1.983	0.00007	0.13087	130.89						
STRETCH(4)	INLET	1.932	0.02862	0.16049	242.92						
STRETCH(4)	OUT	1.944	0.00122	0.16171	242.92						
STRETCH(5)	INLET	1.929	0.01481	0.17682	242.92						
STRETCH(5)	OUT	1.929	0.00000	0.17682	242.92						
OUTLET	PLENUM	1.929	0.02962	0.20612	0.00						

----- DNBID=1 Q1=0.005\*G\*\*0.611 ---- DNBID=2 Q2=(A/AN) (DHE/10FC)\*G ---- DNBID=3 Q3=0.7(A/AN) R2(W/R) / R2(1-(R2, RL)\*\*0.25, ---

---- TCLAD &lt; TEAT

HEAT FLUX OF CLAD --- Q\*FR\*FHFLX

( KANGAN - MIKURADSE EQUATION WAS USED FOR WALL LOSS CALCULATION )

H E A T   F L U X ( CLJD ) = Average heat flux x FR x FFILM x FI OCL x FZ x ( FHFLX/FFILM )  
 = Average heat flux x FR x FHFLX x FLOCL x FZ

COOLED-N THERMAL HYDRAULIC CALCULATION CASE = (IA-1 JA-1)

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## TRIGA TYPE FUEL ( PRESSURE , QNB &amp; DNB CONDITION )

		PRESSURE AT E (KG/CM2)	PRESSURE LOSS (KG/CM2)	TOTAL LOSS (KG/SEC)	COOLANT VELOCITY (CM/SEC)	TEAT (C)	TONG (C)	TCLAD (C)	DTONG (C)	HEAT CLAD QNB DNB DNBR	FLUX (W/CM2) QNB QUND DNBR
INLET	PLENUM	2.074			0.00						
STRETCH(1)	INLET	1.980	0.07176	0.07176	210.35						
STRETCH(1)	OUT	1.980	0.00000	0.07176	210.35						
PLATE	ENTRANCE	1.969	0.00241	0.07417	248.55						
FUEL PLATE ZONE 1		1.972	0.00522	0.07939	248.55	119.16	126.49	86.20	40.28	55.85	* 0.00 208.37 5.16
FUEL PLATE ZONE 2		1.973	0.00184	0.08122	248.59	119.18	125.35	71.56	53.80	38.84	* 0.00 266.93 7.39
FUEL PLATE ZONE 3		1.975	0.00313	0.08834	248.66	119.21	126.75	90.15	36.60	59.56	* 0.00 284.38 4.78
FUEL PLATE ZONE 4		1.976	0.00312	0.08744	248.77	119.24	127.84	107.32	20.52	78.77	* 0.00 280.79 3.56
FUEL PLATE ZONE 5		1.966	0.00311	0.09052	248.82	119.07	128.37	119.74	8.64	92.66	* 0.34 276.06 2.96
FUEL PLATE ZONE 6		1.968	0.00311	0.09359	249.10	119.10	128.68	125.49	3.19	101.16	* 1.73 271.04 2.58
FUEL PLATE ZONE 7		1.970	0.00310	0.09663	249.29	119.13	128.78	127.33	1.45	104.25	* 0.00 265.72 2.55
FUEL PLATE ZONE 8		1.971	0.00369	0.08966	249.40	119.16	128.73	126.19	2.53	101.16	* 1.12 260.39 2.57
FUEL PLATE ZONE 9		1.973	0.00308	0.10266	249.65	119.19	128.46	121.84	6.62	82.66	* 6.45 255.35 2.76
FUEL PLATE ZONE 10		1.963	0.0308	0.10570	249.82	119.02	127.65	111.71	15.93	78.77	* 0.00 250.57 3.18
FUEL PLATE ZONE 11		1.965	0.00301	0.10871	249.96	119.05	126.61	97.33	29.20	58.54	* 0.00 246.94 6.15
FUEL PLATE ZONE 12		1.967	0.00307	0.11174	250.07	119.08	125.26	81.84	43.43	38.64	* 0.00 244.44 6.28
FUEL PLATE ZONE 13		1.968	0.00180	0.11381	250.13	119.10	126.43	98.40	31.03	55.85	* 0.00 243.00 4.35
"WORST CONDITION"		1.963			119.02	176.38				104.28	* 0.00 242.87 2.33
PLATE	EXIT	1.968	0.00174	0.11826	280.13						
STRETCH(2)	INLET	1.972	0.00712	0.11537	234.94						
STRETCH(2)	OUT	1.974	0.00153	0.11680	234.94						
STRETCH(3)	INLET	1.979	0.01380	0.13080	130.09						
STRETCH(3)	OUT	1.983	0.00007	0.13087	130.09						
STRETCH(4)	INLET	1.932	0.02962	0.16040	242.52						
STRETCH(4)	OUT	1.944	0.00122	0.16171	242.52						
STRETCH(5)	INLET	1.929	0.01481	0.17652	242.52						
STRETCH(5)	OUT	1.977	0.00000	0.17692	242.52						
OUTLET	PLENUM	1.779	0.01962	0.20613	0.00						

