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## RAPHAEL IP PROJECT HIGH AND VERY HIGH TEMPERATURE COMPONENT DEVELOPMENTS

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### ABSTRACT

Arising from the EU 5<sup>th</sup> and 6<sup>th</sup> Framework Programs (FP's), the purpose of this paper is to present the achievements gained in the area of HTR & VHTR component development within the 5<sup>th</sup> FP HTR-E project and the future work activities to be realized in the frame of the new RAPHAEL project (6<sup>th</sup> FP).

The HTR-E R&D project started on 1<sup>st</sup> January, 2002 with 14 partners, from industry and research centres involved in HTR development: Framatome ANP, CEA, Zittau university, NRG, FZ Juelich, Empresarios Agrupados, NNC, Jeumont, S2M, Ansaldo, von Karman institute, Heatric, EV Oberhausen, Aubert et Duval. The work programme concerned the technical developments of innovative components of a modern HTR with a direct cycle, with references to industrial projects existing at the time (GT-MHR, PBMR) for direct cycle HTR. The main tasks performed within the HTR-E Work Packages were as follows:

- The helium turbine (WP1), the recuperator heat exchanger (WP2), the electro-magnetic and catcher bearings of the turbo-machine (WP3) and the helium rotating seal: dry gas system, fluid film barrier, canned magnetic bearings (WP4). Based on past experiences and specific calculations, design recommendations of such components were proposed. Experimental tests were also performed to validate the recommended concepts for electro-magnetic bearings and recuperator heat exchanger.
- The tribology (WP5). Sliding innovative components in helium environment were particularly concerned (stator seals, control rod mechanisms...). The experience

feedback was analysed and complementary tests have been carried out by CEA and Framatome ANP.

- The helium purification system (WP6). This work package provided recommendations on impurity content in the helium atmosphere for a modern HTR in accordance with the materials proposed for the innovative components.

In April 2005, as part of its 6<sup>th</sup> Framework Programmes, the European Union started a new 4-year Integrated Project on Very High Temperature Reactors (RAPHAEL: **Re**Actor for **Pro**cess heat, **H**ydrogen And **E**lectricity generation).

The European Commission together with more than 30 participating companies, R&D organizations and universities from different European countries fund the project together. Such a reactor was found to have a large potential in terms of safety (inherent safety features), environmental impact (robust fuel with no significant radioactive release), sustainability (high efficiency, potential suitability for various fuel cycles), and economics (simplifications arising from safety features).

After the successful performance of the 5<sup>th</sup> FP HTR-E project, RAPHAEL IP focuses now on the key technology needs for an industrial VHTR deployment, both specific to very high temperature (900-1000°C) and indirect cycle.

The objectives of the RAPHAEL IP sub-project (SP-CT) concerned with component technology includes the development and validation of the following components with the associated key issues:

- Intermediate Heat eXchanger (IHx): the compactness for both tube and/or plate concepts and their integrity under the load. This is a real challenge and beyond the existing industrial experience,

- Hot helium valves: the role is to isolate the reactor in case of failure of the IHX. Design study based on the past experience in Germany and from AGR (UK) and existing technologies will be performed,

- Gas circulator: the design will be focused on large power for a 600 MWth reactor with low power consumption. Magnetic bearings will also be developed and tested in an IPM Zittau facility.

In parallel with these technological developments, tests using two specific tribometers operating up to 1000 °C will be carried out in order to determine the behaviour of sliding components (hot gas duct seal, control rods...) under VHTR operating conditions. The tests will include the effects of corrosion from Helium and impurities and the investigation of new graphite grades and ceramic materials.

## 1 INTRODUCTION

In a first part, the present paper aims to summarize the achievements gained in the area of HTR component development within the fifth Framework Program HTR-E project, see references 1 to 4.

In a second part, the future work activities to be realized in the frame of the new RAPHAEL project (6<sup>th</sup> FP) are also presented.

## 2 HTR-E PROJÉT

The main objective was to develop innovative technologies which are needed for the components and systems of a modern HTR with a direct cycle and then contribute to the competitiveness with a net plant efficiency of around 50 %, safety and acceptability of such reactors. The critical and innovative components and systems were selected by reference to existing projects with a direct cycle concept (GT-MHR, PBMR).

The key components and systems selected for HTR-E were:

### - The helium turbine, WP1.

The feasibility of a high efficiency helium turbine, inserted in the primary cycle, is the most important technological key issue for the components of a modern HTR.

### - The helium/helium heat exchanger called recuperator, WP2.

The recuperator is a helium/helium heat exchanger inserted in the primary circuit to recuperate a part of the remaining energy at the turbine outlet to preheat the helium at the core inlet.

### - The large capacity magnetic bearings, WP3.

These components are used to support the rotating shafts of turbo-compressors or turbo-machine inserted in the direct cycle of a modern HTR.

### - The helium leak-tightness rotating seal, WP4.

The helium rotating seal is associated with a rotor of the turbo-machine penetrating through the primary vessel. In this case the generator is located outside the primary circuit that simplifies the maintenance operations and that allows to use mechanical bearings for the support.

### - The tribology, WP5.

Sliding and new components (stator seals, hot gas duct seal...) of a modern HTR with direct cycle are particularly

concerned. It must be noted that a bad operating of these sliding components will dramatically impact the safety and availability of the reactor.

### - The helium purification system, WP6.

The performances of this system impact directly the safety of a modern HTR as it controls the level of impurities in the primary helium.

## 2.1 WP1 Helium turbine

The partners involved in this WP1 were: Framatome ANP, CEA, FZJ, EA, VKI with two subcontractors: EVO for experience feedback and A&D for materials.

The helium turbine is not a safety classified component but to confine material fragments within the pressure vessel in case of deblading a safety related function for the stator is recognised. Indeed, based on HTR-L project comments, the deblading accident and the risk of disk rupture, a horizontal shaft was recommended.

The nominal operating conditions consider 850 °C at the inlet and an operating period of 60 000 hours without any intervention on this component, inducing a spare part unit for maintenance and repair. The nominal rotation speed is 3000 rpm that leads to important stresses in the disc (diameter = 1400 mm) and in the blades. Then, the creep damage after 60 000 h for the first stages of the turbine was analysed carefully.

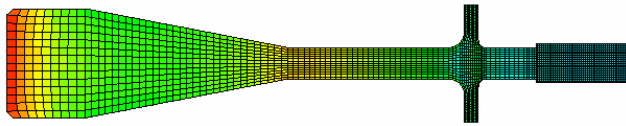
Taking into account the proposed materials within the HTR-M/M1 project, an acceptable damage can not be achieved without a cooling of the discs of the first stages by helium derived from the compressor outlet. A&D proposed the Udimet 720 grade for the disk, without any limit related to the cobalt content. At the opposite, OKBM in ISTC 1313 project limited this cobalt content. The disk material is out of the present stage of the art but seems achievable by conventional cast and wrought or by prealloyed powder metallurgy process. For the blades, materials (Ni-based super alloy) were also proposed within the HTR-M/M1 project. It appears not necessary to cool the blades but a protective coating against corrosion may be needed.

After a preliminary design phase (see reference 1), the stress field was calculated thanks to CAST3M and ANSYS codes, for the discs and the blades to assess the thermo-mechanical damages on these elements (creep, fatigue,...).

In a first step, a simplified axisymmetrical model using linear mechanical behaviours was simulated in order to have an idea of the temperatures and stresses fields likely to occur within the turbine disks (see next figure).

Although the stress concentrations due to the 3D geometry, near the blade foot for instance, were not considered, this simulation shows that with the foreseen materials a cooling of the first stages is necessary. Thus, an estimation of the necessary derivated helium flow was performed and it is shown that such a cooling should not lead to a dramatic decrease of efficiency.

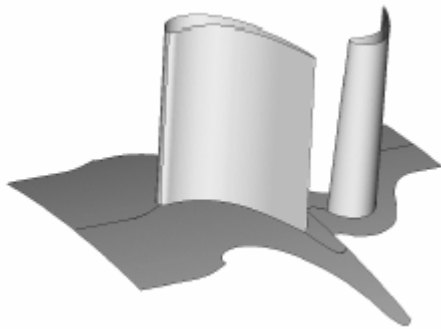
In a second step, the life time of a turbine disk at several temperatures was estimated by Finite Element Method (CAST3M) with the following mesh:



The detailed life time was determined using a non linear mechanical behaviour (viscoplastic Chaboche law) based on the experimental data (tensile, low cycle fatigue and creep tests) of U720 Coarse Grain material obtained within the HTR-M program. It is shown that cooling the disk down to 750°C should be sufficient to reach a lifetime of 60000 hours. However, these simulations raised the lack of experimental data for such long durations. Thus, the proposed results were based on the extrapolations of the mechanical behaviour to such long creep steps.

Aerodynamic design and CFD calculations (DACAT and BLADES codes, inverse method) were performed by VKI for typical operating conditions to optimise the shape of the blades. The main conclusions obtained are the following ones:

The compressor performance exceeds the requirements and initial predictions. The main reason is the careful blade design with rather low loading. This low loading results in a larger number of stages (21 instead of 18 for the HP compressor). This increases the shaft length and could eventually lead to rotor dynamic problems. The turbine efficiency is lower than the initial target value. This is a direct consequence of the small number of stages at 3000 RPM. Increasing the number of stages or increasing the rotational speed to 3600 RPM would allow lower loading per stage, hence higher efficiency as clearly shown by the Smith correlation. Here again this would increase the shaft length and could eventually lead to rotor dynamic problems.



**3D view on optimized turbine stator and rotor**

The amount of cooling flow needed to keep the first turbine rotor root below a given maximum temperature is very small and without measurable impact on the gas turbine efficiency. One single hole in the blade root of 5 to 10 mm diameter is sufficient.

Related to the aerodynamical tests definition, a model is proposed to define the conditions for a correct experimental simulation of the steady performance of each component as well as the unsteady interaction between each component of the circuit. In particular, one wants to avoid surge and stall,

excessive overspeed of the rotating components, overheating or thermal shocks. It is shown that the main requirements are:

- a model compressor/turbine performance map that is similar to the one of the real compressor/turbine
- conservation of the ratio between the different volume of reactor, heat exchangers and coolers
- conservation of the length over area ratio of the ducts.