----- DNB HEAT FLUX CALCULATED BY LUND -----

---- TCLAD &lt; TEAT

HEAT FLUX OF CLAD --- Q'FR'EM'

( KARMAN - NIKURADSE EQUATION WAS USED FOR WALL LOSS CALCULATION )

-----PLOT INPUT DATA FROM FT05 -----4-----5-----6-----7--

WITHX	WITHY	TMIN	TMAX	PMIN	PMAX	NMIN	NMAX
140.000	200.000	0.000	800.000	0.000	4.000	0.000	200.000

NEWIN= NEW

TITLE= JNR-6 TRIGA-16 CORE CASE:GA BENCHMARK

```
IDPLOT= 1 IDPLOT= 2 IDPLOT= 3 IDPLOT= 4 IDPLOT= 5 IDPLOT= 6 IDPLOT= 7
   11      13      23      14      12      22      21
NMBNL = 1 NMBNL = 2 NMBNL = 3 NMBNL = 4 NMBNL = 5 NMBNL = 6 NMBNL = 7
   0       0       0       0       0       0       0
```

-----PLOT INPUT DATA FROM FT11 -----4-----5-----6-----7--

IA= 1 JA= 1 N= 1 NP= 1 K= 1

J	X(J)	TCOOLANT	TCLAD	TMEAT	TSAT	TOMB	PRESS	HEAT FLUX
1	0.000	37.00	86.20	272.02	118.16	126.49	1.97162	55.84
2	9.940	37.49	71.86	200.80	119.10	125.35	1.97270	56.84
3	19.880	38.35	90.15	260.26	119.21	126.75	1.97485	59.83
4	29.820	39.57	107.32	369.40	119.24	127.84	1.97661	78.75
5	39.760	41.07	118.74	428.06	119.07	128.37	1.98573	92.65
6	49.700	42.77	125.49	462.08	119.10	128.68	1.98762	101.14
7	59.640	44.57	127.33	474.20	119.13	128.79	1.98953	104.23
8	69.580	46.37	126.19	462.78	119.16	128.71	1.97144	101.14
9	79.520	48.07	121.84	430.17	119.19	128.46	1.97336	82.65
10	89.460	49.58	111.71	373.79	119.02	127.65	1.96367	78.75
11	97.980	50.79	97.33	205.43	119.05	126.61	1.96460	89.83
12	52.940	51.65	81.04	211.06	119.06	128.26	1.96681	38.84
13	55.880	52.14	98.40	281.22	119.10	128.43	1.96768	55.84

PLOT START, JMAX= 16

```
II = 1 PLOT END
II = 2 PLOT END
II = 3 PLOT END
II = 4 PLOT END
II = 5 PLOT END
II = 6 PLOT END
II = 7 PLOT END
```

NORMAL END

## Appendix B Sample JCL for COOLOD-N2

```
T(01) W(03) I(03) C(03) GRP
// EXEC      LMGOEX,LM='J3907.CLDN9305',Q='LOAD',
//          PRN=TEMPNAME,A='ERRCUT=0',GOSYSIN='DDNAME=SYSIN'
//FT05F001 DD DSN=J3907.JRR4ROD.DATA(GA002),DISP=SHR
// EXPAND DISK,DDN=FT11F001
// EXPAND GRNLPLIM,SYSTOUT=U,OTLIM=300000
//
```

The COOLOD-N2 code is also available for NEC-PC9801 series and IBM PC-AT compatibles (PC version, there are some restrictions).

# 国際単位系(SI)と換算表

表1 SI基本単位とSI補助単位

名 称	記号	単位
長 度	m	メートル
質 量	kg	キログラム
時 間	s	秒
電 流	A	アンペア
熱力学温度	K	ケルビン
物質の量	mol	モル
光 強 度	cd	カンデラ
立体角	sr	ストラーダ

表2 SIと併用される単位

名 称	記号
分、時、日	min, h, d
重さ、体積	g, L
力、トルク	N, N·m
電子エネルギー	eV
原子質量単位	u

TeV  $1.60218 \times 10^{-17}$  J

Tu  $1.66054 \times 10^{-17}$  kg

表3 国内で用いられるSI単位

量	名 称	記号	単位	他のSI単位に相当する量
周 波 数	周波数	Hz	s <sup>-1</sup>	
力	重力	N	kg m s <sup>-2</sup>	
压力、蒸気圧	気圧	Pa	N m <sup>-2</sup>	
電磁力、電場強度	電界強度	D	N m C <sup>-1</sup>	
電力、放熱率	出力	W	J s <sup>-1</sup>	
電気量、電荷量	電荷量	C	A s	
電位、電圧、起電力	電圧	V	W A	
静電容量	電容	F	C V	
電気抵抗	抵抗	Ω	V A	
電気エネルギー	電気エネルギー	Wh	V A s	
磁束密度	磁束密度	T	Wb m <sup>-2</sup>	
電気導通度	導通度	H	Wb A	
セル外気温	外気温	°C	kelvin	
光強度	光度	lm	cd sr	
放射能	放射能	Bq	s <sup>-1</sup>	
吸収線量	線量	Gy	J kg <sup>-1</sup>	
線量当量	当量線量	Sv	J kg <sup>-1</sup>	

表4 SIと併用判定的で  
確実に用いられる単位

名 称	記号
オームスロー	Ω
ヘルツ	Hz
バースペル	bar
ガル	Ga
カナル	Ca
ラード	rad
レーヴィ	rem

1 Å  $1.0 \text{ nm} \times 10^{-10}$  m

1 b  $100 \text{ fm}^2 \times 10^{-2}$  m<sup>2</sup>

1 bar  $0.1 \text{ MPa} \times 10^6$  Pa

1 Gal  $1 \text{ cm}^3 \times 10^{-6}$  m<sup>3</sup> s<sup>-1</sup>

1 Cd  $3.7 \times 10^{-3}$  Bq

1 R  $2.58 \times 10^{-4}$  C kg<sup>-1</sup>

1 rad  $1 \text{ eGy} \times 10^{-1}$  Gy

1 rem  $1 \text{ eSv} \times 10^{-1}$  Sv

表5 SI換算表

倍数	換算式	単位
$10^{-1}$	$10^{-1}$	E
$10^{-2}$	$10^{-2}$	P
$10^{-3}$	$10^{-3}$	T
$10^{-4}$	$10^{-4}$	G
$10^{-5}$	$10^{-5}$	M
$10^{-6}$	$10^{-6}$	k
$10^{-7}$	$10^{-7}$	h
$10^{-8}$	$10^{-8}$	da
$10^{-9}$	$10^{-9}$	d
$10^{-10}$	$10^{-10}$	c
$10^{-11}$	$10^{-11}$	m
$10^{-12}$	$10^{-12}$	n
$10^{-13}$	$10^{-13}$	p
$10^{-14}$	$10^{-14}$	f
$10^{-15}$	$10^{-15}$	a

注

1. 表1～5は国際単位系 第1版、国際単位系局 1985年1月1日より施行するまで、TeV より大きい値は CODATA の1986年標準値による。

2. 表2には海里、マイル、マーラー、マーラーが含まれているが日常の単位なのでこれらは省略した。

3. bar (1. JISでは液体の圧力を表す場合に限り表2のカッコ内に分類されている)

4. EC開発理事会指令で1 bar, barnおよび「血圧の単位」mmHg を表2のカッコ内に入れている。

## 換 算 表

力	No	$10^3$ dyn	kgf	lbf
	1	0.101972	0.224809	
	9.80665	1	2.20462	
	4.44822	0.453592	1	

粘度  $1 \text{ Pa} \cdot \text{s} = 10^3 \text{ St} = 1 \text{ cSt} = (\text{g}/\text{cm} \cdot \text{s})$

動粘度  $1 \text{ m}^2/\text{s} = 10^6 \text{ St} = 1 \text{ cSt} = (\text{cm}^2/\text{s})$

力	MPa (= 10 bar)	kgf/cm <sup>2</sup>	atm	mmHg (Torr)	lbf/in <sup>2</sup> (psi)
	1	101972	9.86923	750062 × 10 <sup>-3</sup>	145.038
	0.0980665	1	0.967841	735.559	14.2233
	0.101325	1.03323	1	760	14.6959
	$1.33322 \times 10^{-4}$	$1.35951 \times 10^{-3}$	$1.31579 \times 10^{-3}$	$1 \times 10^{-3}$	$1.93368 \times 10^{-3}$
	$6.89476 \times 10^{-5}$	$7.03070 \times 10^{-5}$	$6.80460 \times 10^{-5}$	$51.7149$	1

放 射	Bq	Ci	吸 収	Gy	rad	輻 射	C/kg	R	線量 当 量	Sv	rem
	1	$2.70270 \times 10^{-1}$		1	100		1	3876		1	100
	$3.7 \times 10^{16}$	1		0.01	1		$2.58 \times 10^{-4}$	1		0.01	1
	$1.60218 \times 10^{-1}$	$1.63577 \times 10^{-1}$	$4.45050 \times 10^{-1}$	$3.82743 \times 10^{-1}$	$1.51857 \times 10^{-1}$	$1.18171 \times 10^{-1}$					