As an example, two ways of controls have been analysed: inventory control or by-pass. The first one is found more efficient while the second one may induce strong thermal gradients although it looks more reactive. The proposed approach will be very useful to design an appropriate test facility with smaller size, air instead of helium.

Regarding, the mechanical tests necessary to fully investigate the mechanical behavior of the foreseen turbine disks and blades materials are addressed. It was concluded that the mechanical specimens will have to be machined from an ingot not far from the full scale. This is the only way to be representative of the micro structural heterogeneities as well as chemical segregations. Concerning the disks, a test matrix was proposed including the following tests: tensile, creep, Low Cycle Fatigue, fatigue crack growth. The effects of ageing on tensile and LCF and the effect of carburization were considered. An analysis of the anisotropy and discrepancy of properties within the disk is proposed. Concerning the blades materials, the same types of tests were proposed. Moreover, high cycle fatigue tests were added. Higher temperatures and corrosion effects were considered. After the completion of this program, it is possible to launch a detailed design of the turbine, the manufacture of a mock-up and the tests of this mock-up on a large scale facility or in international collaboration (e.g. test loop at HTTR/HTR10 or others). However, the final design of the turbine components (especially the turbine disks as well as blade foot cooling system) will have to be optimized by a turbine manufacturer.

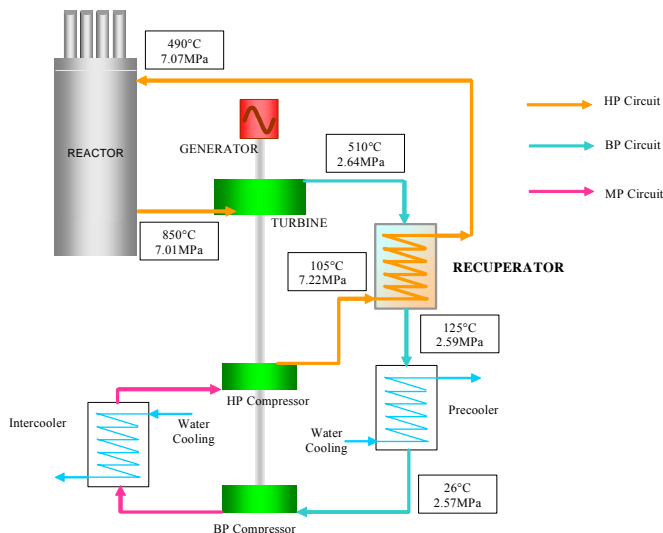
**The main overall conclusions drawn from this WP1 are as follows:**

- A horizontal shaft is preferable than a vertical one, for safety reasons (deblading accidents) and due to catcher bearings limitations,
- 3D design of the blades is feasible, a coating is necessary against the corrosion effect. The effectiveness and the number of stages are as follows:
  - \* Turbine with 12 stages and  $\eta = .90$ ,
  - \* LP compressor with 15 stages and  $\eta = .940$ ,
  - \* HP compressor with 21 stages and  $\eta = .938$ .
- Cooling is necessary at the blade foot and also for the disk, but a small flow rate is sufficient,
- Disk made of U720 is achievable with a diameter of 1,5 m.,
- Further developments need a manufacturer (cooling design, coatings,..),
- An important R&D effort has to be done in order to determine the thermal and mechanical properties of the U720.

## 2.2 WP2 Recuperator

The partners involved in this WP2 were: Framatome (France), CEA, FZJ, NRG, ANSALDO and HEATRIC. The requirements of the GT-MHR and PBMR recuperator heat exchanger were specified by FANP (using available public information for PBMR). This component is used to 'recuperate' a part of the remaining energy in the LP gas (2.63 MPa) at the turbine outlet (510 °C) in order to heat the HP gas (7.07 MPa) at the HP compressor outlet (105 °C) before the core inlet (490 °C). This component is not located on the primary barrier but it is fully immersed inside the pressure vessel. Consequently, it is not safety classified. Note that there is a strong incentive to consider a compact design for this component in order to limit the impact on the size of the pressure vessel.

The preliminary GT-MHR analysis of normal and design transients based on the magnitude of temperature and pressure by comparison with the normal operation shows that the most severe transients is the loss of offsite power with turbo-machine trip (cold shocks):



- $\Delta T = - 335 \text{ }^\circ\text{C}$  for the LP recuperator inlet, in 5 seconds, from initial values : 508 °C,
- (severe slope  $\approx 70 \text{ }^\circ\text{C/s}$ ), then, the temperature rises again in 508 °C in 120 seconds after initiation,
- temperature maximum = 650 °C at the end of the turbo-machine run-out,
- number of scram occurrences: 300.

In terms of in service inspection and repair requirements, note that a modular concept was specified in order to be able to locate and repair the leaky module by plugging. The main concepts suitable for the recuperator application are mainly (see reference 1):

- Tubular concept based on past HTR experience on steam generators and heat exchangers,
- Printed circuit concept (Heatric company),
- Plate-fin concept (Nordon, MHI, Ingersoll Rand...).

The engineering studies (NRG) were performed on two basic concepts (Heatric and Nordon) including preliminary design, CFD analysis of flow distribution, heat exchange and pressure losses performances and identification of potential

high temperature region, thermo-mechanical resistance analyses (ANSYS code). The main results obtained on the Heatric mock up from a 3D analysis were as follows:

- Hot side of IHX mock up is subject to the most severe loads,
- Steady state after cool down is reached after 1-2 hours. However, most of the temperature evolutions are within the first 20 minutes.
- Life prediction is difficult (very sensitive to the code used). Depending on the code (ASME or RCC-MR), the minimal fatigue life changes drastically (safety factor over than 20). So no reliable fatigue life prediction is possible for the mock up with the available fatigue data.

The same type of analysis was performed by NRG on the NORDON concept. The results show an increase of the thermo mechanical stresses in comparison with the Heatric concept inducing a reduction of the corresponding life time duration.

The Heatric mock up was equipped with a series of gages and thermocouples as showed on the following picture and then tested on the Claire test loop of the Esther platform:



The main conclusions on HEATRIC mock up tests are as follows

### - In steady state :

The hydraulic performances are in good agreement with Heatric predictions. But related to the thermal performances, CEA observed important discrepancy between Heatric predictions and experimental results due to fluid properties, longitudinal conduction and a flow mal distribution between channels linked to the design of the headers. Nevertheless 95% efficiency was reached.

### - In transients:

Submitted to temperature transients, HEATRIC mock up withstands 100 cycles without any leak (Helium tests were performed every week)

Experimental stresses are higher than those calculated.

The Nordon mock up was manufactured with delays and then not tested due to the lack of time.

### The main overall conclusions drawn from this WP2 are as follows:

- Heatric mock up is able to fit with the steady state requirement but caution should be taken for flow distribution design,
- The Heatric heat exchanger resists after 100 cold/hot shocks, but plastic deformation occur inside,

- For final statements, need of tests under real gas environment,
- The PFHE remains a potential candidate for Recuperator technology but caution have to be taken on the design of small height fins and lateral bars
- The FEM analysis seems to predict more mechanical problems than PCHE.

**2.3 WP3 The large capacity magnetic bearings**

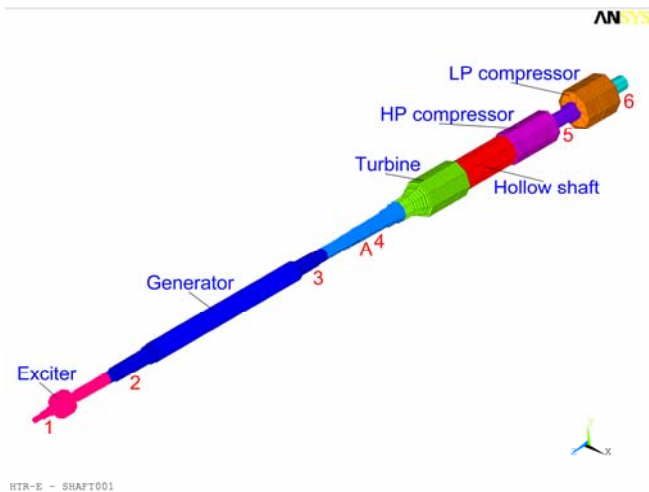
The partners involved in this WP3 were: Framatome ANP, S2M, NRG, FZJ and Zittau University with EDF as observer.

The functional requirements (see reference 1) related to the axial and radial active magnetic bearings and the axial and radial catcher bearings are based on the two following machines:

- the power turbine of a GT-MHR type reactor,
- the high pressure turbo compressor of a PBMR type reactor.

Note that the AMBs by themselves are not sized to withstand the seismic loads and that; in this case, the catcher bearings will react to support a part of these loads.

Taking into account these required load capacities, geometrical constraints and other requirements detailed, a conceptual design has been elaborated for the AMBs support of such a machine, see the figure hereafter. This conceptual design is suitable for a maximum temperature < 150 °C. Note that the axial magnetic bearing is not sized for 105 tons and that one solution to support the GT-MHR rotor would be to consider two discs with two axial magnetic bearings.



The load analysis was performed with dynamic simulation. The simulation tool MLDyn was parameterised with the values given from the requirements and the design study. With help of the simulation tool, all operating modes (start up, unbalances, speed levels, static and dynamic loads) were investigated for stiff rotors.

The required modal analyses, harmonic response analyses as well as the frequency response analyses were performed with Finite Element analyses (ANSYS code). The

numerical results provided the boundary and loading conditions for the design and the simulation tool (MLDyn).

An alternative of Active Magnetic Bearings for the rotating components support considering Permanent (Passive) Magnetic Bearings was investigated by a feasibility study. The fail safe characteristics of this system are particularly interesting to reduce the loads to the catcher bearings in accidental conditions (loss of power).

A specific feasibility study on mechanical catcher bearings was performed. As part of catcher bearing tests, the study covers the past experience from the German HTR R&D Programme. In this study, it is pointed out one important item which is the retainer bearing in the case of AMB technology. The role of this retainer or catcher bearing is to prevent contact between rotor and static circulator parts upon unintended de-energisation of the magnets or malfunction of the electronics. The reference design for the test of the catcher bearings was the HTR-500 auxiliary blower for decay heat removal having an impeller diameter of 1.25 m. A specific test facility was built (FLP500 which was transferred to Zittau University, see picture hereafter) to simulate the conditions during coasting of the blower from 6000 rpm.



**FLP 500 Test Facility**

The magnetic, thermal and structural behaviour of the magnet supported system for the various load cases were determined by means of transient Finite Element analyses (MSC.Marc code).

**The main overall conclusions drawn from this WP3 are as follows:**

Permanent Magnetic Bearings

- PMBs could be used in addition to the investigated AMBs
- The main features of PMBs are:
  - \* gained permanent radial stiffness
  - \* guidance of the shaft
  - \* additional load capacity
- that promises:
  - \* reduced loading on the CBs,
  - \* thereof the expected lifetime of the CBs may increase,
  - \* otherwise the manufacturing problem of the proposed CBs (size of ceramic balls) may be solved.

*It has to be noticed that PMB concept will be investigated in frame of RAPHAEL SP-CT WP2.*

#### Catcher Bearings

- required load capacity of radial CBs is determined by seismic loadings,
- further investigations and experiments are necessary,
- limitation of the axial CB capacity is a feasibility issue for heavy rotor.

#### Rotor arrangement of the turbo machine

- a horizontal shaft or an arrangement of two vertical shafts with a flexible coupling was recommended to be developed.

#### Tools

- the use of MLDyn for dynamic simulation, MLDia for Diagnostics and ANSYS for FEM-calculations was proofed.

### **2.4 WP4 Helium leak tightness rotating seal**

The partners involved in this WP4 were: Framatome ANP, S2M, NRG, FZJ and Zittau University with EDF as observer.

For a direct cycle HTR (GT-MHR type reactor) considering a single shaft for all rotating components (turbo-compressor and generator), it is interesting to locate the generator outside the pressure vessel. Then, the support design of the shaft can be different and simpler (located outside the pressure vessel, using a mechanical bearing, generator and turbo-compressor shaft separately supported). In addition, as the accessibility to the generator becomes possible without opening the pressure vessel, the maintenance operations on this component are easier. But this arrangement (outside generator and fully immersed turbo-compressor) leads to insert a helium leak-tightness rotating seal on the primary helium containment barrier to allow a passage for the rotating shaft.

It is first of all necessary to evaluate the maximum leakage objectives, taking into account the potential gas activity, the radio protection and the safety objectives.

In a second step the different leakage points on a modern HTR design were listed and classified (static joints, welds, gliding joints, rotating joints...).

State of the art is established for each joint class, in order to check that there is an industrial solution, adapted to the modern safety and radio protection requirements.

The main requirements were extracted from a GT-MHR type turbo-machine shaft and PBMR type power turbine shaft. The experience feedback on oil seals used is not satisfactory and a dry seal was recommended.

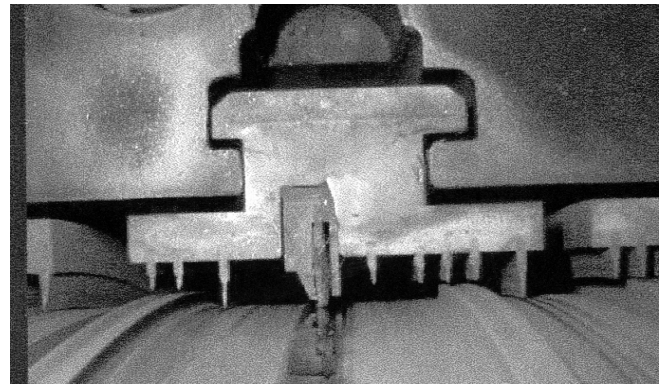
Different available concepts are already tentatively identified and analysed taking into account the state-of-the-art and feedback experience on similar systems used in industry:

#### **CMBs**

With help of a single effect analysis on basis of theoretical investigations for Canned Magnetic Bearings it is planned to find an innovative technical solution for helium leak-tightness rotating systems. A non magnetic can is recommended and depicted the influence of the can (thickness 0.2 to 0.5 mm) on the bearing capacity/size and

also on the sensors sensitivity. This influence will be much more significant for small machines. The experimental tests to be performed in order to analyse such conceptual designs are also specified. In connection with the analysis of the existing devices, a suitable concept of dry system as regards the HTR conditions is defined, and justified by theoretical analysis. The state of the art shows that the present industrial systems can not meet the objective of admissible leakage for a modern HTR (10 to 30 % of total primary helium mass per year). That's why the development of an innovative buffered dry gas face seal is recommended.

Dry gas seal was studied. The proposed solution is composed of a tandem of two hydrodynamic seals with Raleigh pads, a labyrinth between the vessel and the first seal, and an injection of clean helium between the labyrinth and the first seal. The large shaft size is the challenge of this work package due to the lack of experience. Hydrostatic seal concept with an orifice controlling element is another alternative solution if necessary. The potentiality of brush seal is underlined (see here after)



The experimental tests program required for the best promising concept were defined. It is also analysed the capability to carried out these tests on HTTR and HTR 10 test loops. At the end of this feasibility study, the best promising concept was identified. The manufacturing of a mock-up and the experimental tests campaign is possible, after the 5<sup>th</sup> FWP, on a large scale facility or in international collaboration (e.g. test loop at HTTR/HTR10). Based on the study results performing during the first two years of the contract (2002-2003), the following conclusions can be elaborated:

- The admissible leakages were defined by the partners,
- A design based on a buffered dry gas face seal was proposed by Jeumont,
- The rotating seal with an initial gap meets the requirements: the seal face deformation is less than one  $\mu\text{m}$  and the stress level in the seal ring is smaller than 70 Mpa,
- Dry rotating seal (hydro-dynamic seal with injection of clean helium under pressure) is feasible and meets the leakages limits even with a higher Delta P induced by pressure fluctuations in the circuit, coming from HTR-L design recommendation.

## 2.5 WP5 Tribology in helium conditions

The partners involved in this WP5 were: Framatome ANP, NNC and FZJ

Due to the lack of oxygen in the Helium circuit, and to the elevated operating temperatures, gliding or rotating surfaces are not covered by the usual protective and lubricating oxide coating. Nevertheless, it is known from 1975's years from work on THTR reactor that the HTR Helium used is not completely an inert gas since it contains impurities such as  $\text{CH}_4$ ,  $\text{H}_2\text{O}$  (steam), which are able to modify the material surfaces. This environment could result in enhanced jamming risks for mobile parts. As components such as control rods, fuel handling machine, turbine... could be affected, the safety and plant availability are strongly concerned by a bad operating of these gliding components.

It is therefore necessary to collect the available information and lessons from previous experiences, to evaluate the risks (safety, investment, availability) and to define the further developments that could appear useful to reduce these risks at the appropriate level. Jamming or seizing up of mobile parts (gliding seals, rotating parts, gliding parts) is enhanced in non-oxidising and/or particular environment conditions. In some cases, sticking can even be observed. It appears that the work in this field could take several complementary ways to be defined prior to any development. It would take into account the following items: materials selection, surface treatment or coating, usual operating conditions (temperature, temperature or pressure variations...), tribological operating conditions of the component such as contact load or torque, contact velocities, kind of contact, transient or stationary operating conditions...

A review of the knowledge concerning tribology in non-oxidising and/or particular conditions such as Helium +  $\text{CH}_4$ ,  $\text{H}_2\text{O}$  (as steam), was first established. This is performed by using surviving operating and design experience from HTR such as Dragon project (UK), or others such as THTR 300 (D), or Fort Saint Vrain (USA)...when available, and FBR reactors, experience from other industries if possible and literature.

Both tribometers were used:

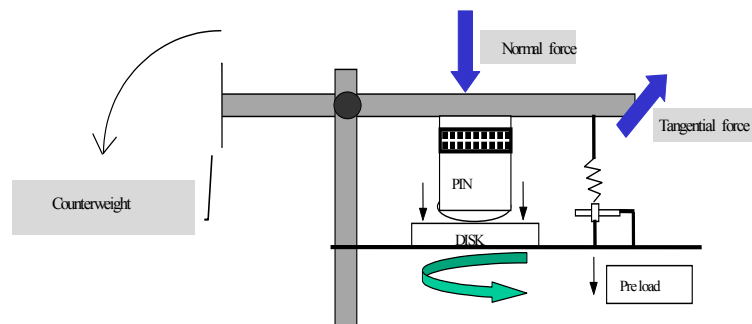
- the new tribological equipment of the CEA, based on a former GA design and presented hereafter:



Figure 1a: View of CEA He tribometer

This tribometer used a plane-plane contact with an alternative motion of 6 Hz and can applied load up to 20 MPa. The environment can be a temperature level up to 1000 °C and He with impurities under a pressure level of 5 bar. The gas sampling shall be made through the furnace, near the samples to be tested, in the hot part of the helium volume.

- The pin on disk tribometer bench from FANP Technical Centre can reach at a temperature level of 600° C with a load capacity up to 500 MPa (representative of real loading conditions). The maximal pressure of helium with or without impurities is 1 bar.



In parallel, a review and a classification of the tribological problems on a HTR design was established (on the basis of GTMHR or PBMR design) : this review particularly pointed out for the different involved components, the potential materials, the tribology type (gliding, rotating, continuous or not), the environment conditions (temperature, gas, pressure variations,...), the typical contact conditions (contact pressure, shift, speed, acceleration, frequency, ...) and will specify if it occurs during transient (start or outage periods, control rod drop) or operating conditions.

From the bibliography, it appears particularly that contacts such as metal-metal contact would not be allowed, but compromise with contact metal-carbide could be studied for different carbide content / metallic matrix ratio. Other materials could present a good tribological behaviour in such HTR Helium atmosphere (Molybdenum based coating for rod mechanism, new composite materials for heat insulator,..). A review of the existing technologies (coating, surface treatment, composite materials) on the basis of an analysis of available data base, and of their performances

with respect to the requirements and to the operating conditions is reported.

A selection of materials and antagonistic couples were made for two principal test campaigns. It could represent a first step before a bigger experimental approach of the tribological problems understanding in HTR.

Tests of these materials were performed on samples in representative configurations (sliding + fretting, sliding + bearing (to be chosen from the result of the State-of-the-Art among gears and pinions / racks of control rod drive mechanisms, hot gas duct and insulator) and in as representative as possible conditions: approached chemical environment (Helium with oxidising agents or carbide containing Helium, from possibly applicable WP6-data), temperature 500-750°C according to the components, without pressure (1 to 5 bar) but with concentrations representative to the partial pressure of additive elements at the working temperature.

Tests were performed with a common methodology, common material procurement, machining and surface preparation of samples to allow a better comparison of the results obtained on the materials or coatings, and antagonistic couples proposed for components. The selected base materials are either HR230 or 9%Cr with different type of coatings such as ZrO<sub>2</sub>/ Y<sub>2</sub>O<sub>3</sub>; ZrO<sub>2</sub>/CaO; SiC; CR<sub>3</sub>C<sub>2</sub>. Tests are realized without impurities.

**The main overall conclusions drawn from this WP5 are as follows:**

- The best results have been obtained at 700°C on CERMET coating (Cr3C2-NiCr) deposited by HVOF,
- The Coating Zirconia (ZrO2/Y2O3) + solid lubricant (CaF2) has to be improved (adhesion) for a resistance at 900°C,
- SDG2005 (WC, WC3 / Cr2C3, Ni) deposited by Super Detonation Gun has a good behaviour at 250°C,
- Necessity to elaborate a specification for coating realization,
- Need of regulation about tribological tests,
- Tests have been realized with a sliding movement, and in the future, need to test other types of movement (sliding, dwell motion ...).

**2.6 WP6 Helium purification system**

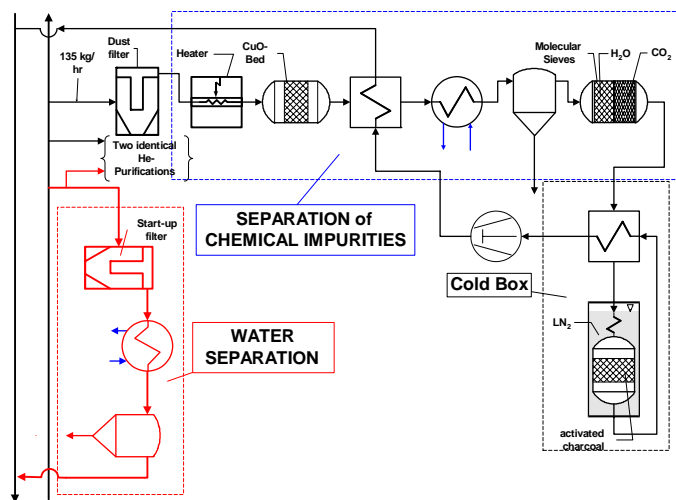
Reliable Helium Supply and Purification Systems (HPS) are essential for the safe operation of a modern HTR. They partially also serve for additional functions with respect to operation or control of accident sequences to keep radiological releases as low as reasonably possible. The HPS is of utmost importance with regard to the material behaviour under high temperatures (corrosion, erosion) also under normal conditions because impurities and humidity is inserted via fuel loading, coolant replenishing and any opening of the circuit for maintenance and repair. This point is particularly important for the turbine materials.

Specific experience is available from former and present HTRs (DRAGON, Peach Bottom, AVR, THTR, FSV, HTR, HTR-10) and from the operation of large test loops like HHV, KVK and EVO. But these rather complex and expensive

system are mainly based on the state of the art in the 1960<sup>th</sup> - 1970<sup>th</sup>. Meanwhile more advanced processes are available and have to be evaluated for actual direct cycle plant concepts with TRISO particle fuel.

Available literature and data from former HTR projects were collected and additional relevant knowledge from other sources (HTR, HTR-10, Magnox, AGR, helium test loops) were included in a state-of-the-art information on the helium purification systems of different HTRs (Dragon, AVR, THTR-300, HTR, HTR-10).

Hereafter is presented the helium purification circuit:



It should be noticed, the importance of the cold trap in liquid nitrogen for the noble gases.

In conclusion, in term of impurity level (or contaminants) the proposed values are as follows:

- Normal operation :

Contaminants	Values (vpm)
H2O	< 0.1
CO	< 2.0
N2	< 1.0
H2	< 5.0
CH4	< 1.0

- Purified helium returned to the primary circuit: all contaminants < 0,1 vpm expected for each impurity.

**3 RAPHAEL IP Project**

Initiated in April 2005, the main objective of the Sub-Project on Component Development (SP-CT) of RAPHAEL IP is to develop innovative technologies needed for the main components and systems of a VHTR, taking into account strong requirements for the competitiveness, safety and the acceptability for Competitiveness is based on the use of these technologies in high performance components and systems, to allow electricity generation with net plant efficiency of about 50% or for high efficiency heat applications at very high temperature for industrial processes (e.g. hydrogen production). Similarly, high performance components and systems with respect to thermo-mechanical resistance,



lifetime, maintenance, behaviour in case of failure, etc. are required for the safety demonstration and for contributing to the acceptability of such. The FP5 addressed the development of the components of direct cycle. As already mentioned in the introduction of the description of the activities of this Project (background), an indirect combined cycle is more adapted, at least as a first step, for the VHTR with application to hydrogen production. The objective of the Sub-Project is therefore to develop conceptual designs for key components of the indirect cycle and to address their viability. The effort will be focused on the most challenging of these components:

- The heat exchangers,
- The gas circulators

Moreover tribology and corrosion phenomena are key issues in the selection of materials and the design, operation and lifetime of components and systems in helium atmosphere. Therefore a programme is proposed to study these phenomena for the different situations to be considered in the system. It will be focused on VHTR conditions, which were not covered by FP5 programme, in particular concerning the graphite/metal, graphite/graphite and ceramic contacts that will be investigated and tested in closer representative conditions.

This sub project is divided in three main work packages, as follows:

- **WP-CT1 Heat exchangers, valves and vessel :**

This Work Package contains the following items:

**1) The intermediate heat exchanger and isolation valves**

The IHX will have to be studied in relation with VHTR requirements and different options (heat application, combined cycle...) will have to be considered. The key issue for this component is its relative compactness and its integrity under the applied loads (temperature and pressure difference). The development of such a heat exchanger able to meet the anticipated requirements in term of heat exchange performance, pressure drops, mechanical resistance, etc. is a real challenge and beyond the present industrial experience. A feasibility study is proposed for two kinds of concepts: i.e. plate concept and the tube concept. For the tube concept the past experience in Germany will be recovered and elementary tests (forming, welding) will be performed for the selected high temperature materials. Thermo-fluid dynamic analysis and thermo-mechanical analyses will be carried out to assess the performances of the two concepts for steady states and during transients. High performances hot helium valves could be also necessary for the secondary pipes in order to isolate the reactor in case of failure of the IHX. A design study is proposed based on past experience in Germany and also on a technical survey of present technologies (e.g. AGR).

**2) The helium/water cooler**

The helium/water heat exchangers used as coolers are to be studied for VHTR. The key issues are relative to safety and compactness. A design study is proposed for two kinds of concepts: i.e. plate and tube concepts.. Qualification tests in air of representative mock-ups are proposed using the CEA air test loop.

**3) PCIV design study:**

Based on Prestressed Cast Iron Vessel (PCIV) a design study will be performed on main vessel in order to determine the geometrical characteristics for a power of 600 MWth. The heat transfer from vessel to coolers will also be assessed.

**4) Helium test loops:**

The testing of mock-ups of the key components of the primary circuit in helium atmosphere is an important element for the validation of the design. The possibility of making complementary tests at the Brasimone facility of ENEA, will be considered.

- **WP-CT2 Gas circulator**

The first task is the recovering of past experience from former HTRs and AGRs and projects and from other industries. A feasibility study is proposed for this component for a large power VHTR (600 MWth) for which the blower should have a power of 10 to 15 MW, which is much higher than in former HTRs (maximum 4 MW). A design study is proposed including the thermo-fluid dynamic design of the impeller, the electric motor and the bearings. It will also include a dynamic simulation in order to analyse the blower behaviour in case of external loads due to the process transients as well as to prove the reliability of the bearings (electromagnetic bearings + catcher bearings and/or permanent magnetic bearings). Experimental validation of rotor dynamics control in case of thermal and mechanical loads is foreseen on FLP 500 test facility in IPM Zittau.

- **WP-CT3 Tribology and corrosion**

Sliding components (hot gas duct seal, control rods...) of a modern HTR with indirect cycle are particularly affected by tribological issues. It must be noted that poor operation of these sliding components can dramatically impact the safety and availability of the reactor. These issues have been fully mastered in the reactors built in the past, but with lower temperatures, different materials and different configurations. This legacy cannot therefore be simply extrapolated to present possible designs and materials and least of all to fulfilling the requirements of VHTR operating conditions.

Therefore qualification tests will be carried out in this WP, with representative conditions, considering VHTR conditions (Temperature, Pressure, Velocity, etc.), and with a test facility allowing a large range of tribological contacts, long-term tests and coupling of mechanical, tribological and corrosion tests. Some tests relative to the new graphite grades will be also carried out. For possible extrapolations for even higher temperatures (> 1000°C), ceramic materials will be also investigated.

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## ACRONYMS

HTR	High Temperature Reactor
GT-MHR	Gas Turbine Modular Helium Reactor
IHX	Intermediate Heat eXchanger
EVO	Energieversorgung Oberhausen

EA	Empressarios Agrupados
S2M	Société Mécanique Magnétique
HP/LP	High Pressure/Low Pressure
AMB	Active Magnetic Bearing
AGR	Advanced Gas cooled Reactor
PX/SPX	Phenix/Superphenix
FZJ	Forschungszentrum Jülich
A&D	Aubert and Duval
WP	Work Package
PBMR	Pebble Bed Modular Reactor
ISTC	International Science and Technology Center
HHV	High Temperature Helium Test Plant
VKI	Von Karman Institute
NRG	Nuclear Research and consultancy Group
MHI	Mitsubishi Heavy Industries
CB	Catcher Bearing
FBR	Fast Breeder Reactor
FEM	Finite Element Method
FP	Framework Program